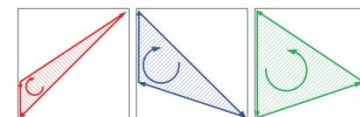
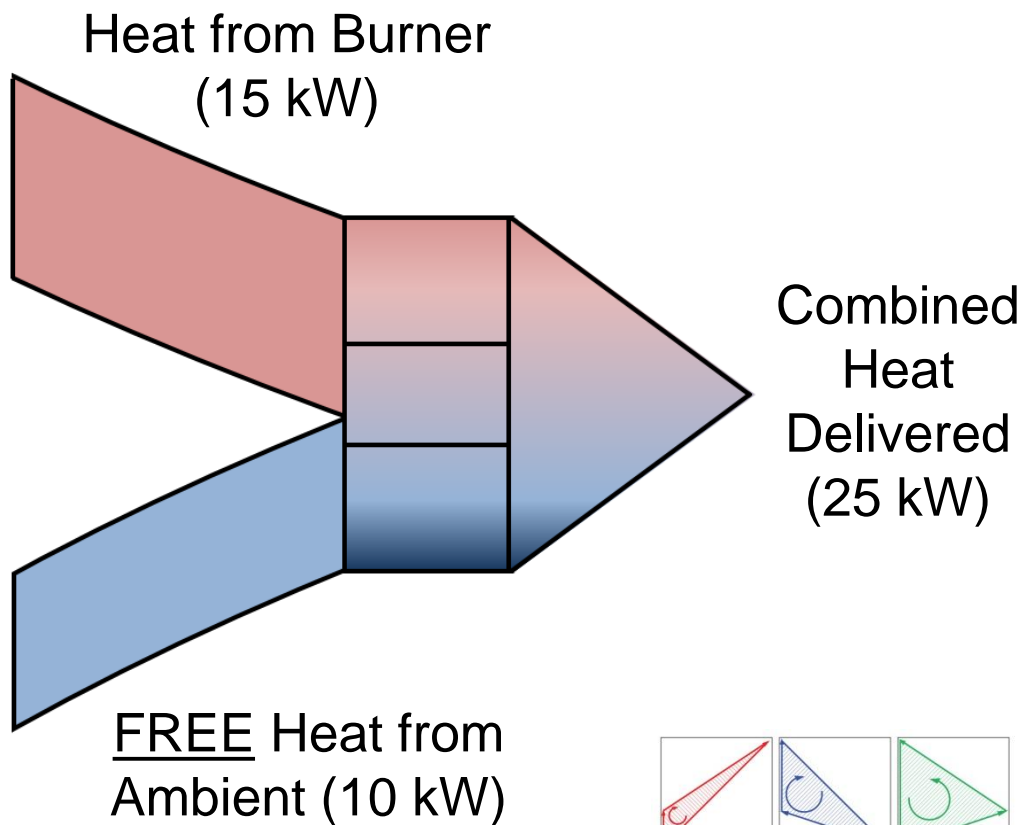
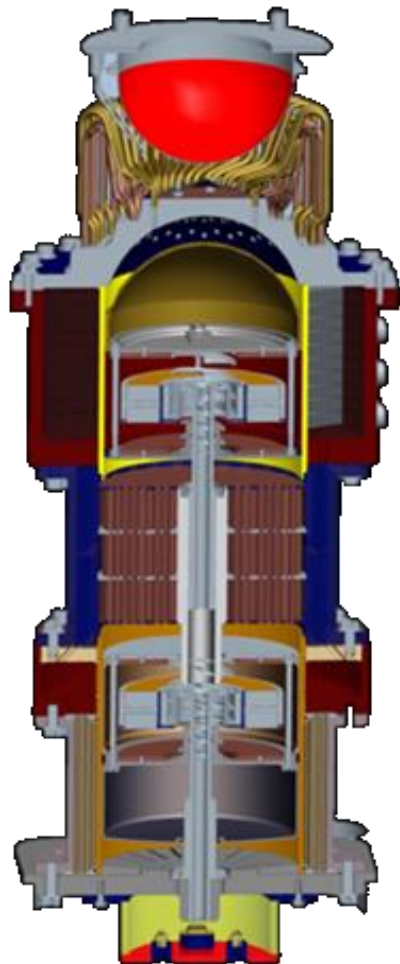


