



HCCI and Stratified-Charge CI Engine Combustion Research

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Annual Merit Review and Peer Evaluation



Program Manager: Gurpreet Singh

Project ID: ace_04_dec

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Overview

Timeline

- Project provides fundamental research to support DOE/Industry advanced engine projects.
- Project directions and continuation are evaluated annually.

Budget

- Project funded by DOE/VT:
FY08 – \$695k
FY09 – \$700k

Barriers

- Extend HCCI (LTC) operating range to higher loads.
- Improved understanding of in-cylinder processes.
- Control HC & CO emiss. at low loads.

Partners / Collaborators

- Project Lead: Sandia \Rightarrow John E. Dec
- Part of Advanced Engine Combustion (AEC) working group:
 - 15 Industrial partners: auto, engine & energy
 - 5 National Labs & Univ. of Wisconsin
- GM – bimonthly meetings & discussion
- Chevron – funds complementary project
- LLNL – 1) support kinetic-mechanism devel., 2) CFD modeling, & 3) cooperative project on detailed exhaust speciation.

Objectives

Project objective: to provide the fundamental understanding (science-base) required to overcome the technical barriers to the development of practical HCCI and HCCI-like engines by industry.

FY09 Objectives:

- Determine the development of natural thermal stratification in an HCCI engine, using planar-imaging thermometry.
 - Thermal-imaging diagnostic developed as part of this task.
- Evaluate the potential of intake boost for extending the high-load limit of HCCI by using EGR to control combst.-phasing advance – multi-year task.
 - FY09: Determine potential of boost with EGR for gasoline at rep. engine speed.
- Determine the performance of ethanol as a fuel for HCCI engines.
 - Conducted cooperatively with M. Sjöberg in the Advanced SI-Engine Fuels Lab.
- Support CFD modeling and the development/improvement of chemical-kinetic mechanisms for HCCI at LLNL \Rightarrow provide data and analysis.



Milestones

FY2008

- Complete analysis of detailed exhaust-gas speciation measurements for iso-octane. (February 2008) – Status: Completed
- Determine the potential benefits of EGR for reducing the maximum pressure-rise rate and extending the high-load HCCI limit. (August 2008) – Status: Completed

FY2009

- Determine the magnitude and distribution of the natural thermal stratification in an HCCI engine at a typical operating condition. (February 2009) – Status: Completed.
- Determine the potential of EGR for increasing the allowable intake-pressure boost for gasoline-like fuels, at a representative engine speed. (August 2009) – Status: ~60% complete as of March 2009.

Approach

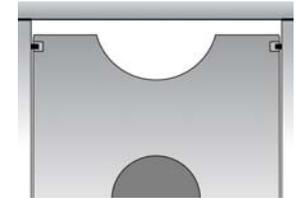
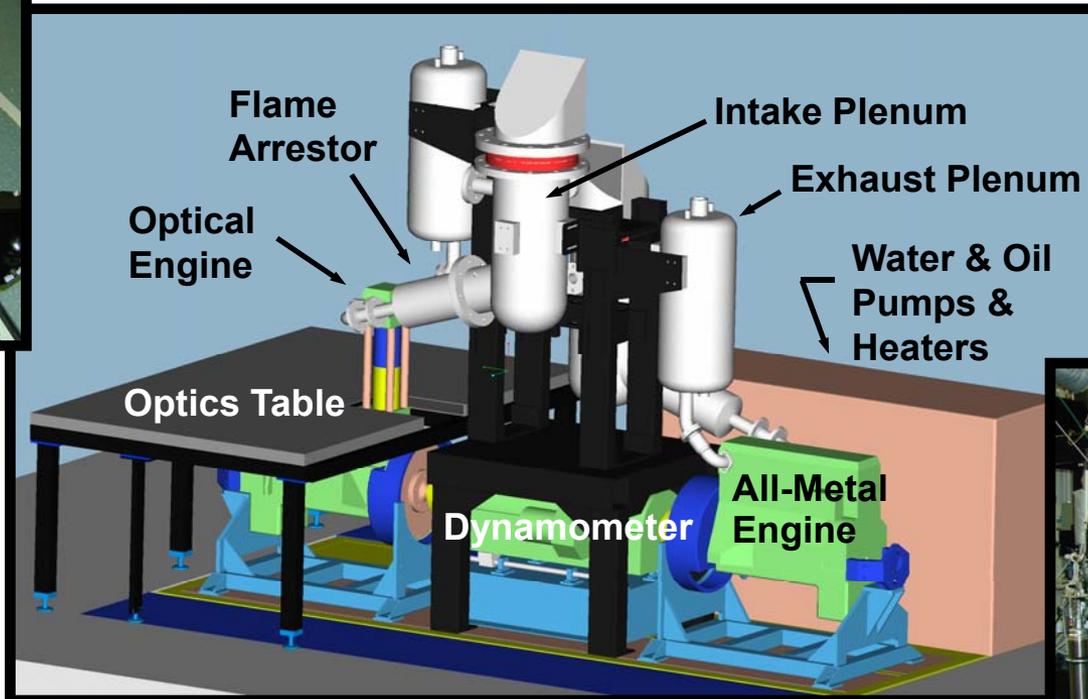
- Use a combination of metal- and optical-engine experiments and modeling to build a comprehensive understanding of HCCI processes.
- Metal engine \Rightarrow design well-characterized experiments to isolate specific aspects of HCCI/SCCI combustion & relationships between parameters.
 - Intake boost: Systematically increase boost \Rightarrow adjust T_{in} and/or EGR to retard timing to allow max. fueling at each P_{in} without knock, but with good stability.
- Optical engine \Rightarrow detailed investigations of in-cylinder processes.
 - Thermal stratification (TS): Develop temperature-imaging diagnostic \Rightarrow Apply to obtain T-map images showing temporal and spatial development of TS.
- Computational Modeling \Rightarrow supplement experiments by showing cause-and-effect relationships that are not easily measured.
 - Initiating LES modeling with J. Oefelein, Sandia to understand mechanism of TS.
 - In-house CHEMKIN (Senkin) single- and multi-zone kinetic modeling.
 - Collaborate with LLNL to improve kinetic mechanisms, and on CFD modeling.
- Combination of techniques provides a more complete understanding.
- Transfer results to industry.

