



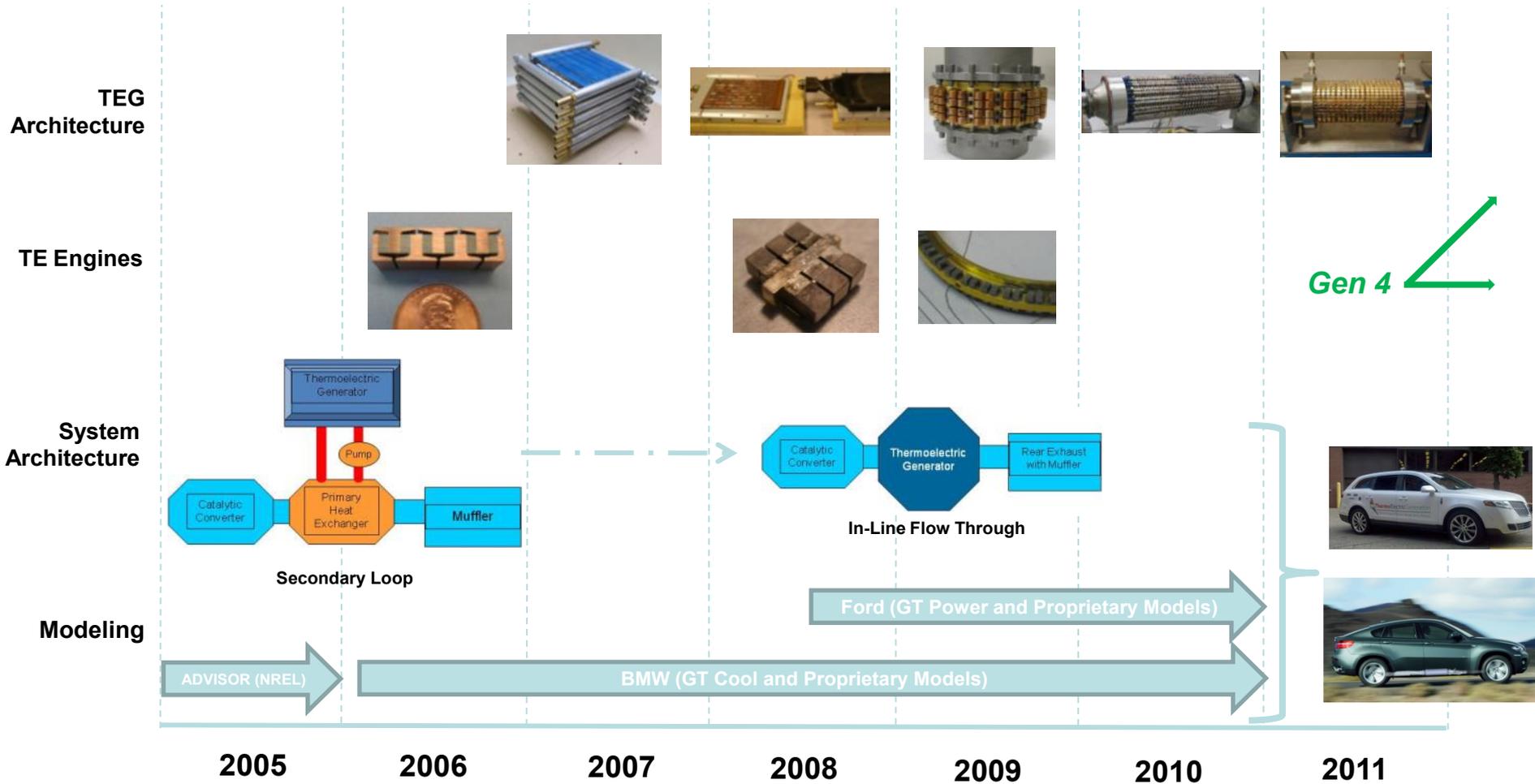
***Thermoelectric Generator  
Performance for Passenger  
Vehicles***

***Doug Crane and John LaGrandeur***

***March 20, 2012***



# Technology Summary



# Stack Design TE Engine

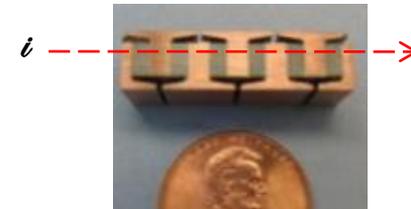
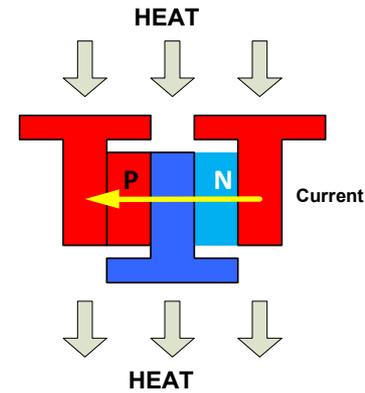
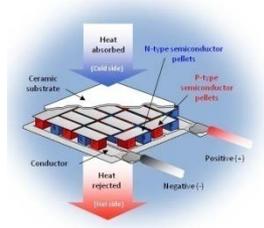
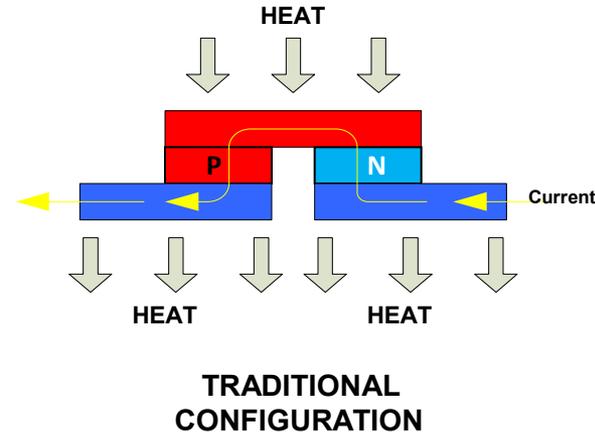
## Traditional TE module

architecture consists of n & p type elements arranged in a planar array.

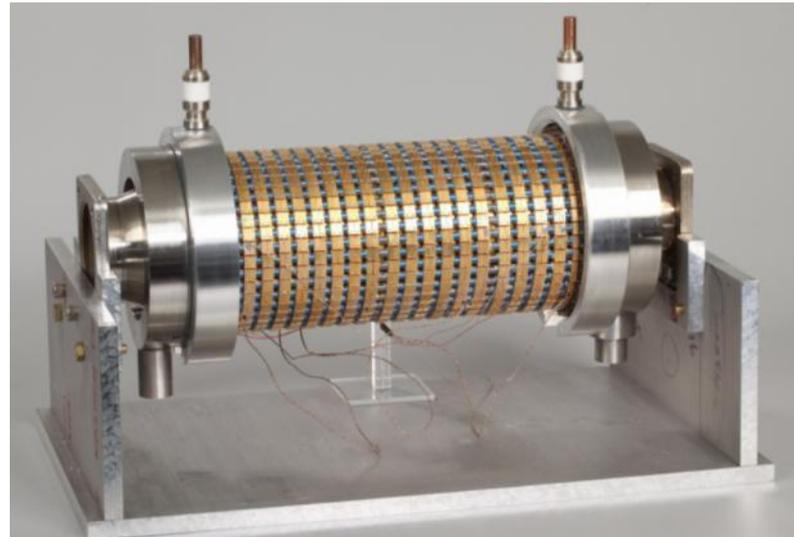
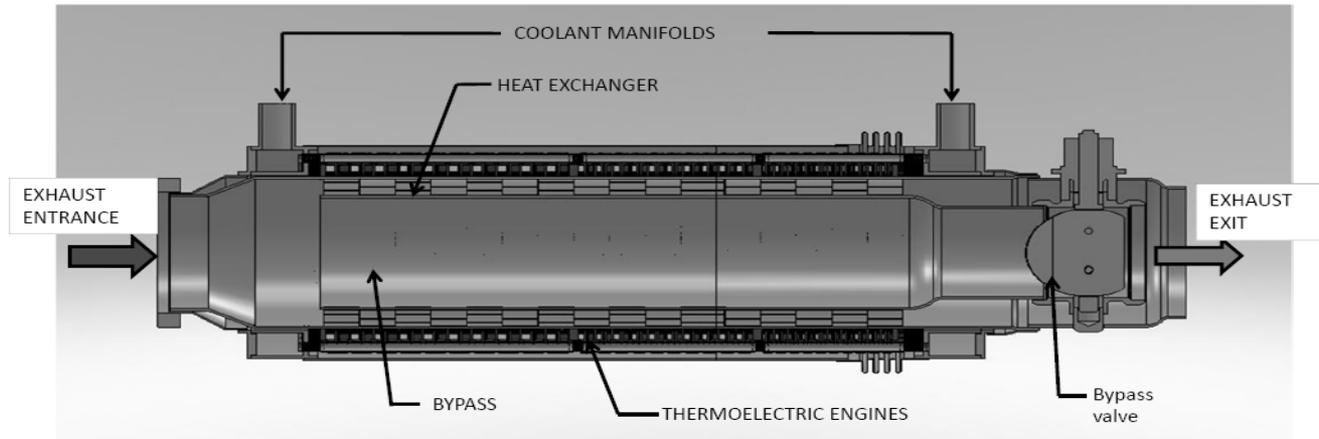
- Module size is constrained by thermally induced stress in the elements