# The Natural Gas Heat Pump and Air Conditioner

2015 Building Technologies Office Peer Review



U.S. DEPARTMENT OF  
**ENERGY**

Energy Efficiency &  
Renewable Energy

ThermoLift, Inc.  
**Paul Schwartz, CEO**  
[pschwartz@tm-lift.com](mailto:pschwartz@tm-lift.com)

DE-FOA-0000823

# Project Summary

## Timeline:

Start date: 10/1/2013 (8/1/2013)

Planned end date: 3/31/2016

## Key Milestones

1. Concept & CAD model; Q1 FY2014
2. Thermal Simulation; Q2 FY2014
3. 20kW Demonstrator; Q3 FY2015
4. Testing at Oak Ridge; Q1 FY2016

## Budget: \$750,000

Total spent to date: \$525,331

Total future: \$224,669

## Target Market / Audience:

Residential and small commercial buildings and specialized industrial applications.

## Key Partners:

DOE	NYSERDA
Stony Brook Univ.	Oak Ridge Natl. Lab.
National Grid	Star Gas
ATA	Par Group
Fala Technologies	LoDolce

## Project Goal:

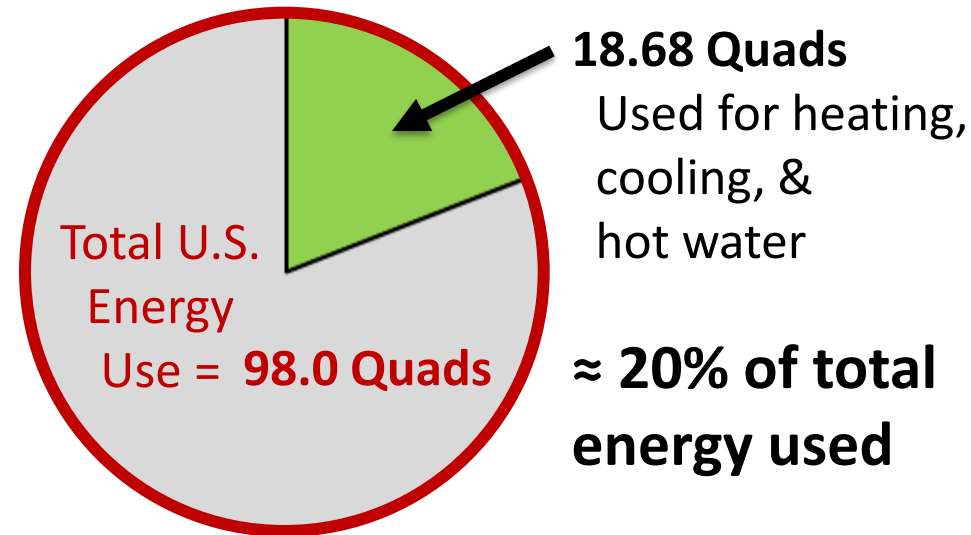
To develop a Vuilleumier heat pump (VHP) which includes novel improvements that will yield higher performance than the already high COP results of previously developed VHP. The heat pump will use natural gas to provide heating, cooling, and hot water with a single device.

# Purpose and Objectives

## Problem Statement:

- In cold weather climates, typical heat pumps are prohibitively costly to operate.
- In warm climates, peak electricity is very expensive.
- Common HVAC devices (compressors, furnaces, boilers, etc.) are outdated / inefficient.