Sandia HCCI / SCCI Engine Laboratory

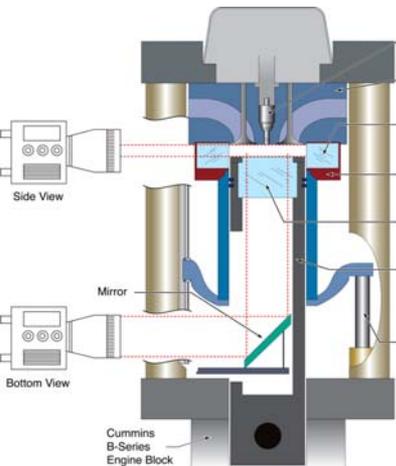
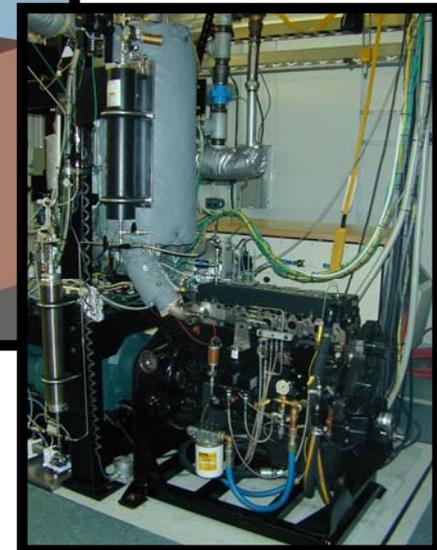


Optical Engine

- Matching all-metal & optical HCCI research engines.
 - Single-cylinder conversion from Cummins B-series diesel.



All-Metal Engine



- Bore x Stroke = 102 x 120 mm
- 0.98 liters, CR=14

Accomplishments

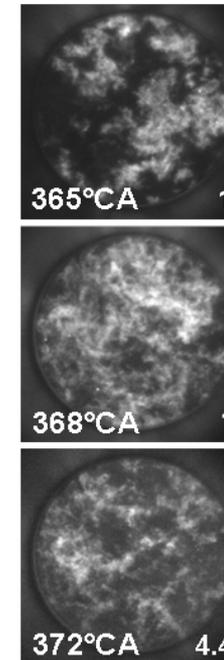
- Determined the evolution of natural thermal stratification in an HCCI engine, including its distribution and magnitude at a typical operating condition.
 - Developed a planar temp.-imaging diagnostic for TS in HCCI engines.
- Conducted initial investigation showing the potential of intake boost for extending the high-load limit of HCCI for gasoline fuel.
 - Showed that EGR is effective for controlling boost-induced timing advance.
 - Achieved a substantial load increase at a rep. 1200 rpm operating condition.
- Determined the behavior of ethanol as an HCCI fuel over a range of operating conditions.
 - Cooperatively with M. Sjöberg of the Advanced SI-Engine Fuels Lab.
- Initiated detailed exhaust-speciation analysis for PRF80 \Rightarrow 2-stage ignition.
 - Project conducted in cooperation L. Davisson at LLNL.
- Supported chemical-kinetic and CFD modeling work at LLNL.
 - Provided data and analysis for: 1) improving chemical-kinetic mechanisms, and 2) CFD modeling of fuel stratification to improve low-load comb. eff. & emissions.

Importance of Thermal Stratification (TS)

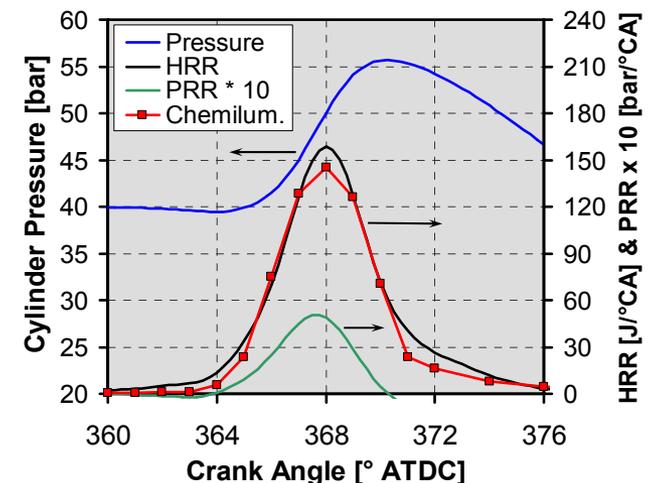
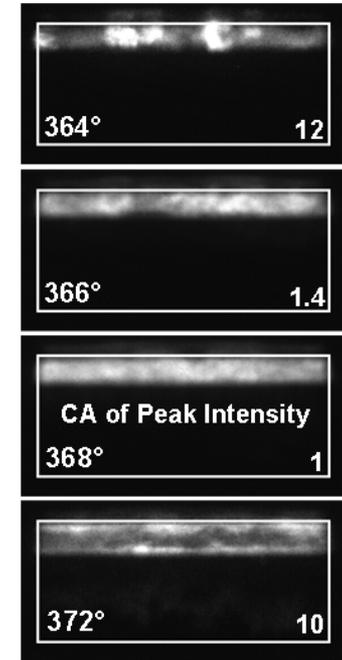
- TS causes autoignition to occur sequentially from hottest region to coldest.
 - Reduces max. pressure-rise rate (PRR).
 - Allows higher fueling without knock.
- Amplify the benefit of the TS by retarding combust. timing \Rightarrow further increases in load.
- Chemilum. images show:
 - Non-uniformities over whole field of view.
 - Hot reactions start intermittently near the mid-plane.
 - At time of max. PRR most combustion is from bulk gases (central region).
 - BL combust. occurs after max. PRR.