## A stack design has been

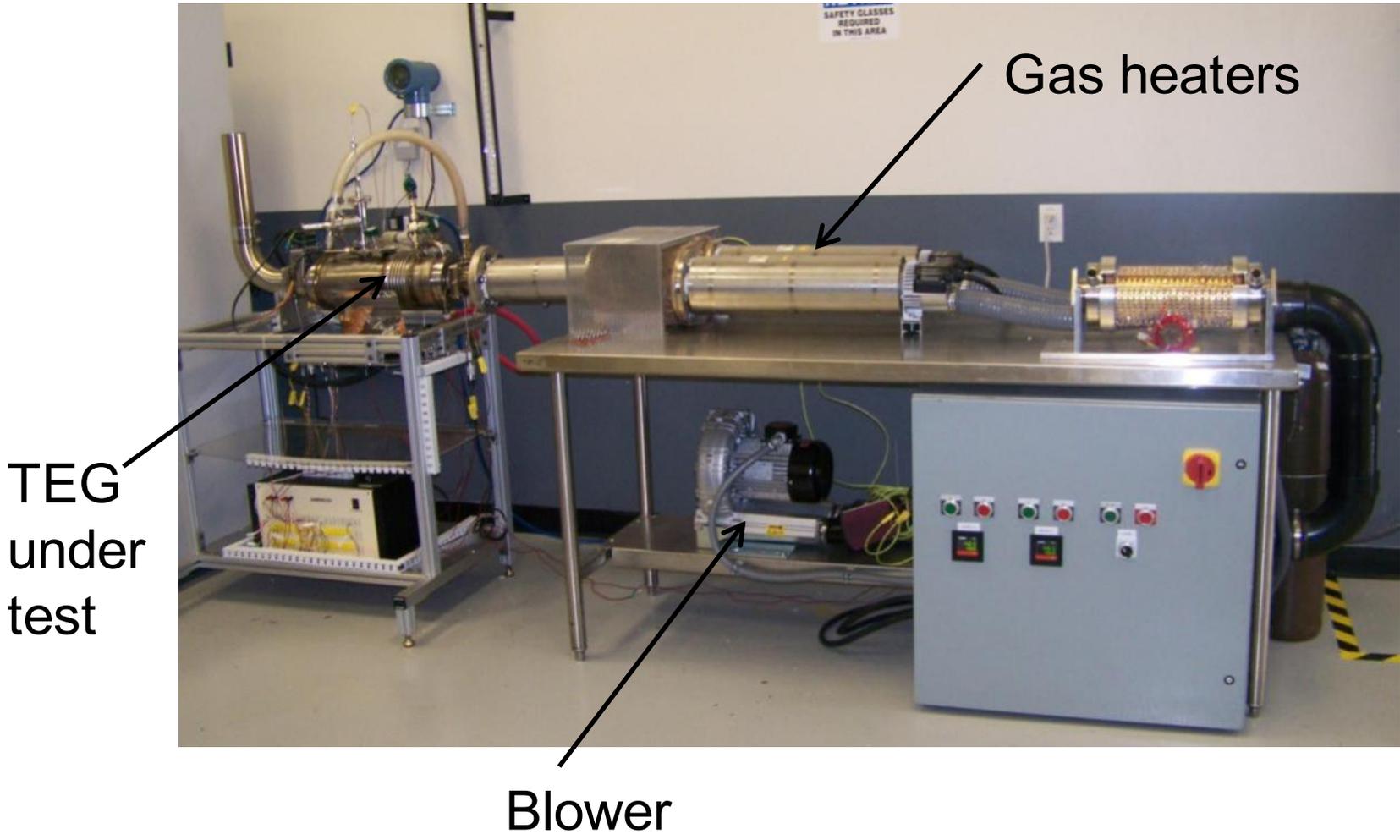
developed which enables shorter n & p leg lengths (less TE material = lower cost) and improved design flexibility.



# Cylindrical TEG



# Bench Test Setup



# Test Conditions for Medium Temperature TEG

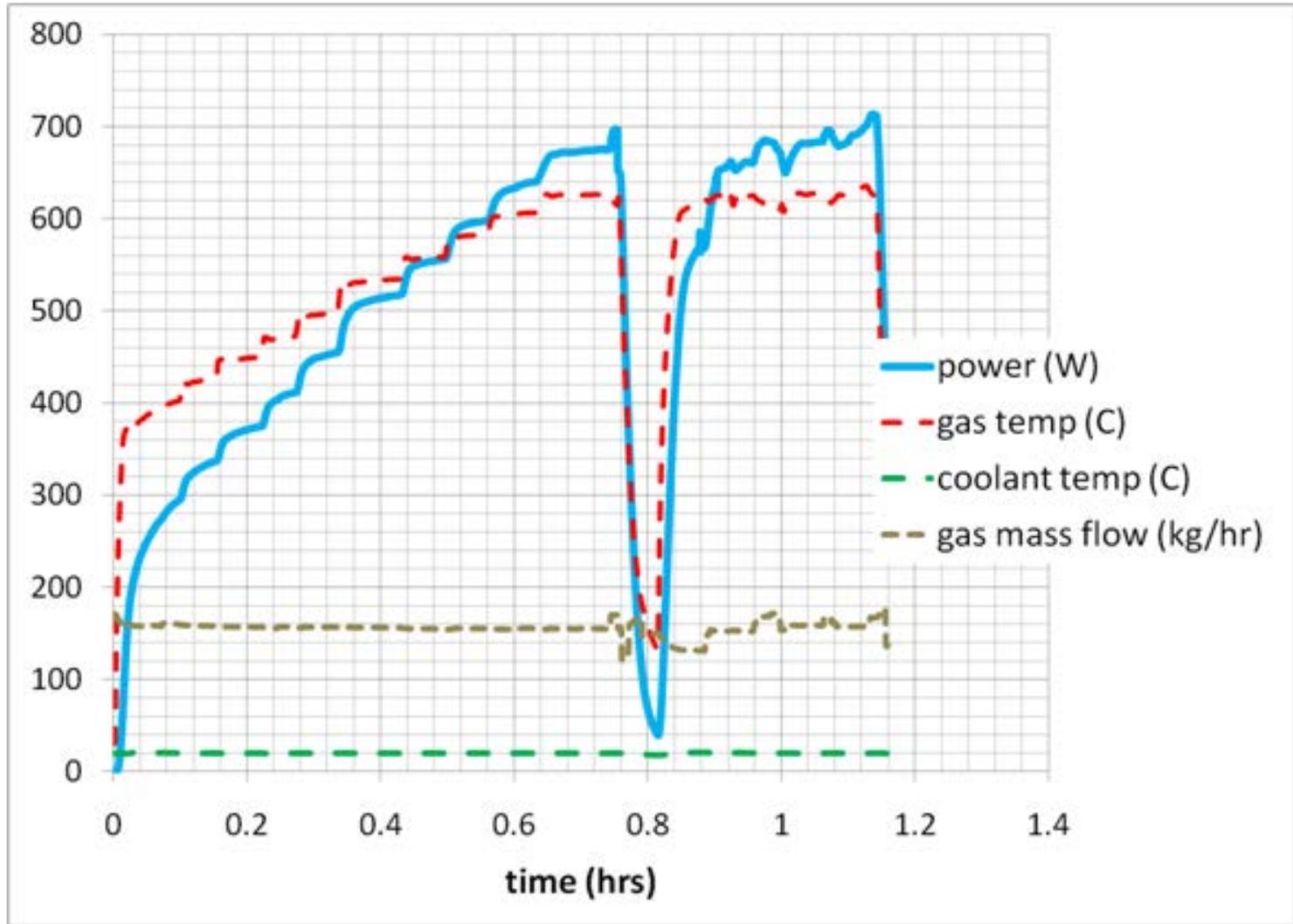
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Test	1	2	3	4	5	6	7	8	9	10	11	12
Tfh,in (C)	390	390	390	425	425	425	510	510	510	620	620	620
Tfc,in (C)	20	20	20	20	20	20	20	20	20	20	20	20
vdot,h (g/s)	13.5	13.5	13.5	20.5	20.5	20.5	30.1	30.1	30.1	45	45	45
vdot,c (g/s)	170	250	330	170	250	330	170	250	330	170	250	330
max power output (W)	56.1	56.5	57.6	119	121	122	261	270	272	495	580	595

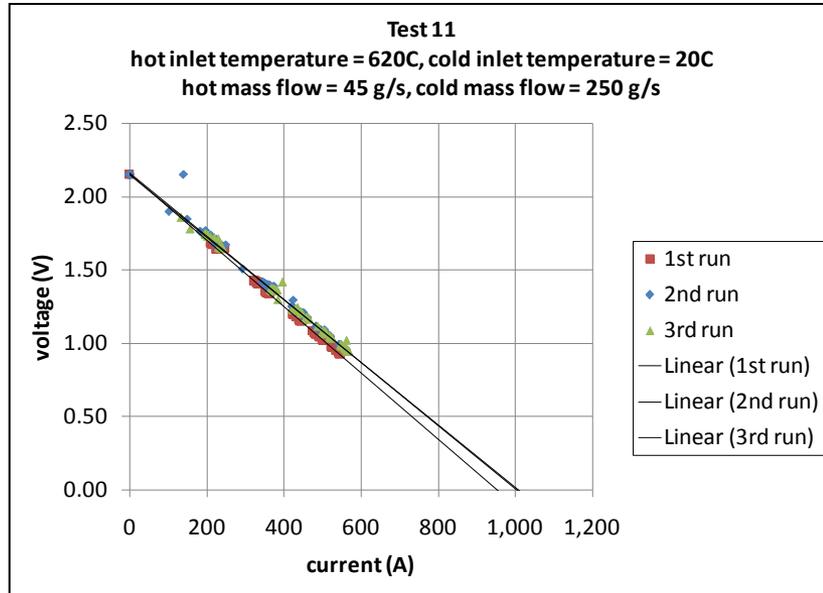
Test	13	14	15	16	17	18	19	20	21	22	23	24	25
Tfh,in (C)	390	390	390	425	425	425	510	510	510	620	620	620	620
Tfc,in (C)	40	40	40	40	40	40	40	40	40	40	40	40	20
vdot,h (g/s)	13.5	13.5	13.5	20.5	20.5	20.5	30.1	30.1	30.1	45	45	45	48
vdot,c (g/s)	170	250	330	170	250	330	170	250	330	170	250	330	330
max power output (W)	49.3	49.2	49.6	103	104	106	228	237	241	436	461	N/A	608

Note: Test 24 not completed due to the chiller overheating.