## Target Market and Audience:



## Impact of Project:

1. The project goal is a 20kW device, capable of delivering complete heating and cooling demand (2,000 sq. ft. building).
2. Growth and Impact Plan:
  - a. Near-term (<1 yr after project) – Incorporate manufacturing design refinements and conduct durability studies. Begin demonstration projects and pilots through partners including national laboratories, gas utilities, DoD, and others.
  - b. Intermediate-term (1-3 yrs) – Product launch, 5kW units installed during the first year, 15,000 in year two.
  - c. Long-term (3+ yrs) – Expansion / global adoption, Target: 150k-250k unit production.

# Approach

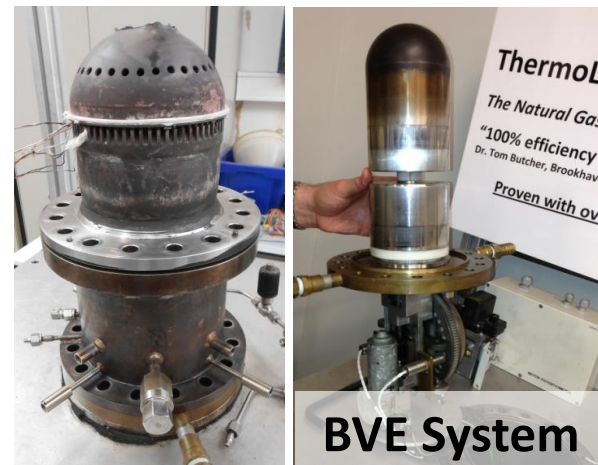
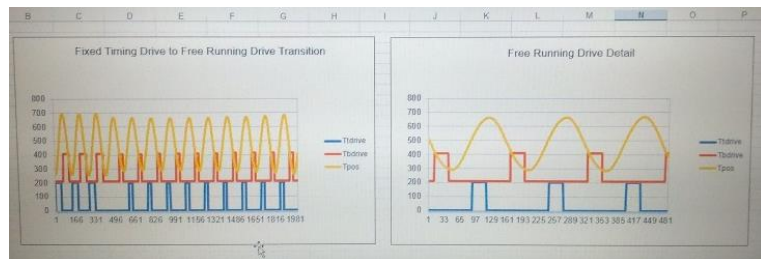
**Approach:** ThermoLift is modernizing a proven Vuilleumier cycle device for heating, cooling, and hot water. The end product will be a single 20kW natural gas-driven device for residential and commercial applications. Engineering development is focused on incorporating innovative improvements, optimizing device performance, and reducing the complexity and manufacturing costs.

## Key Issues:

- Design of novel electromechanical drive.
- Design of optimized heat exchangers.
- Concerns due to high temperatures & pressures.

## Distinctive Characteristics:

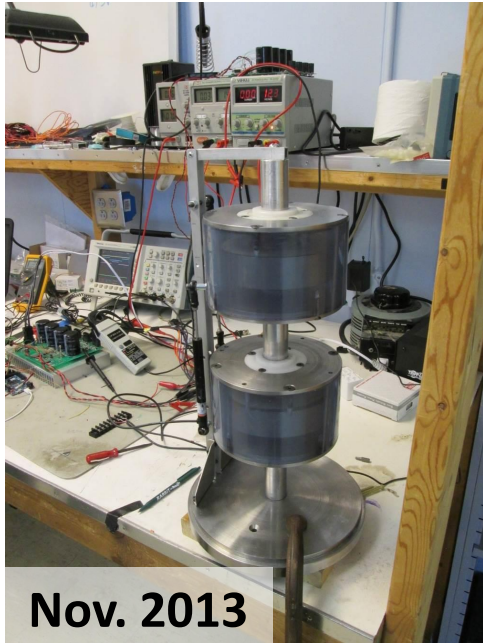
- Single natural gas-driven device
- Minimal electricity → grid independence
- No refrigerants – No compressors
- High cold climate performance