- TS of the bulk gas is critical for high-load HCCI operation.
- Understanding TS is important for increasing the high-load limit of HCCI.

Bottom-View

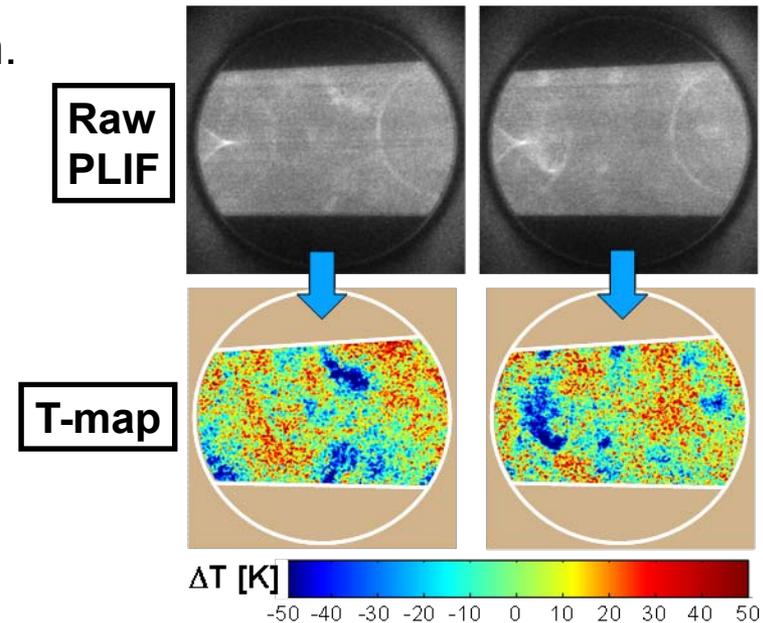
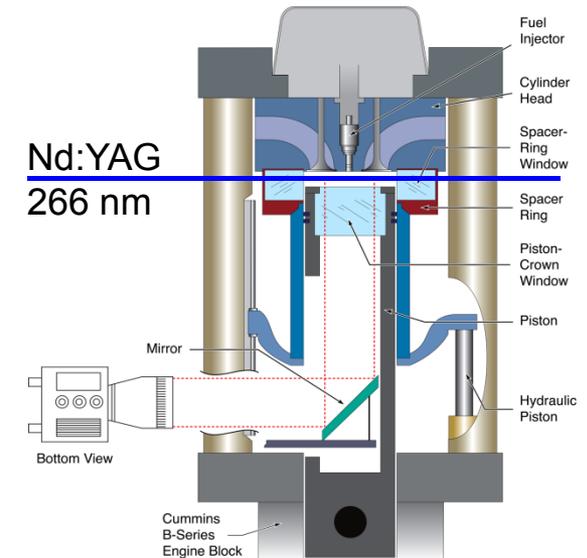


Side-View



Planar Imaging Thermometry

- Diagnostic: single-laser toluene PLIF.
 - PLIF intensity varies with temperature.
 - Good sensitivity in desired range, 600 – 1050 K.
- PLIF setup:
 - 2% toluene + 98% iso-octane
 - Laser excitation: 266 nm, 58 mm wide sheet.
 - Intensified camera with 277nm LP & UG5 filters.
 - Run inert with N₂ to prevent quenching.
 - ⇒ OK since TS develops prior to combustion.
- Calibrate temp. sensitivity in-cylinder.
- For well-mixed fueling, variations in PLIF intensity correspond to temp. variation
 - Temperature fluctuations shown relative to the mean of each image.
 - Bright regions in raw image correspond to cold pockets.