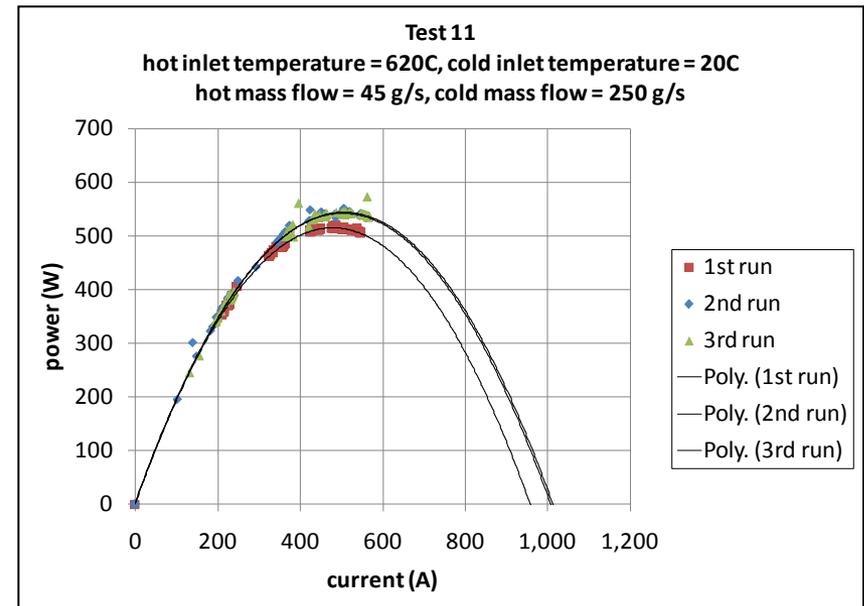
# Cylindrical TEG Performance



# Run Repeatability

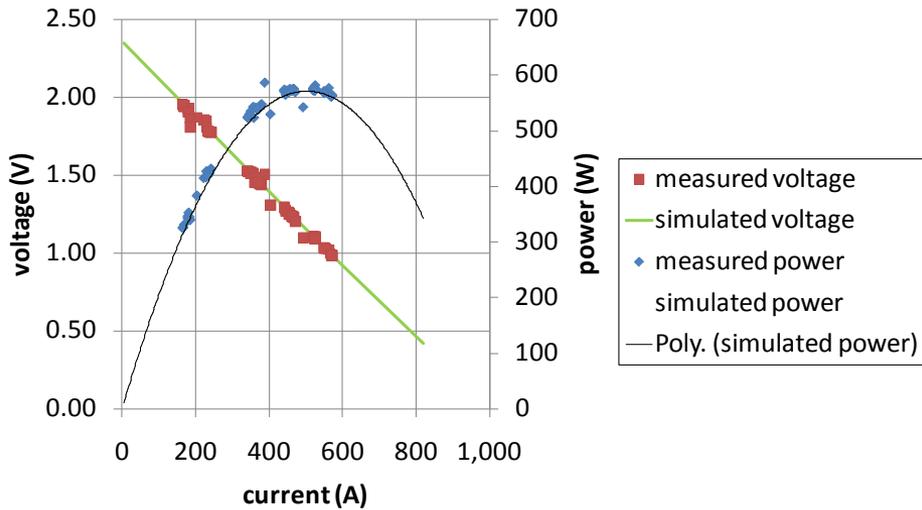


- The three runs were performed over a period of two weeks (over 25 hours of testing) and show good repeatability.
  - There is a 9% decrease in electrical resistance from run 1 to run 2 due to “settling in” of the device interfaces
  - This reduction in electrical resistance caused a 5% increase in peak power

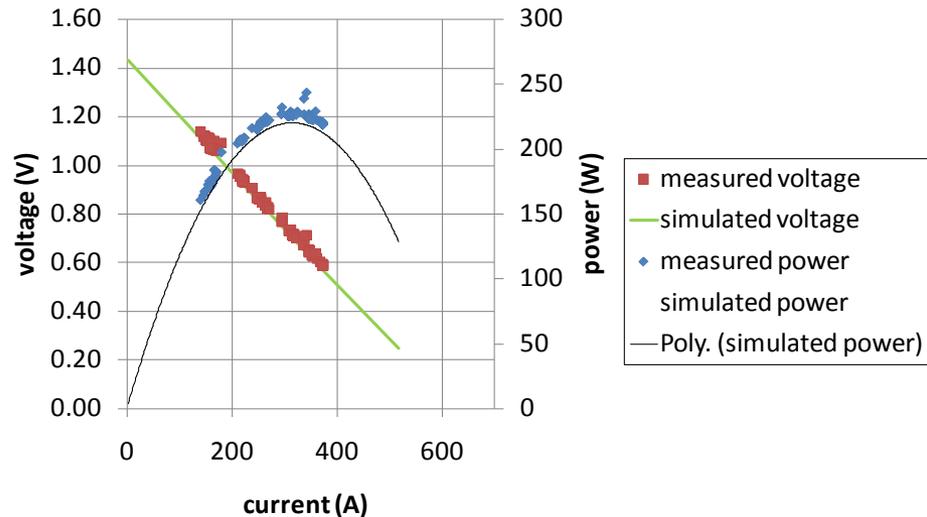


# Power & Voltage Validation

**TEG Performance - Test 11**  
(hot inlet temperature = 620C, cold inlet temperature = 20C)  
(hot mass flow = 45 g/s, cold mass flow = 250 g/s)  
(6/29/11)

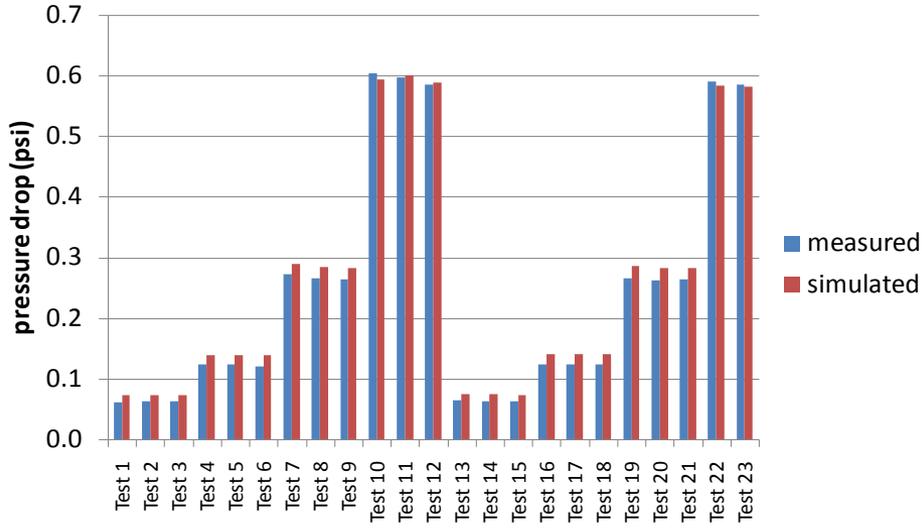


**TEG Performance - Test 19 (no first & last ring)**  
(hot inlet temperature = 510C, cold inlet temperature = 40C)  
(hot mass flow = 30.1 g/s, cold mass flow = 170 g/s)  
(6/29/11)