Temp °F	DOE Target	BVE Demonstrator
-13	1	1.3
17	1.15	1.45
47	1.3	1.65



# ThermoLift First Prototype



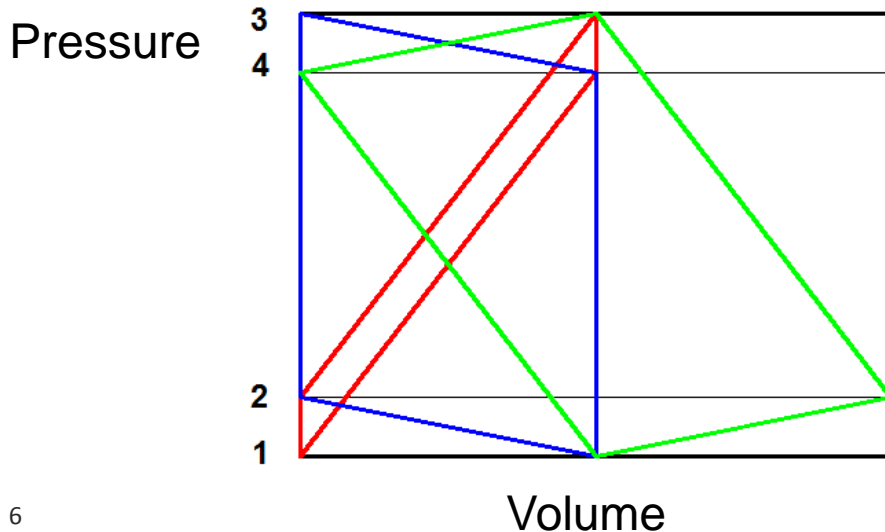
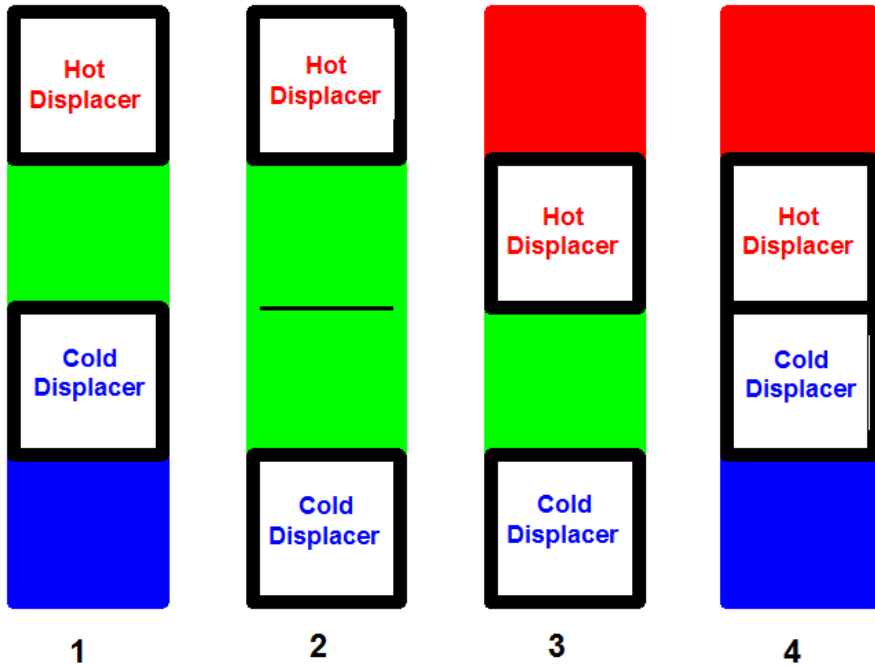
ThermoLift first developed a benchtop proof-of-concept for the novel independent mechatronic design. A complete heat pump prototype was designed, simulated, and built by late 2014.



ThermoLift began testing and generated numerous iterations and simulations develop improved components.