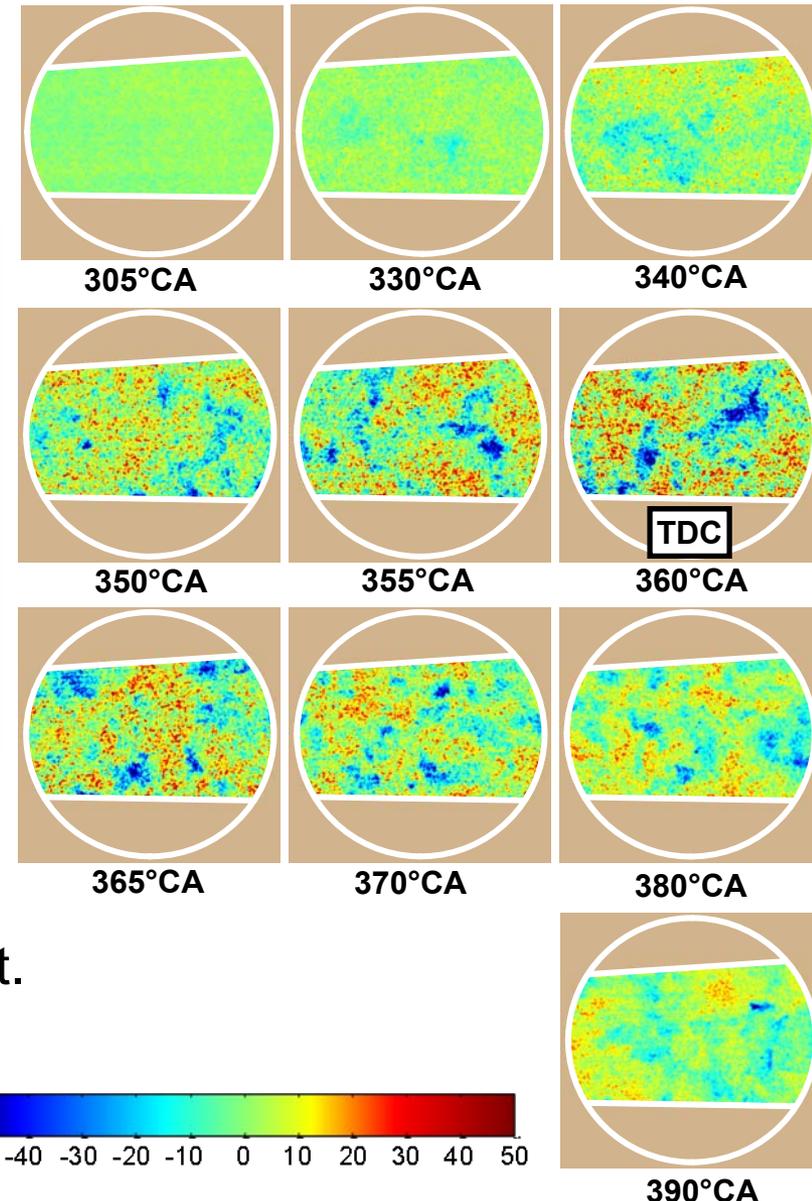


Temporal Evolution of TS

- Laser elevation adjusted with crank angle to remain in mid-plane (20 - 4 mm below F-D).
 - Representative of bulk gas.

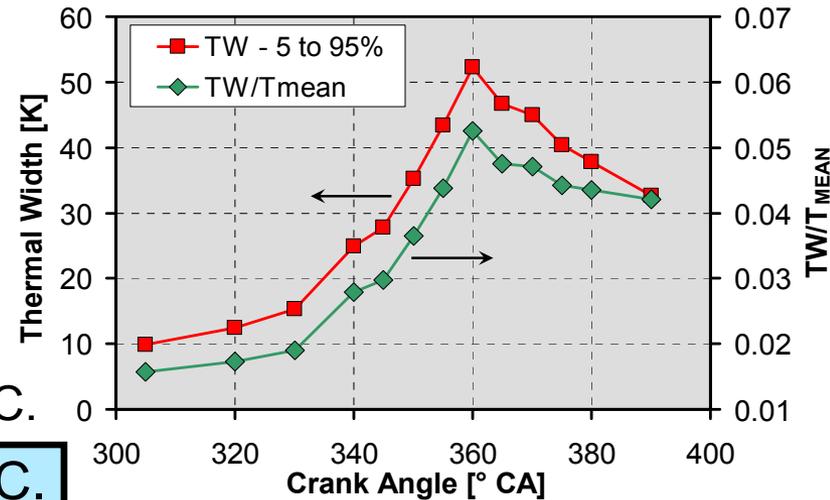
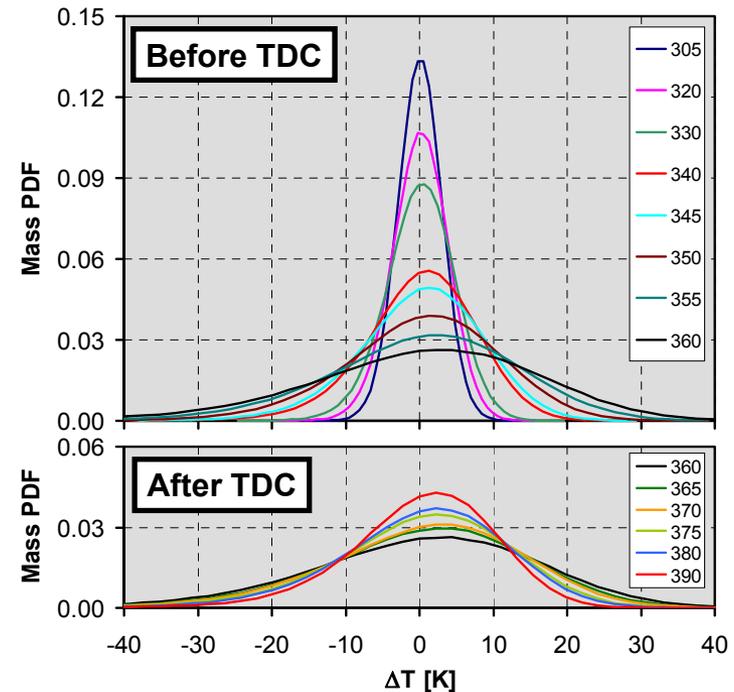
- TS develops progressively as cold pockets convected into central region.
 - Temperature nearly uniform at 305°CA.
 - > Virtually no TS remains from intake.
 - > Insufficient time with $T_{\text{gas}} > T_{\text{wall}}$.
 - Substantial TS by TDC (360°CA) +/- 35 K.
 - > Sufficient for significant spread in autoignition time of various regions.

- TS distribution is random cycle-to-cycle.
- Scale of cold pockets near TDC is 5–11 mm, similar to 8mm TDC clearance height. (Fine-grain speckle pattern is shot noise.)
- Magnitude of TS appears to diminish after TDC.