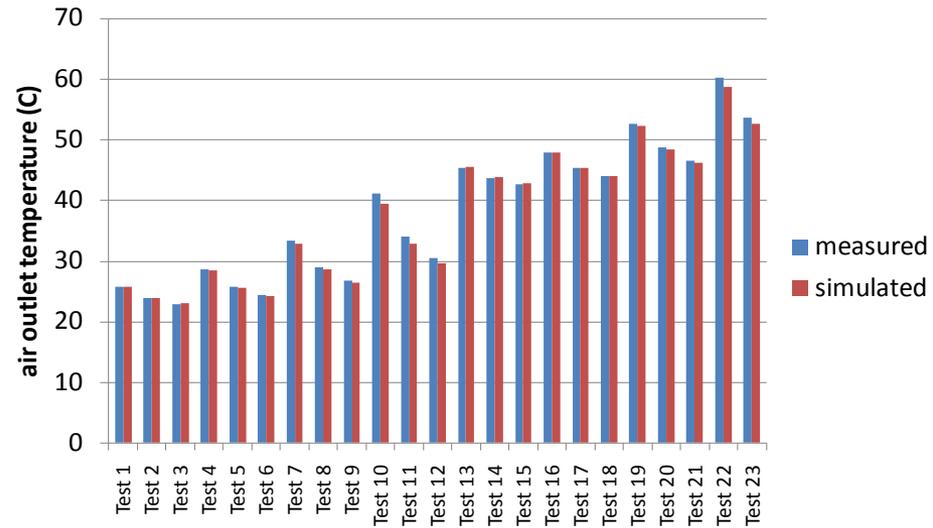


# Air Pressure Drop & Water Outlet Temperature Validation

**TEG Model Validation**  
**Air Side Pressure Drop**  
**7/11/11**

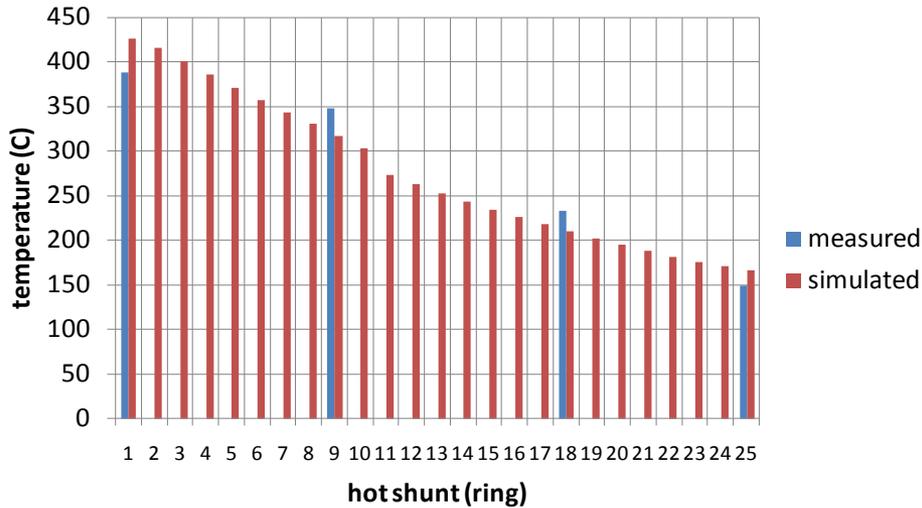


**TEG Model Validation**  
**Water Outlet Temperature**  
**7/11/11**

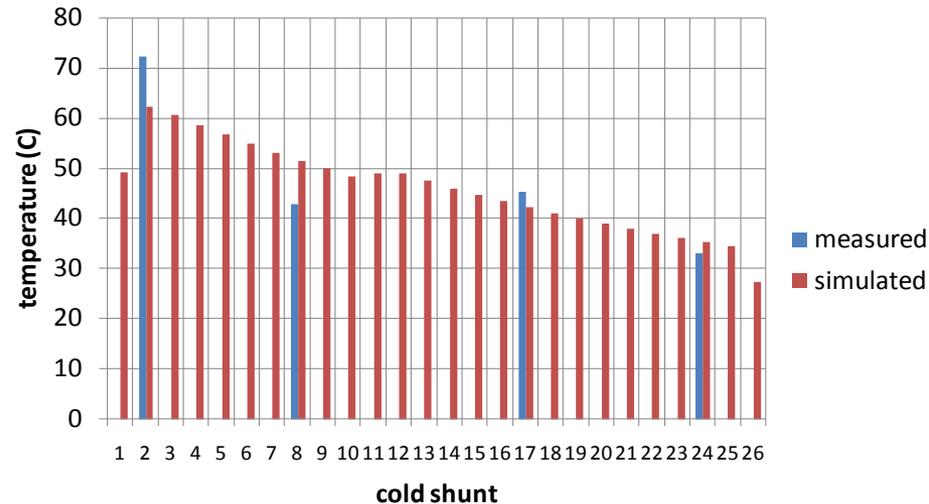


# Hot & Cold Shunt Temperature Validation

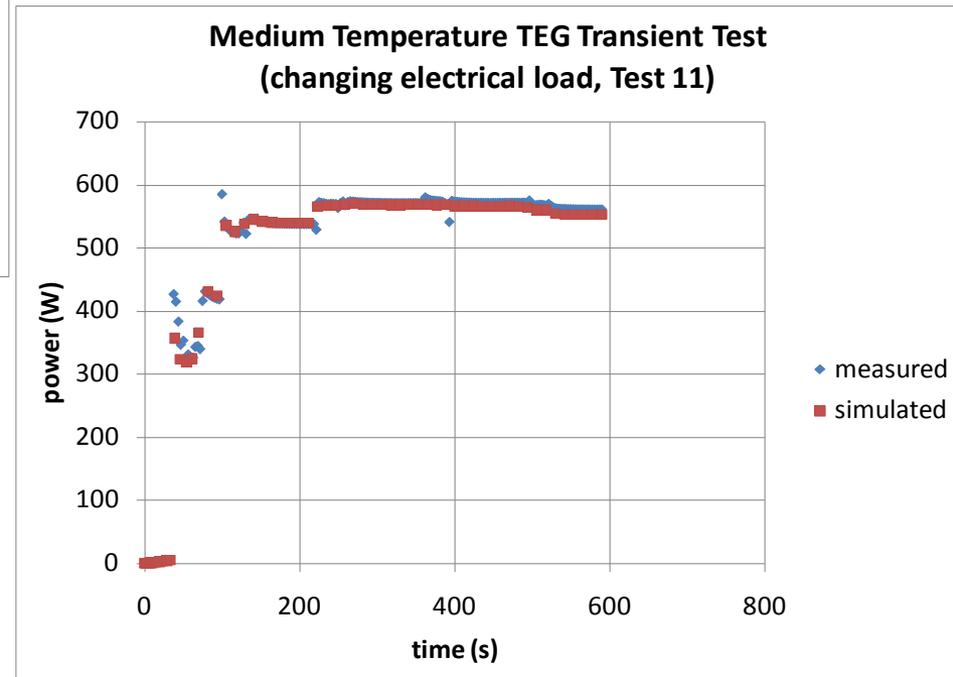
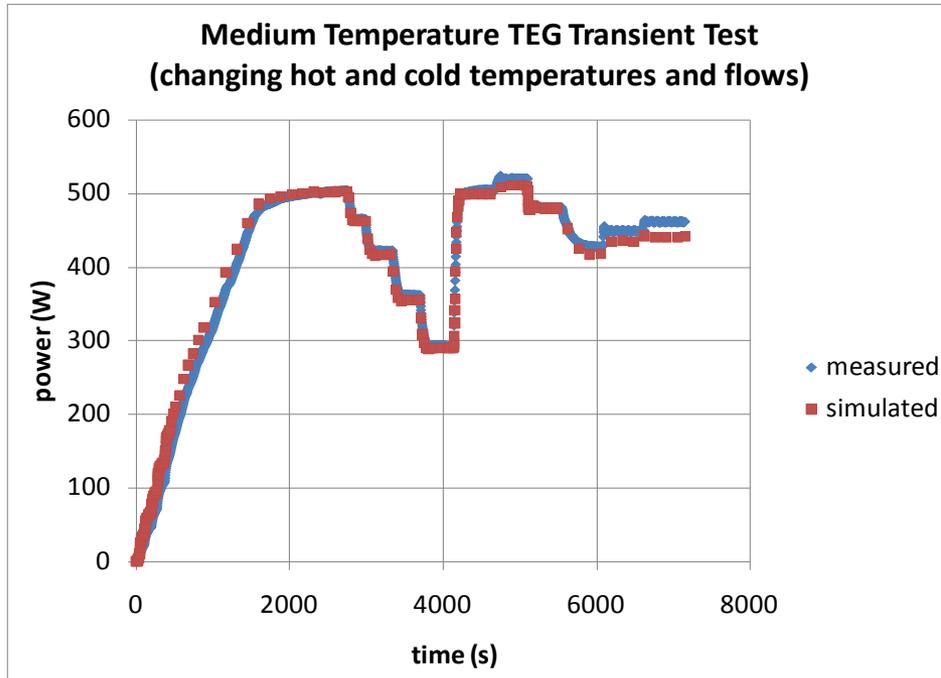
**TEG Model Validation - Test 11**  
(hot inlet temperature = 620C, cold inlet temperature = 20C)  
(hot mass flow = 45 g/s, cold mass flow = 250 g/s)  
(6/29/11)



**TEG Model Validation - Test 11**  
(hot inlet temperature = 620C, cold inlet temperature = 20C)  
(hot mass flow = 45 g/s, cold mass flow = 250 g/s)  
(6/29/11)