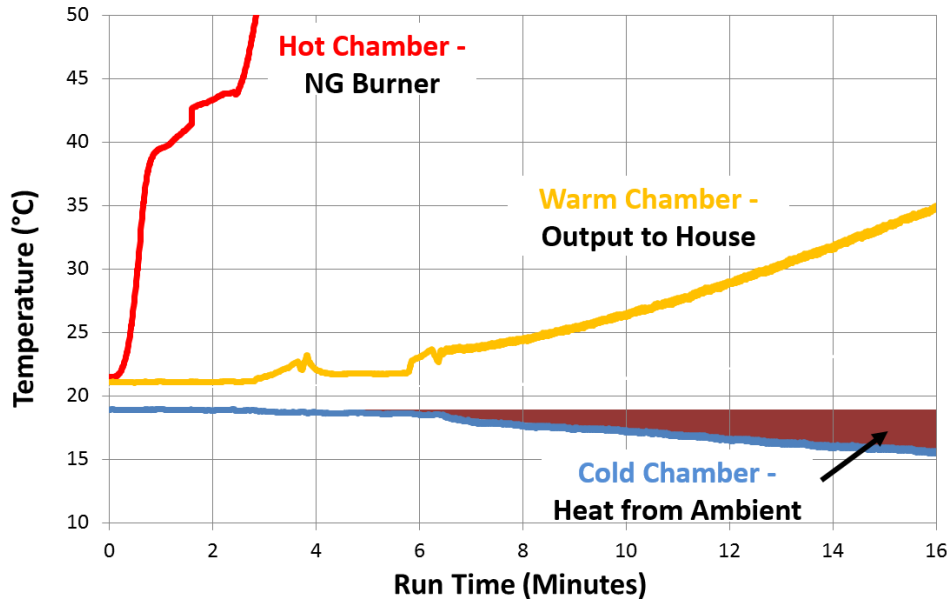
# ThermoLift 4 Phase Process (Patent Pending)



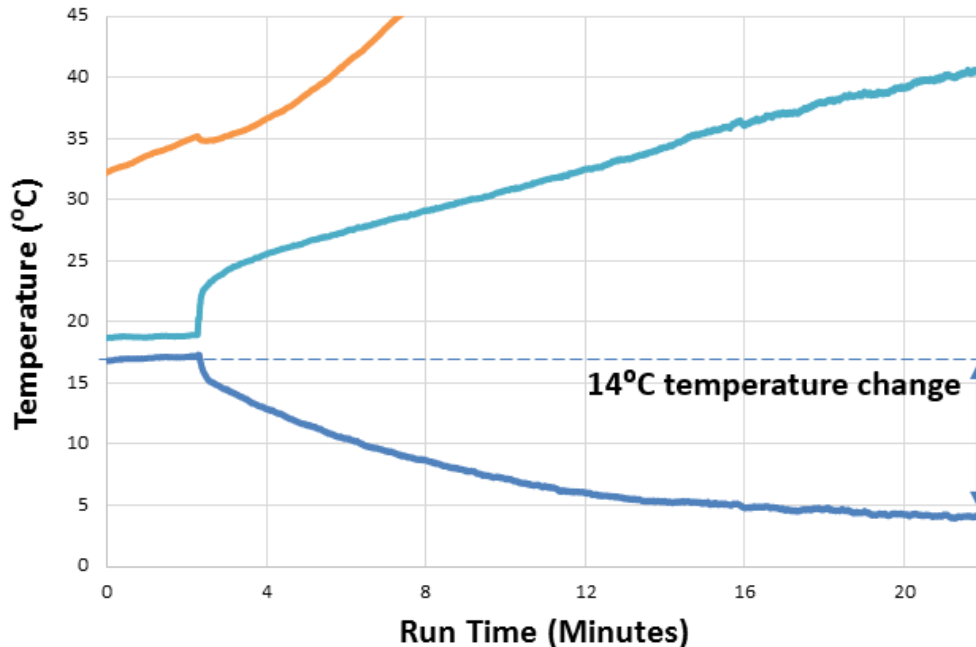
## Novel displacer system incorporating electronic controls:

- Enables independent and discontinuous motion.
- Thermodynamic advantages based on ability to control gas flow differently.
- This advancement was unachievable with mechanically-linked displacers in previous Vuilleumier devices.
- Enables customization and optimization at partial load conditions and for cooling applications.