PDF Analysis of TS Images

- Apply probability density functions (PDF) to quantify changes in temp. distribution with crank angle.
 - 305°CA: PDF very narrow.
 - > Little time for development of TS.
 - > Analysis \Rightarrow almost all width is shot noise.
 - 330 - 340°CA: Significant broadening.
 - 340 - 360°CA: Progressive increase.
 - 360 - 390°CA: PDF width decreases, in agreement with images. \Rightarrow cause?
- Define: thermal width (TW) = 5-95% of PDF width.
 - Max. TW at TDC \approx 50 K \Rightarrow agrees with multi-zone model results.
- Normalize by T_{MEAN} to remove effects of compression & expansion.
 - Reduces, but not eliminate TW decrs. ATDC.
- Heat transfer dominant \Rightarrow changes at TDC.

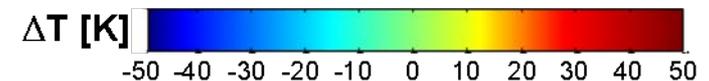
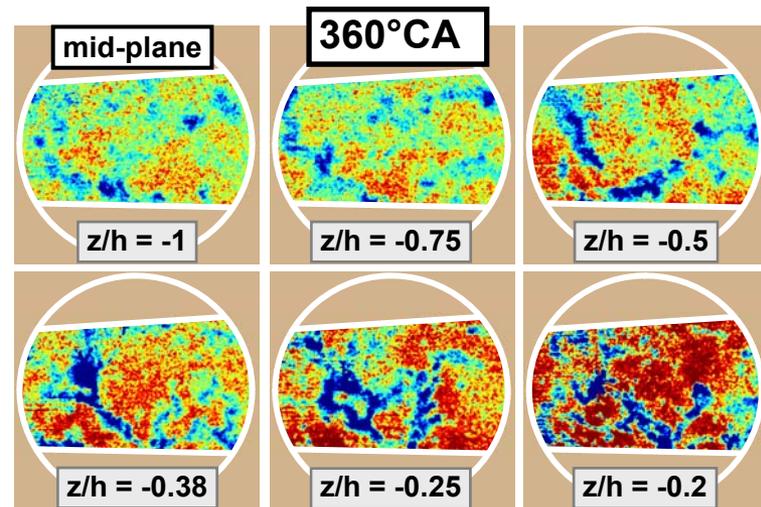
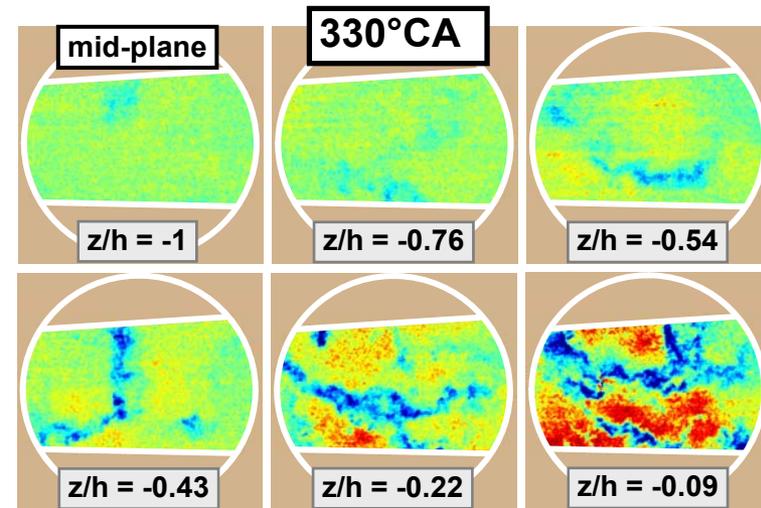
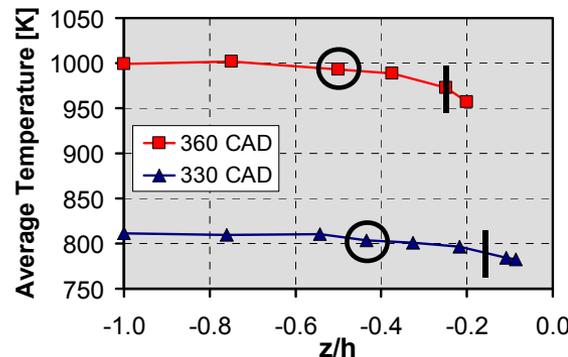


Thermal Distribution of Boundary Layer (BL)

- Incrementally scan laser from mid-plane to outer BL (to 0.8 mm below firedeck).
- 330°CA: bulk-gas temp. is nearly uniform.
 \Rightarrow Significant TS only for $z/h \geq -0.43$.
- 360°CA: TS developed throughout bulk gas.
 \Rightarrow TS greater in outer BL, $z/h \geq -0.5$.
- Avg. T profiles also show deficits for these z/h .