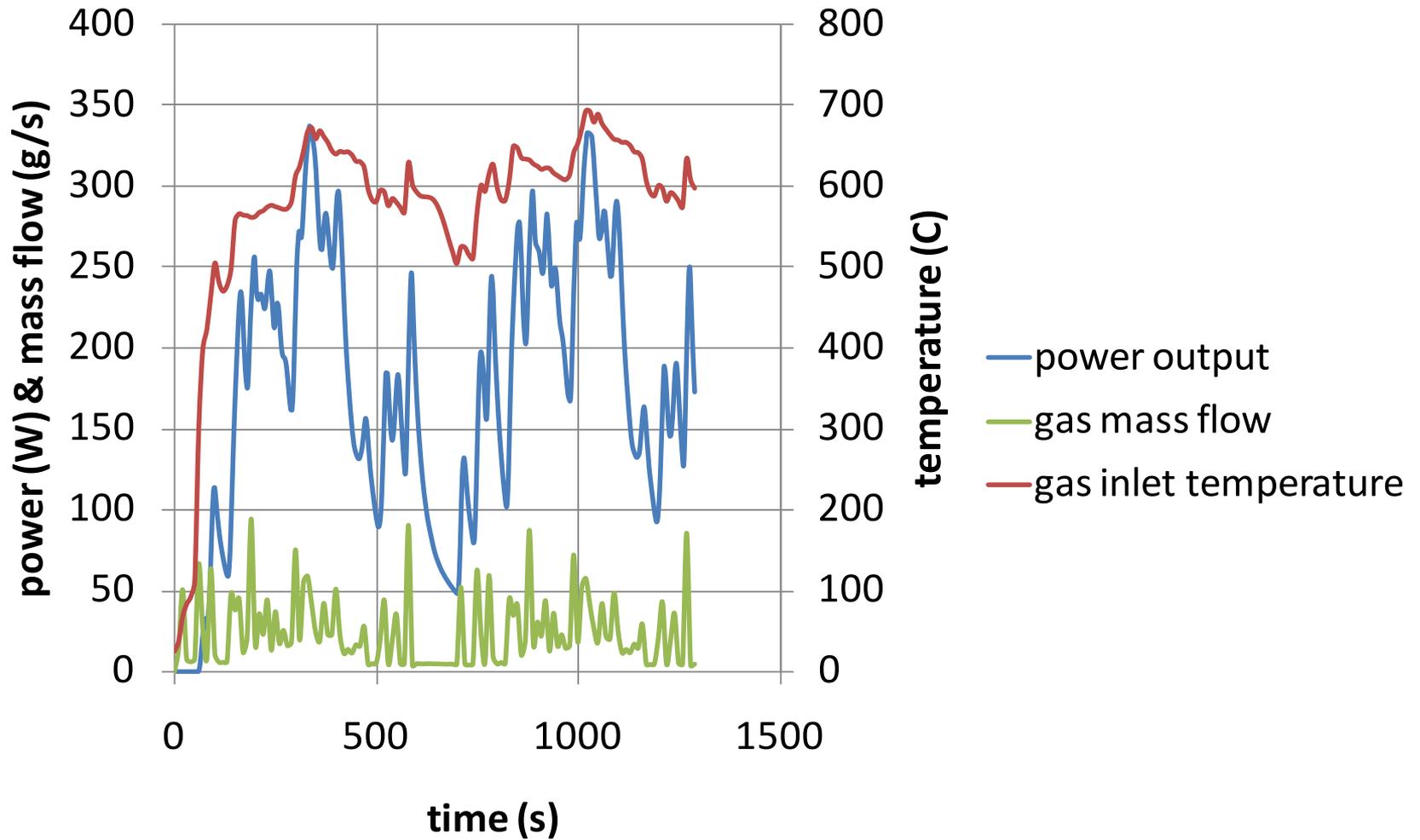


# Medium Temperature TEG Transient Model Validation



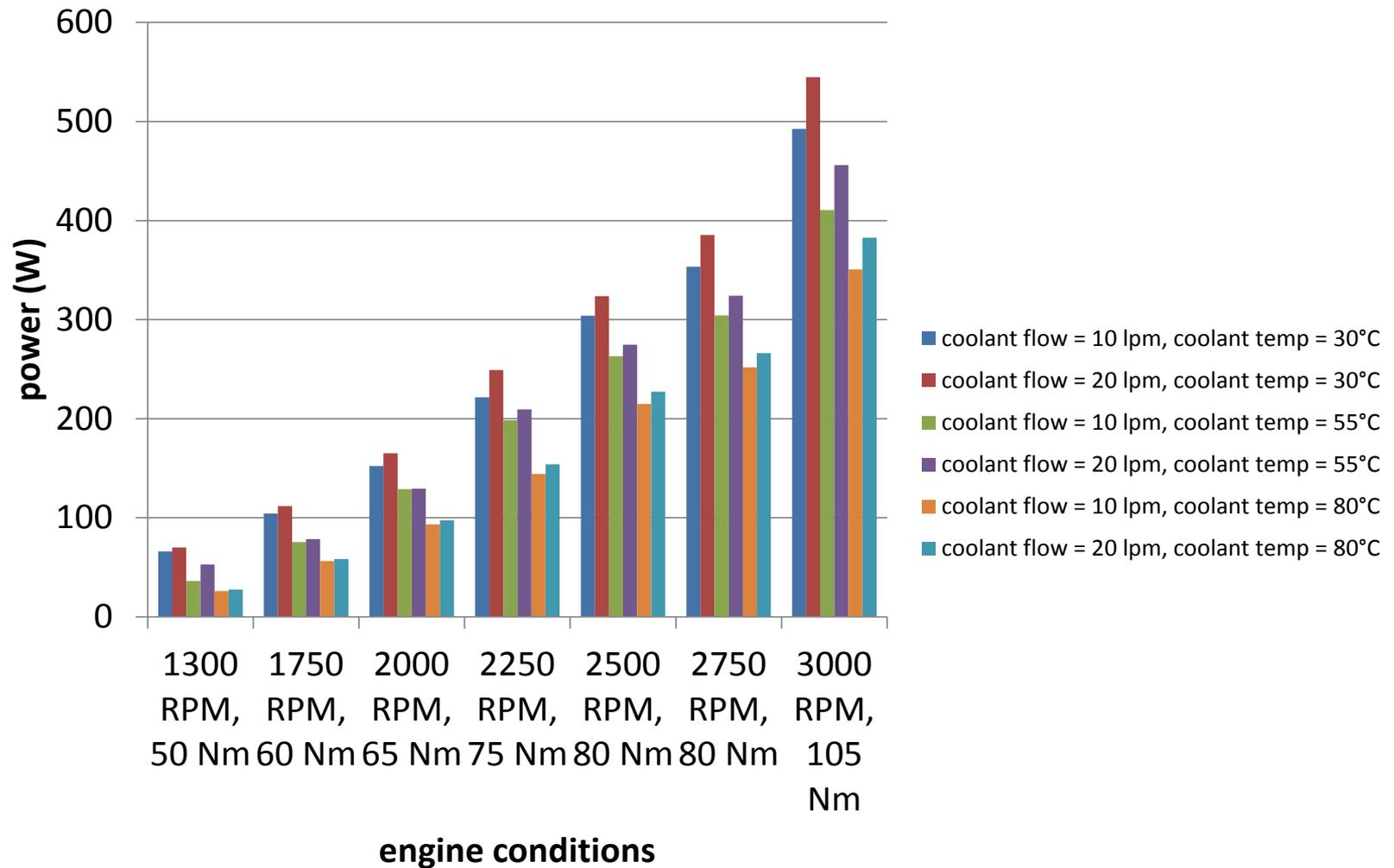
# US06 Simulated TEG Performance

Simulated TEG Performance for US06 Drive Cycle  
(cold inlet temperature = 80C, cold flow = 250 g/s)  
7/15/11



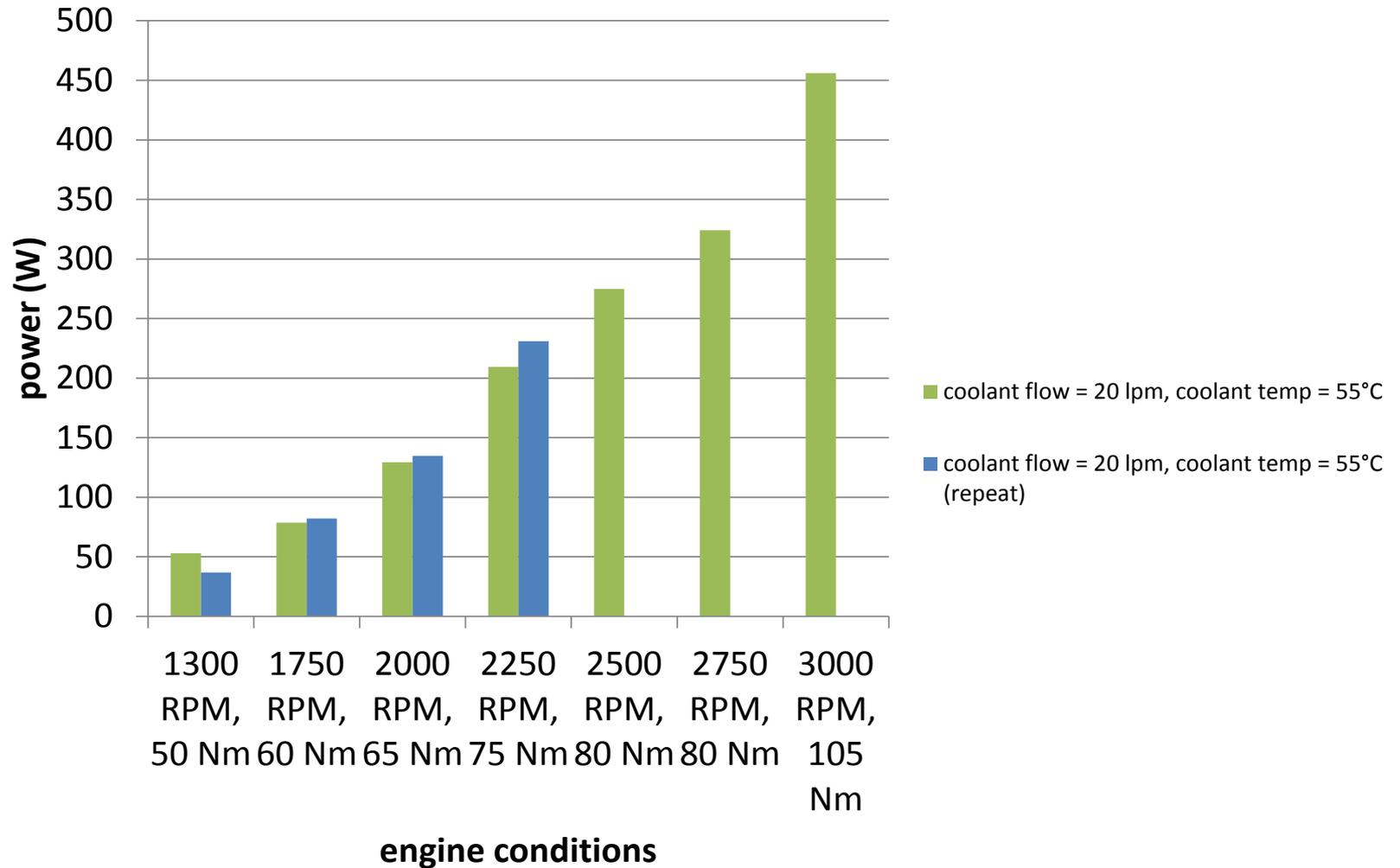
# Steady State Power Vs Coolant Flow/Temp

Measured Steady State TEG Performance on Dynamometer  
01/31/12



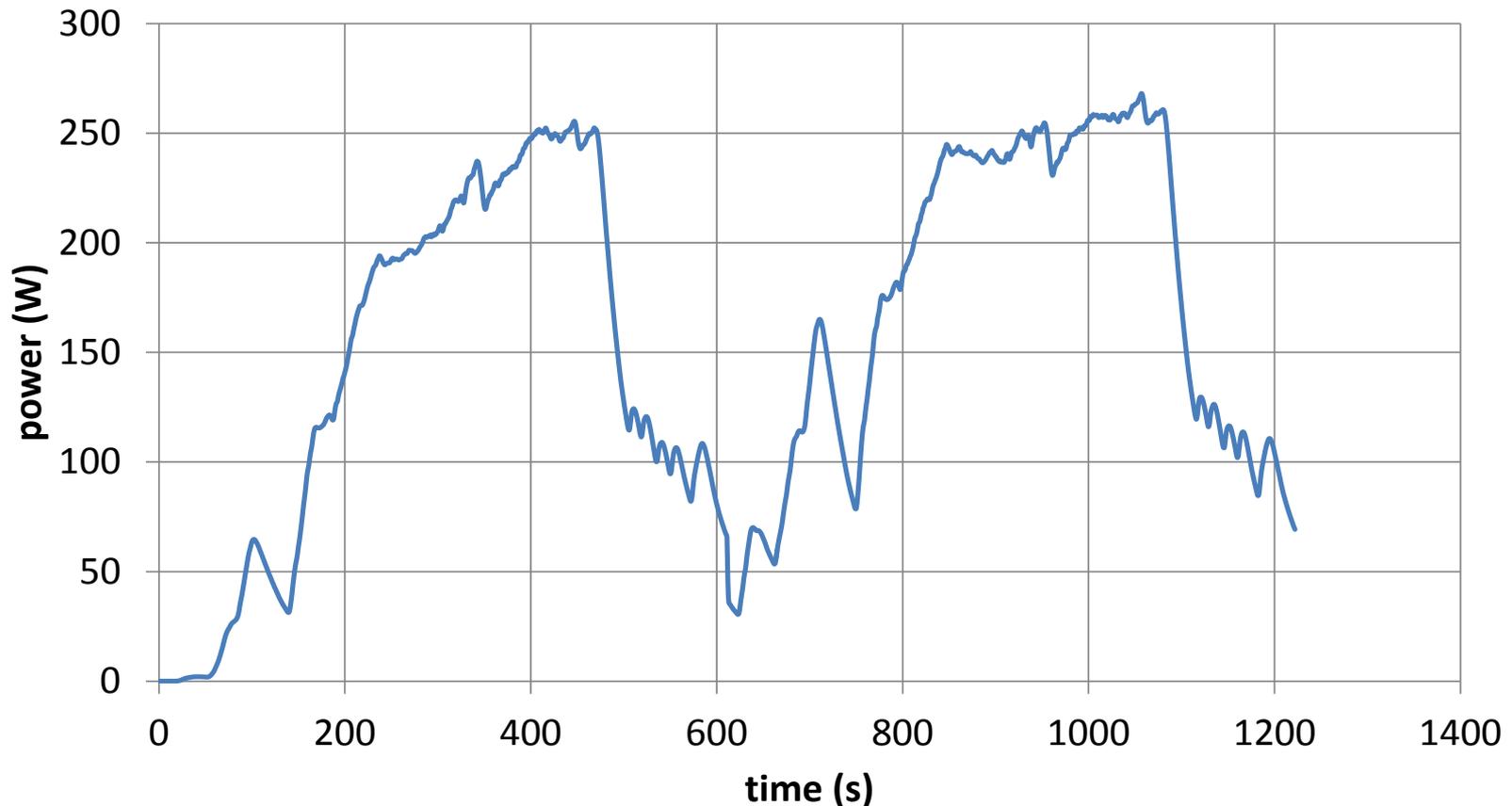
# Repeatability: Steady State Power

Measured Steady State TEG Performance on Dynamometer  
01/31/12



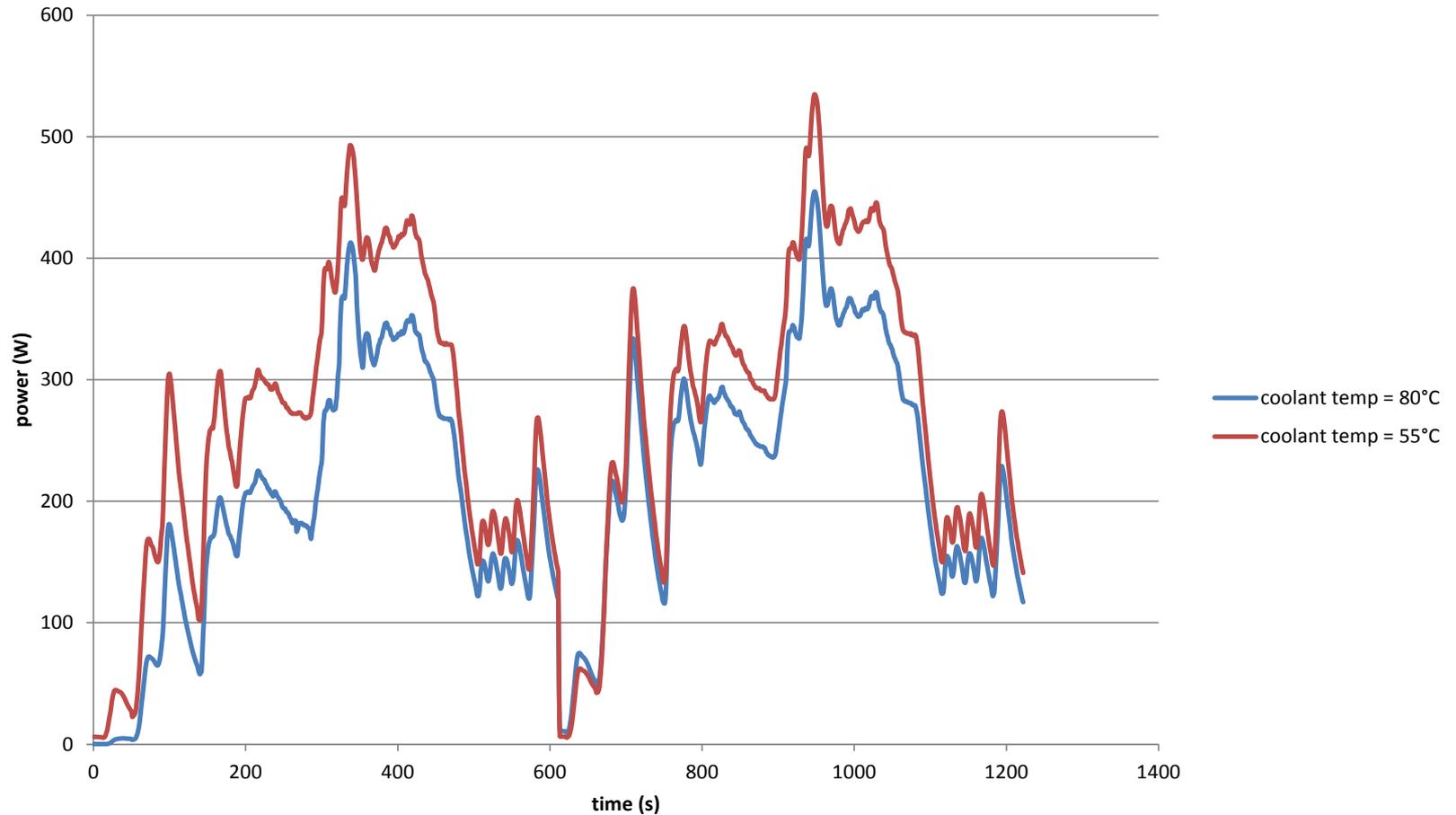
# Engine Dynamometer US06 Drive Cycle Results

**US06 Driv Cycle Measured Dyno Results**  
**coolant temp = 80C and coolant flow = 20 lpm**  
**2 cycles run back to back with 60s idle in between**  
**basic bypass valve strategy employed**



# Engine Dynamometer US06 Drive Cycle Results

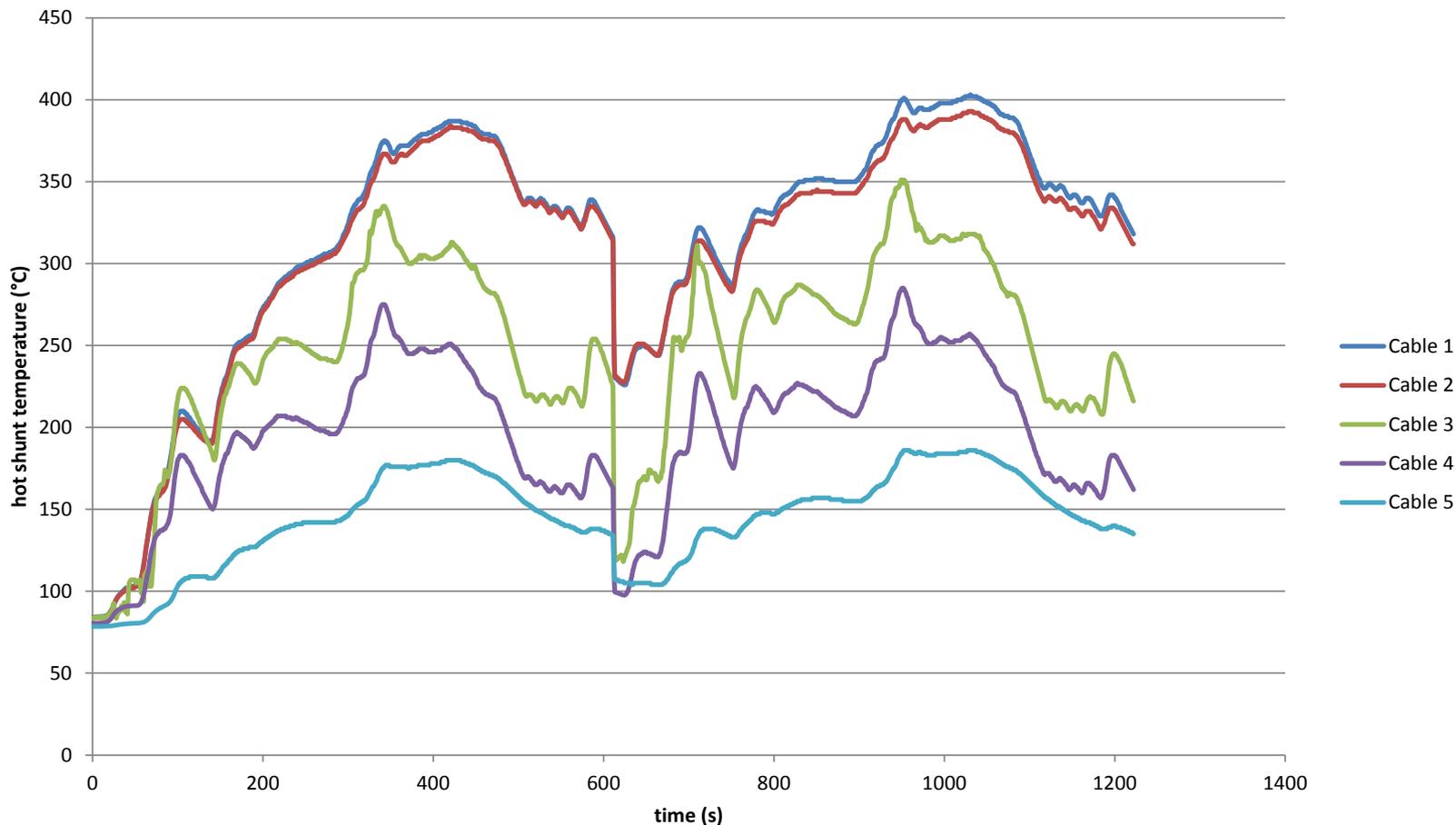
US06 Drive Cycle Measured Dyno Results  
coolant flow = 20 lpm  
2 cycles run back to back with 60s idle in between  
01/31/12