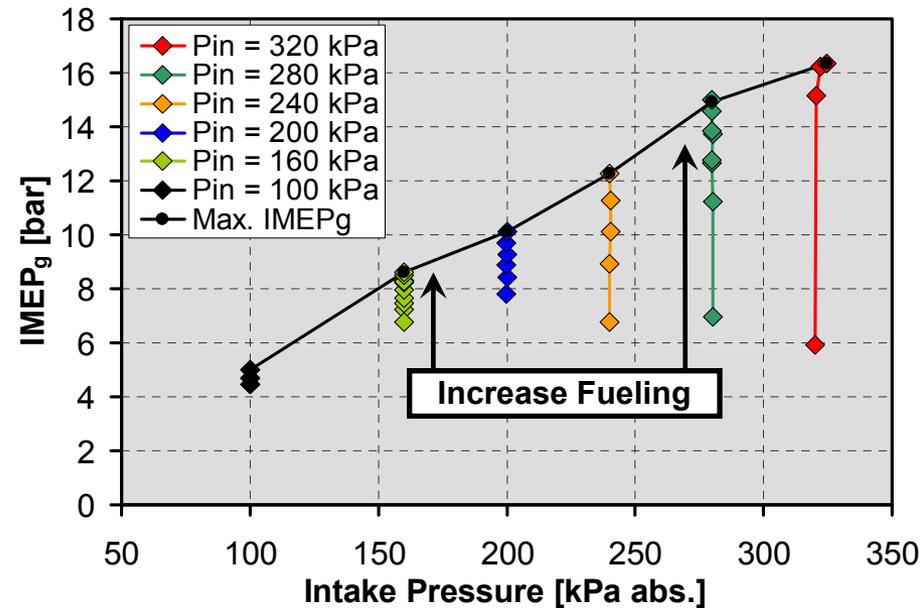
- TS progresses inward from wall, $330^\circ \rightarrow 360^\circ$
- Most BL temp. deficit occurs in last 0.8 mm at the wall \Rightarrow drops to $T_{\text{wall}} \approx 400$ K.

- BL thickness based on a 5% deficit from centerline value
 \Rightarrow 1-1.5mm.
- Agrees with previous chemilum. & PLIF studies.



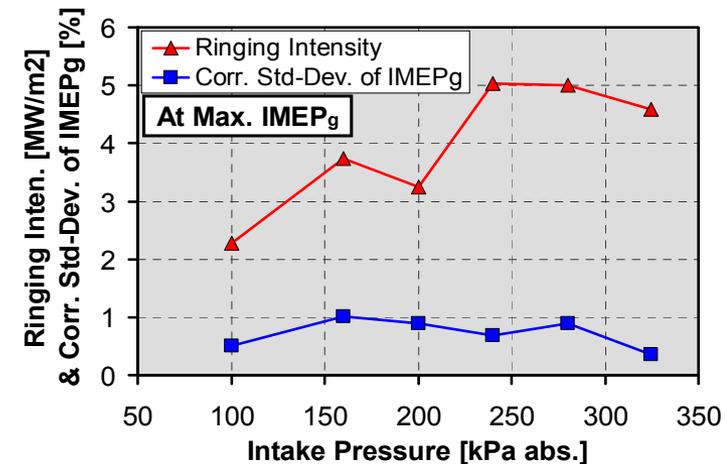
Intake Boost for Extending High-Load Limit

- Investigate the potential of boosting for extending HCCI to higher-loads.
 - Required to match full-load diesel or SI.
- Current work: gasoline, 1200 rpm.
- Boost enhances autoignition \Rightarrow advances comb. timing \Rightarrow Knock!
 - Compensate with reduced T_{in} .
 - For $P_{in} > 160$ kPa, $T_{in} \rightarrow T_{amb}$ \Rightarrow limits allowable fueling.



- Add cooled EGR to further slow autoignition.

- Achieved **IMEP_g = 16.3 bar**, $P_{in} = 324$ kPa.
 - Very high IMEP_g for HCCI/LTC, convent'l fuel.
 - Near stoich., C/F = 38.5, EGR = 60%, $P_{exhaust} = 326$ kPa, $T_{exhaust} = 407^\circ\text{C}$.
- Ringings ≤ 5 MW/m², No Knocking.
- Std-Dev of IMEP_g $\leq 1\%$, very good stability.

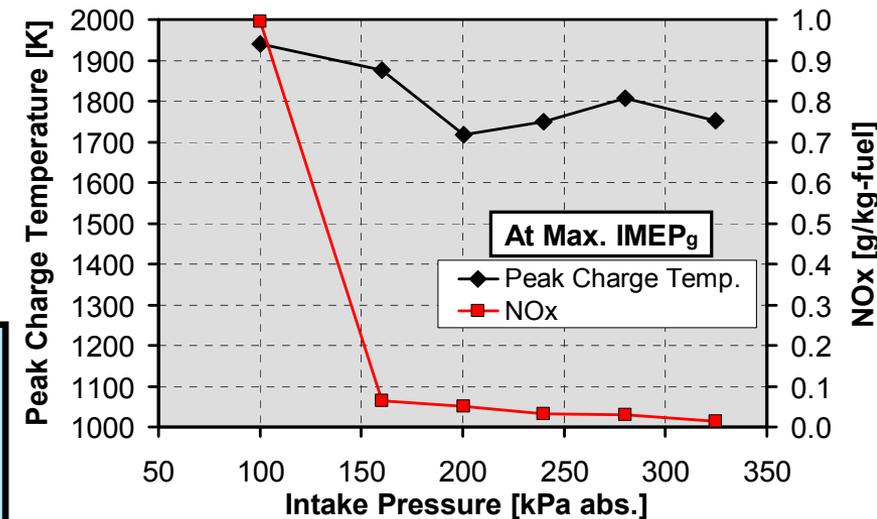
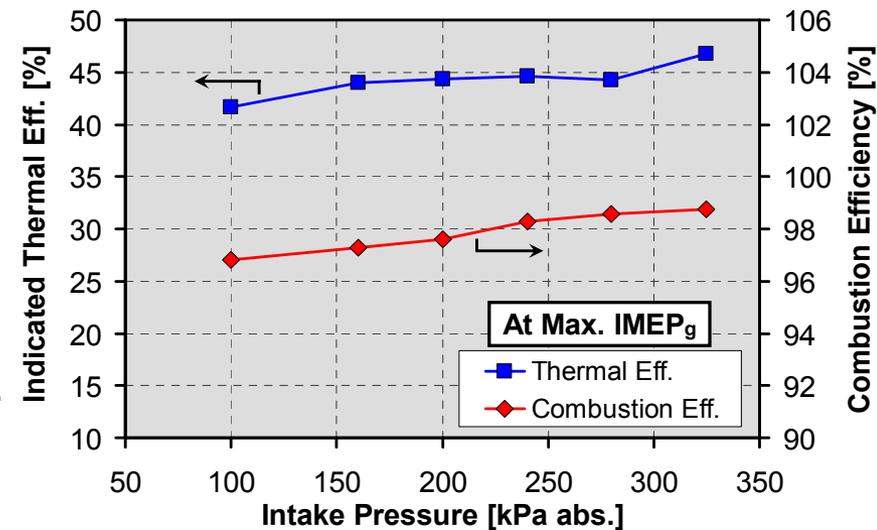