Note: Bypass valve closed for cycle

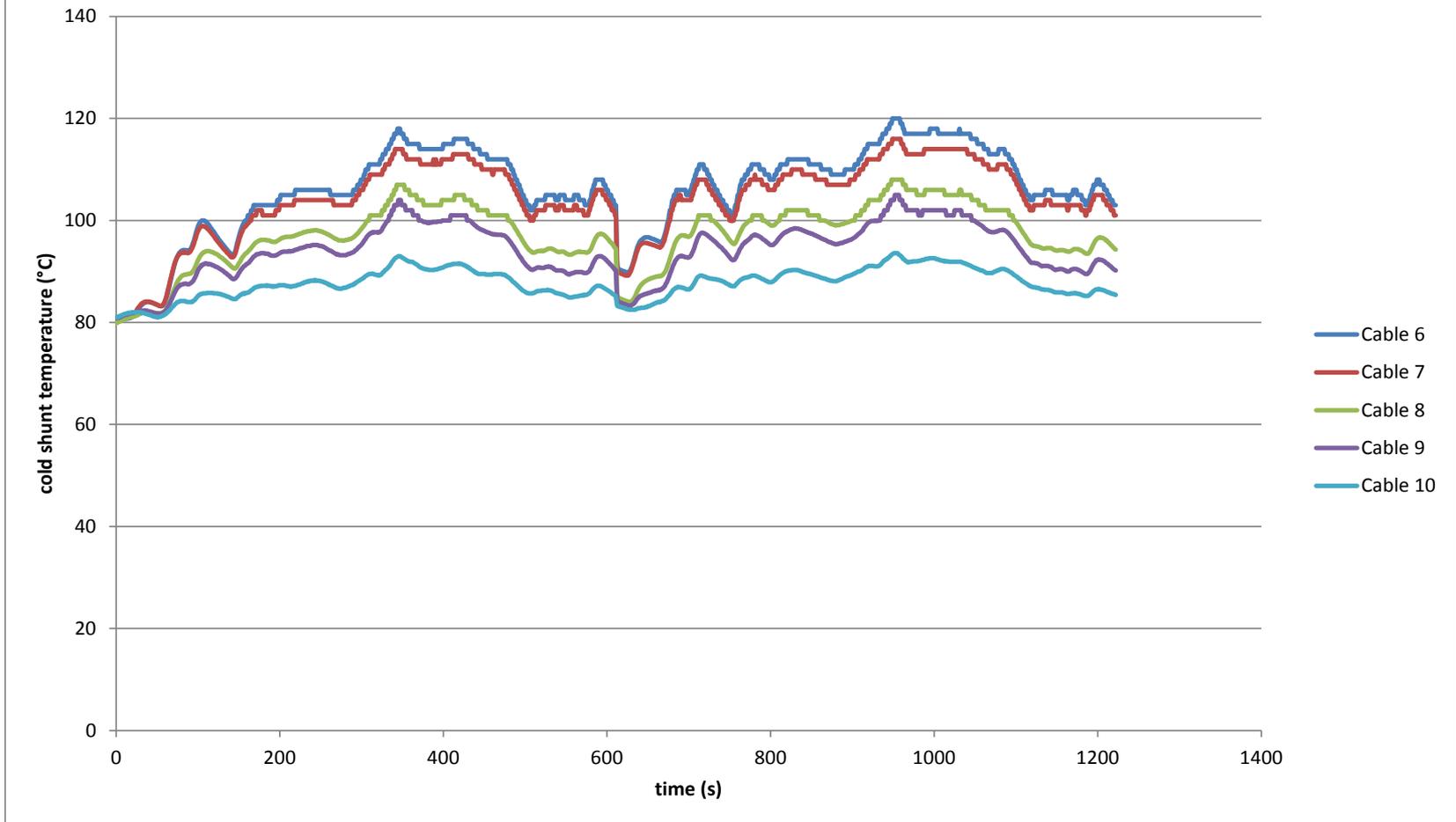
# TE Mat'l Surface Temperatures (Hot) Vs US06

**US06 Drive Cycle Measured Dyno Results**  
**coolant flow = 20 lpm & temp = 80°C**  
**2 cycles run back to back with 60s idle in between**  
**01/31/12**



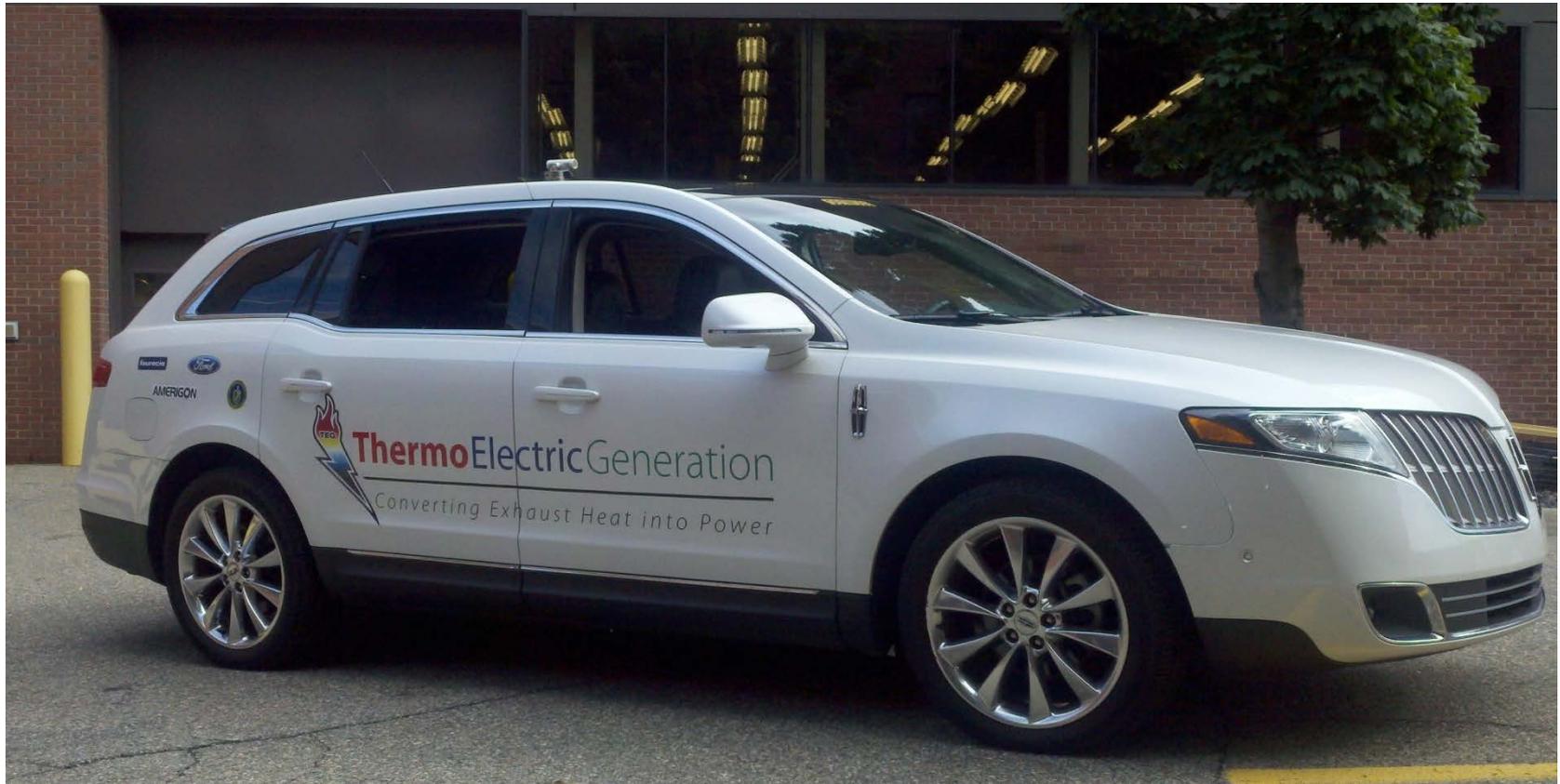
# TE Mat'l Surface Temperatures (Cold) Vs US06

US06 Drive Cycle Measured Dyno Results  
coolant flow = 20 lpm & temp = 80°C  
2 cycles run back to back with 60s idle in between  
01/31/12

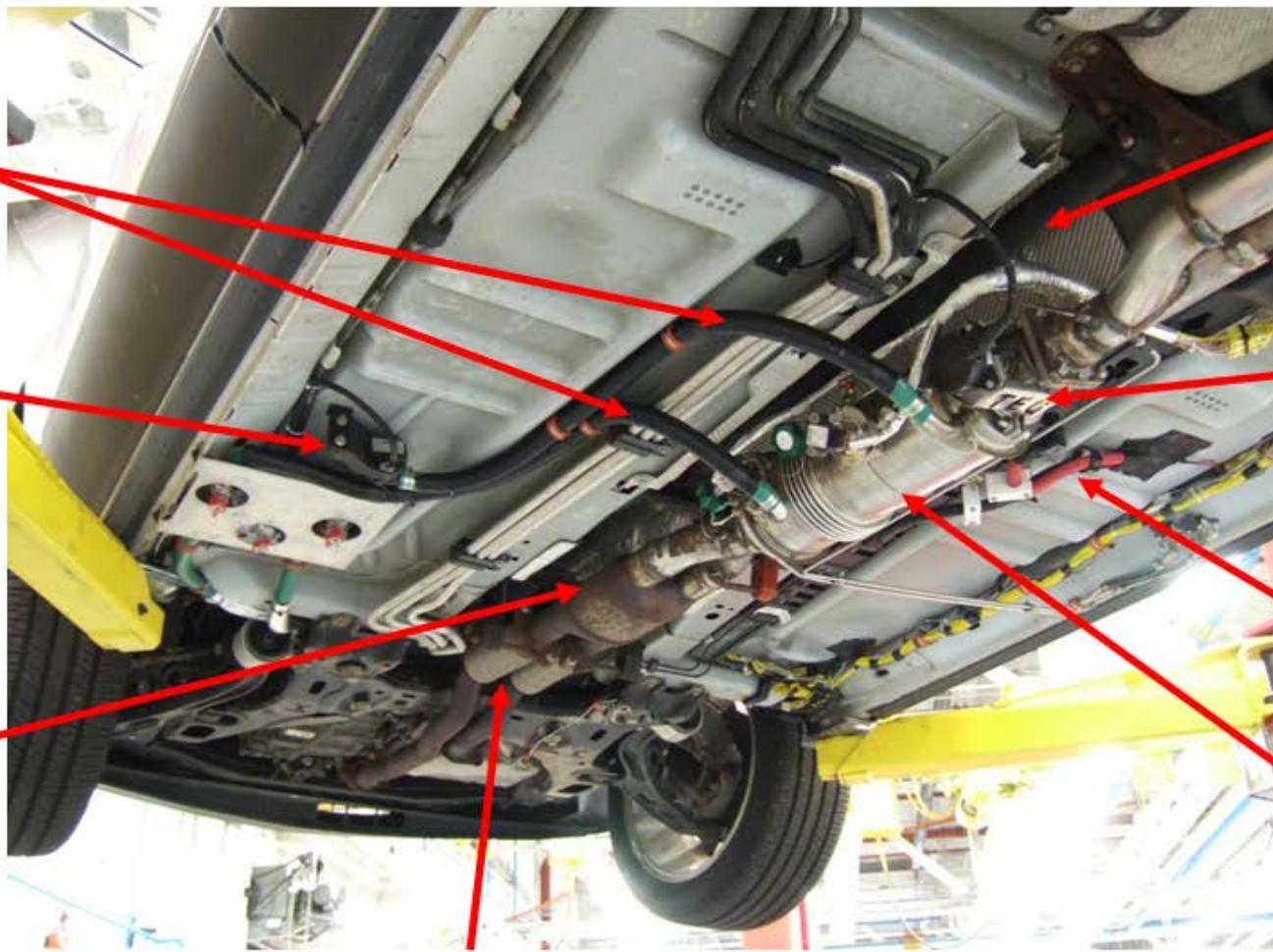


# ***Ford Lincoln MKT AWD***

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# Vehicle Integration



Coolant Lines

Electric Pump

Driveshaft

Bypass Valve

Electrical Connections

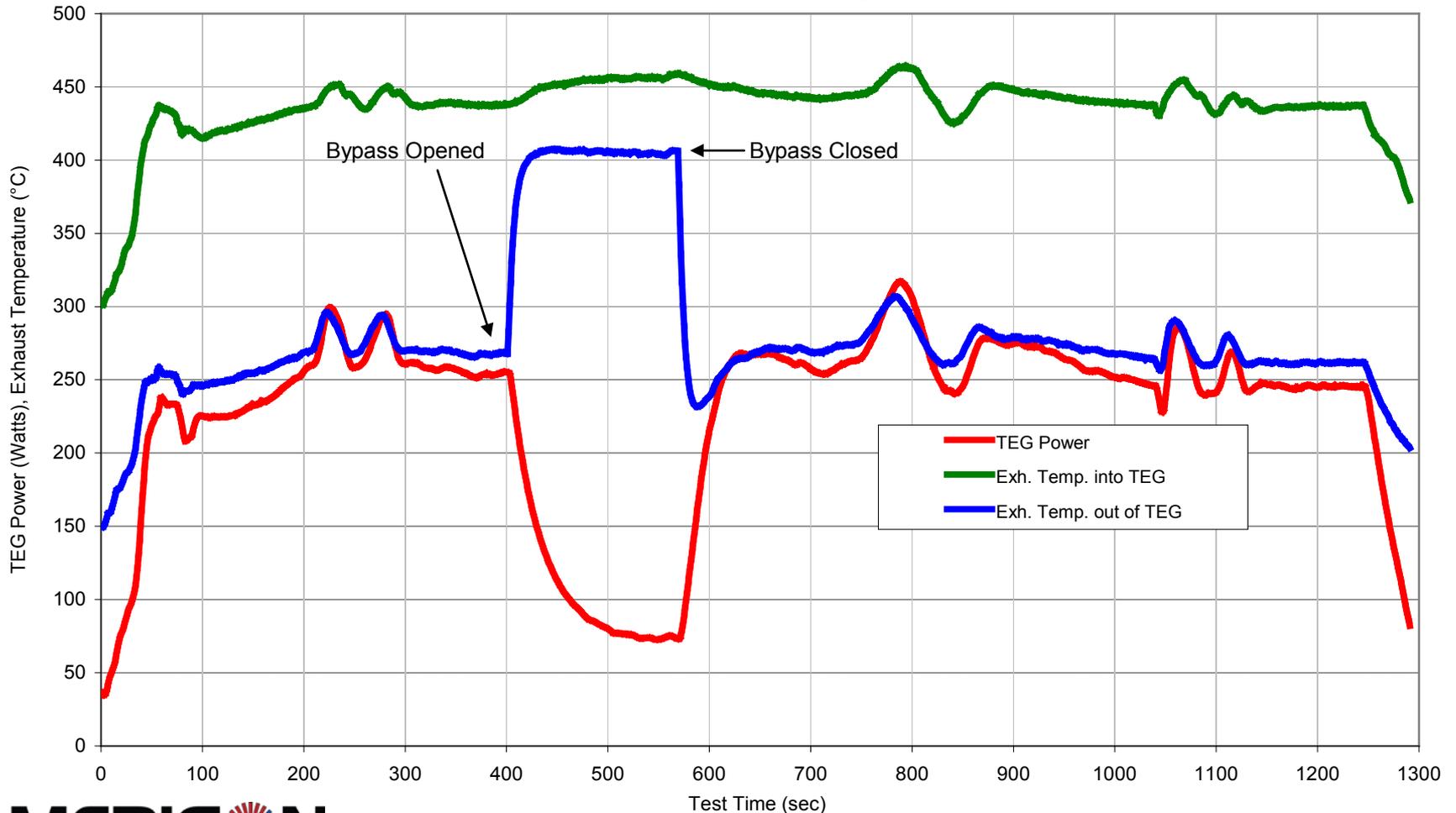
TEG

Underfloor Catalyst

Flex Couplings

# Vehicle On-Road Test Results

## Vehicle Speed 65mph



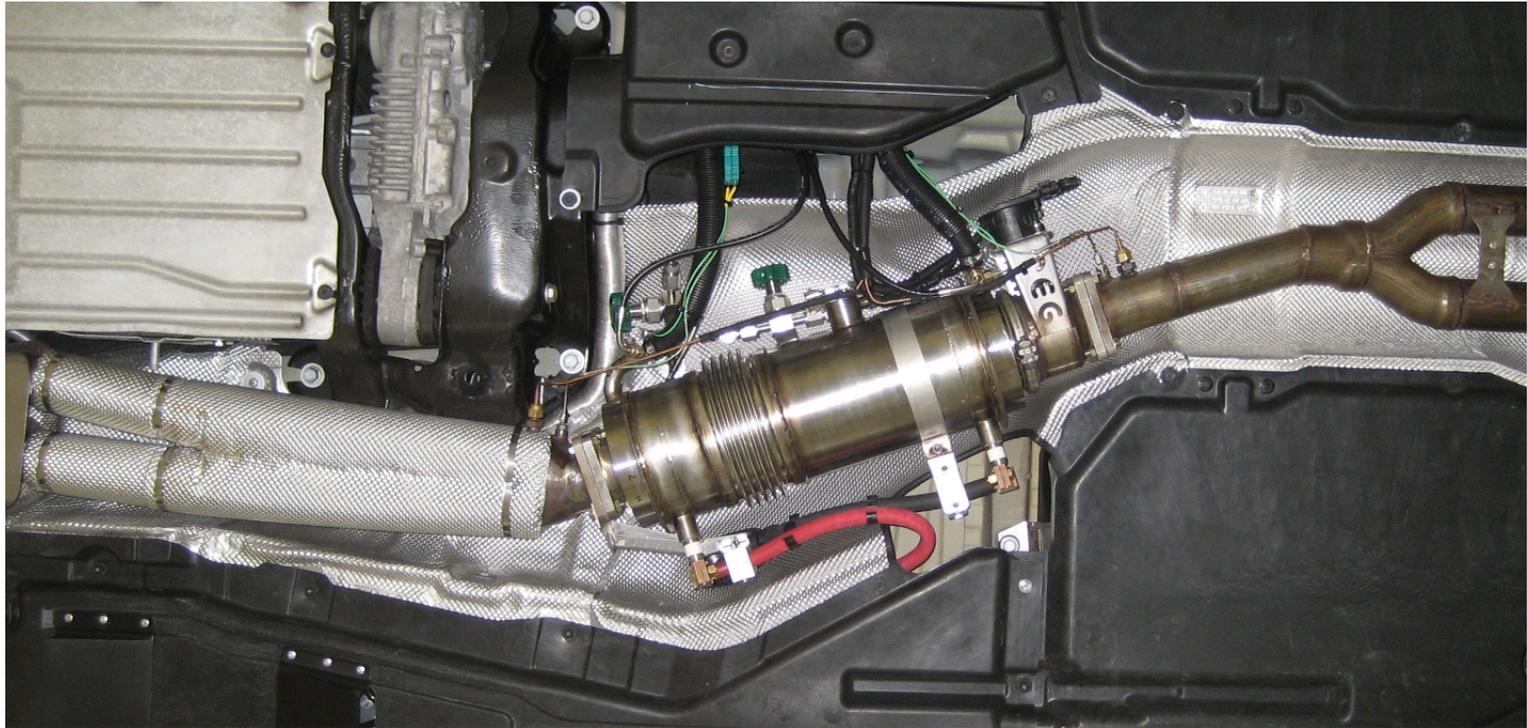
# ***BMW X6***

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# ***BMW X6 Installation***

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# Summary

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TEGs have been built and bench tested with over 700W of power output achieved with air inlet temperatures up to 620C.

TEGs were installed in a BMW X6 and Lincoln MKT with at least 450W of power output achieved in road tests for both vehicles.

Repeatable performance has been achieved in bench and engine dynamometer testing. Repeatable performance still being achieved in over 6 months of vehicle testing.

Steady state model has been validated to within 10% error for a wide range of operating conditions and load resistances for power, voltage, temperatures, and pressure drops for TEGs with varying TE material

Transient model has been validated to within 10% error for a wide range of operating conditions and load resistances for TEGs with varying TE material

Validated transient model has been used to simulate automotive drive cycles such as the US06

# ***Outlook and Further Work***

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Further work is required to address technical and economic risks for TEG commercialization:

- Material and system costs
- Design robustness and performance
- FE Benefits Vs regulatory and customer drive-cycles

The partnership between BMW, Ford, and Amerigon will continue in a follow-on DOE TEG program with the following key objectives:

- 5% FE gain for a passenger vehicle measured over the US06 drive cycle
- Economic feasibility defined for 100K/annum manufacturing volume

# ***Acknowledgements***

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**US Department of Energy: John Fairbanks**

**DOE NETL: Carl Maronde**

**BMW: Boris Mazar, Andreas Eder and Carsten Spengler**

**Ford Motor Company: Clay Maranville, Dan Demitroff, and Quazi Hussain**

**Faurecia Exhaust Systems: Rita Fehle, Robin Willats, Boris Kienle, and Ed Kinnaird**

**Amerigon/BSST/ZT Plus: Steve Davis, Dmitri Kossakovski, Eric Poliquin, Vladimir Jovovic, Joe Dean & Lon Bell & the rest of the Amerigon Team**