Efficiency and NOx for Boosted High-Load

For maximum IMEP_g at each boost

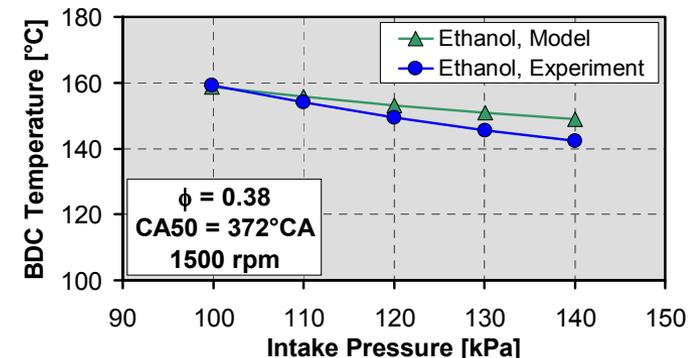
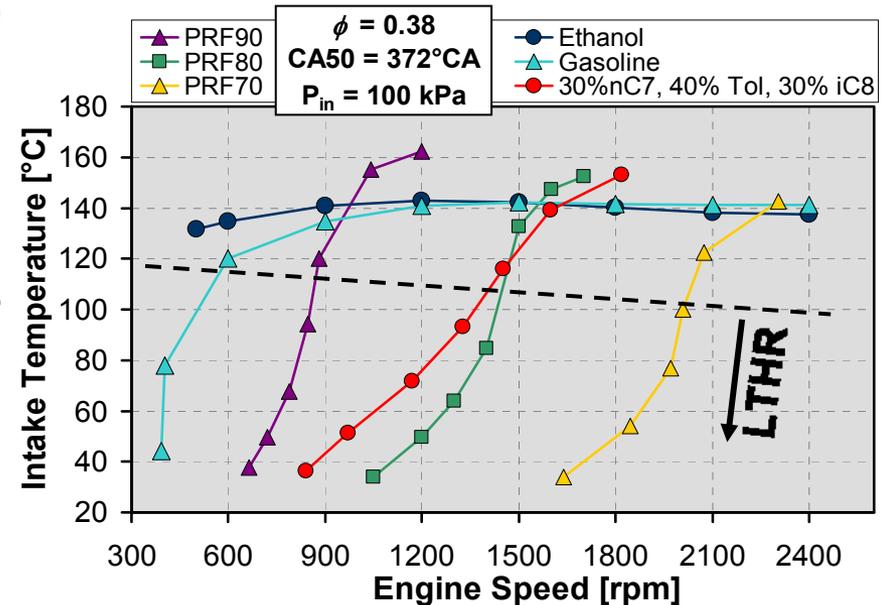
- Indicated Thermal Eff. increases slightly with boost. \Rightarrow Th. Eff. \sim 45%.
- Combustion Eff. increases, 97 \rightarrow 99%.
 - Higher wall temps. \Rightarrow improve combst.
 - Increased EGR reduces HC & CO emiss.
- NOx emissions below US-2010 stds. (should also meet tier II, bin 5).
 - Extremely low for all boosted cases (< 0.1 g/kg-fuel, \sim 1-2 ppm).

- For max. IMEP_g = 16.3 bar, P_{in} = 3.2 bar.
 - Ind. Thermal Eff. = 47%
 - Comb. Eff. = 99%
 - NOx = 0.015 g/kg-fuel, T_{peak} = 1750 K.



Ethanol as an HCCI Fuel

- Ethanol is a component in most pump gasoline (0 – 15% fraction).
 - Also being considered as an alternative fuel at levels up to 85%.
- Important to understand ethanol's potential for HCCI.
 - Ignition quality \Rightarrow RON = 107, MON = 89
 - Effect on performance and operating range, *i.e.* speeds, boost, etc.
- Speed sweep \Rightarrow CA50 = 372°CA
 - Autoig. similar to gasoline for RPM >900.
 - Most fuels: $T_{in} < 100^{\circ}\text{C}$ as speed is reduced \Rightarrow indicates LTHR (cool flame).
 - Ethanol shows no LTHR!
- Boost has only moderate effect on T_{in} .

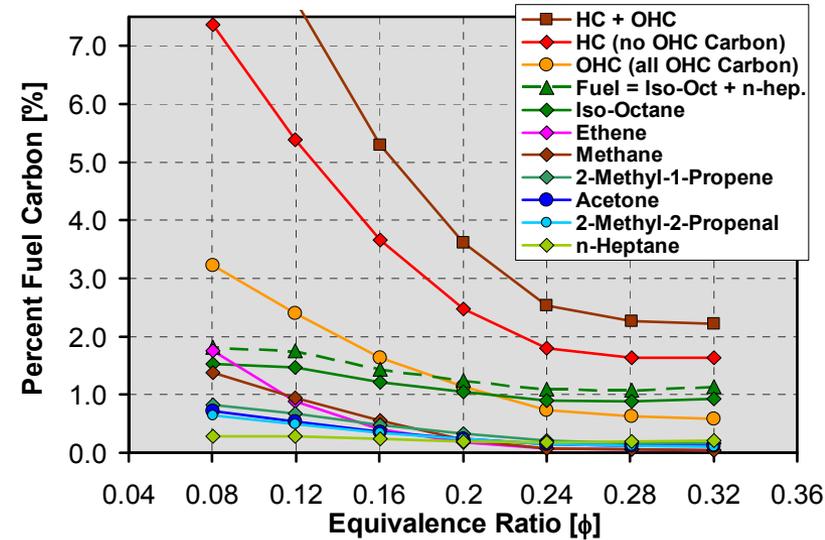


- Performs similarly to gasoline, conds. studied.
- Good potential for HCCI fuel / fuel-component.
- Ethanol is a true single-stage ignition fuel.
 - May offer advantages for control & boosted oper.

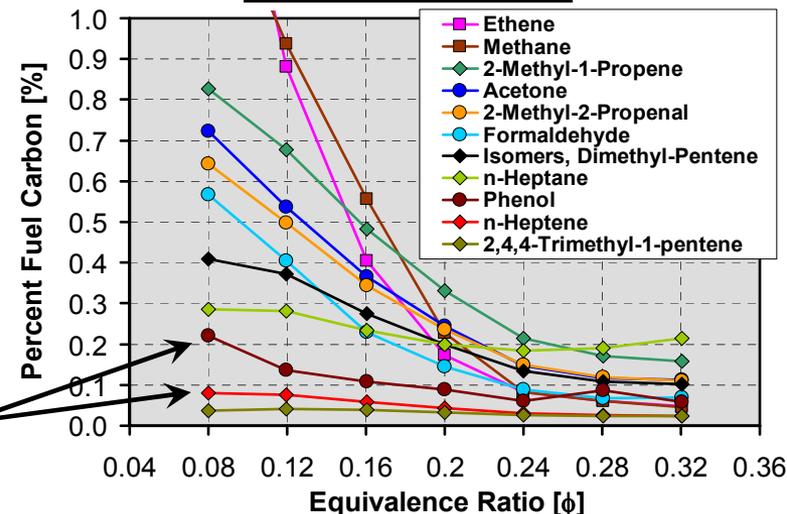
Detailed Exhaust Speciation – PRF80

- Joint project with LLNL \Rightarrow spec. analysis. Sandia \Rightarrow engine op. & data interpretation.
- Conducted fueling(ϕ)-sweep & data for near-misfire conditions. Results provide:
 - Data for aftertreatment & model validation.
 - Improved understanding of combst. process.
- PRF80 is a 2-stage ignition fuel at conditions studied. \Rightarrow Affects emissions compared to iso-octane & gasoline.
 - OHC fraction is greater for PRF80.
 - Unreacted-fuel fraction is much lower. \Rightarrow Cool-flame reactions incr. fuel breakdown.
- Ratio of *n*-heptane / iso-octane is 19-23%, compared to 25% in fuel.
 - *n*-Heptane breaks down more readily, but it induces substantial iso-octane breakdown.
- Relatively high conc. of *n*-Heptene & Phenol \Rightarrow former due to *n*-Heptane reactions.

Main Species



Selected Species



Future Work

- Complete investigation of intake boost for extending the high-load limit of gasoline-fueled HCCI at a representative speed, 1200 rpm (FY09).
- (FY10) Expand boost study to include a range of higher engine speeds, boost levels, and back-pressures for realistic turbo-charger efficiencies.
 - Two-stage fuels to be done as part of Chevron-funded project.
- Extend TS study: 1) improve diagnostic S/N & optical setup, 2) investigate methods of increasing TS, and 3) determine cause of flows producing TS.
 - Collaborate with J. Oefelein to apply LES modeling \Rightarrow mechanism/enhancement.
- Additional ethanol studies over a wide range of operating parameters: EGR, load, & boost to high levels \Rightarrow with M. Sjöberg, Adv. SI-Fuels Lab.
- Complete exhaust-speciation analysis for 2-stage ignition fuel, PRF80, and compare with single-stage ignition fuels, gasoline and iso-oct. \Rightarrow with LLNL.
 - Analyze emiss. species for near misfire with single- and two-stage ignition fuels.
- Continue to collaborate with LLNL on improving chemical-kinetic mechanisms and on CFD/kinetic modeling.

Summary

- Quantitative temperature-map images show that thermal strat. (TS) develops progressively during latter compression stroke \Rightarrow throughout charge.
- Data indicate that TS results from wall-heat transfer and convection.
 - Future work will focus on understanding the mechanism for bulk-gas TS, and potential methods for increasing it to increase the high-load limit of HCCI.
- EGR substantially improves boosted HCCI operation with gasoline fuel.
 - Achieved 16.3 bar IMEP_g, Ind. Thermal-Eff. = 47%, no Knock & no NO_x or PM.
 - Near high-load limit for conventional diesel. Shows significant potential for extending HCCI range – full time HCCI?
- Ethanol is a promising HCCI fuel. Performance is generally similar to gasoline, but no low-temp. (“cool-flame”) chemistry.
 - Possible advantages for control and for boosted op. \Rightarrow additional studies req’d.
- Detailed exhaust speciation of a two-stage ig. fuel (PRF80), shows significantly different behavior from single-stage fuels, iso-octane & gasoline.
 - More breakdown of fuel & fuel-like species to smaller species, higher OHC fract.