# Iterative Testing (Oct. 2014 - Mar. 2015)

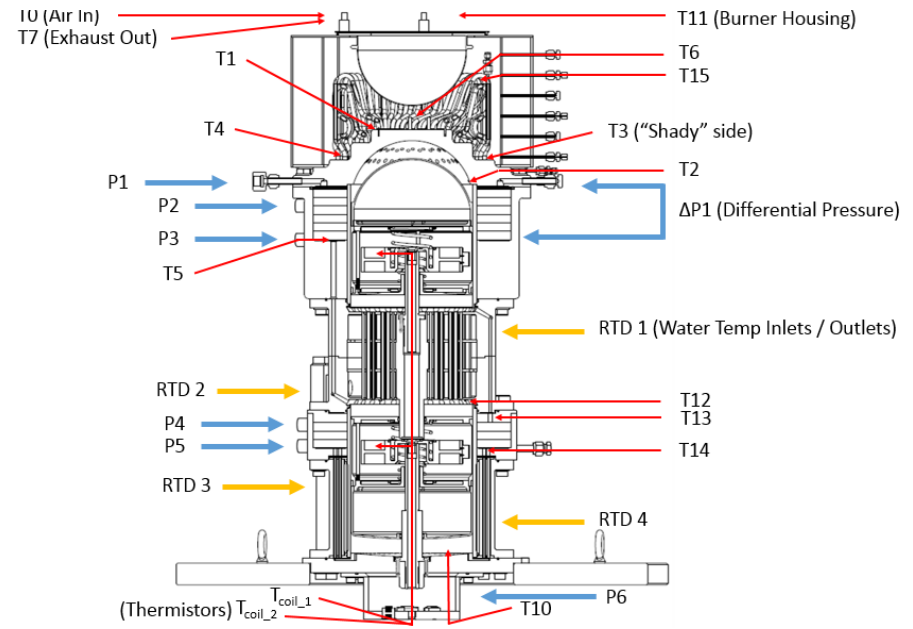
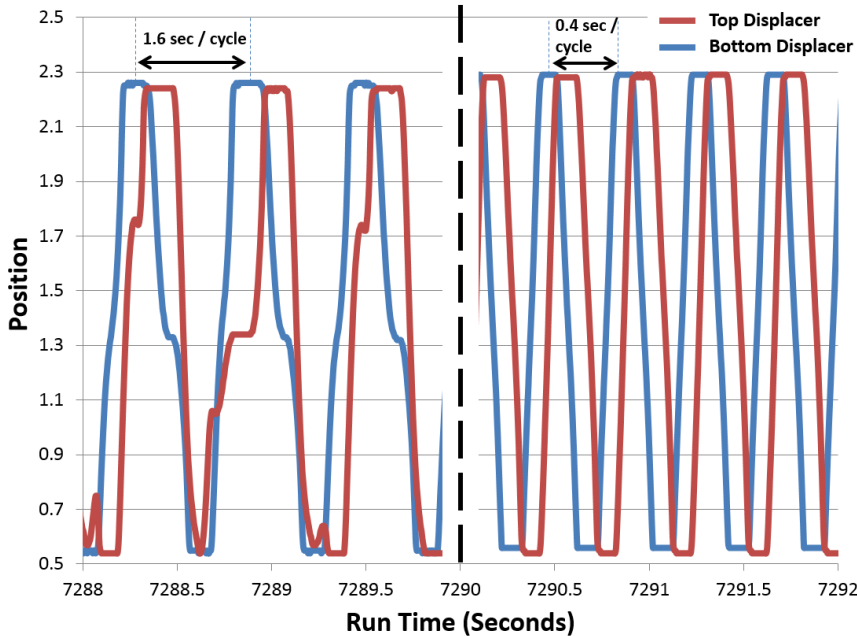
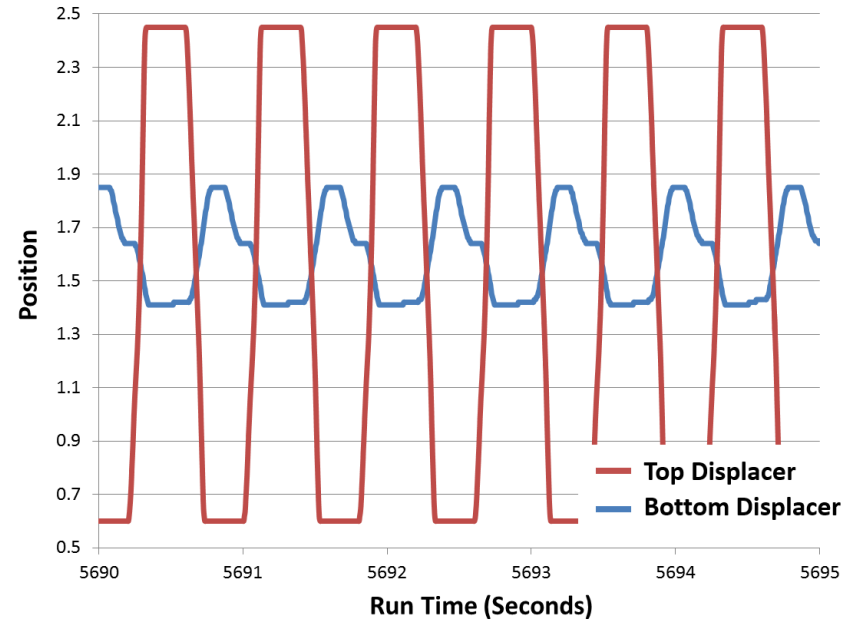
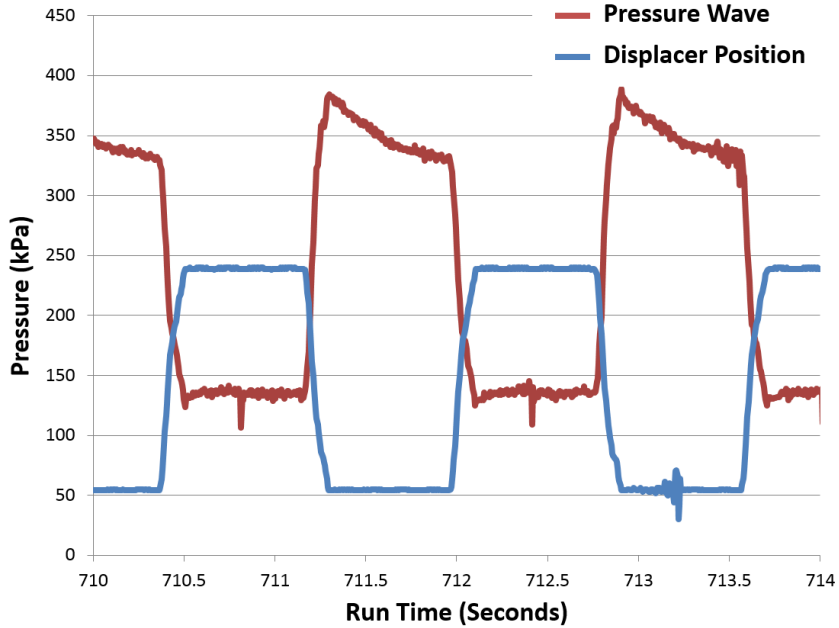


Prototype Iteration	Run Time	Cold Temp. Drop
Oct. 2014	4-6 min	4° C
Jan. 2015	6-9 min	6° C
Feb. 2015	20 min	9° C
Mar. 2015	>25 min	>14° C



Initial test results have unequivocally illustrated that the thermodynamics (Vuilleumier cycle) that enable ThermoLift's technology are working. Over the past two months of testing, ThermoLift was able to increase the operating speed of the displacers, improve overall heat lift, and overcome mechanical binding problems enabling extended run time and experimentation.

# Thermodynamic Data & Instrumentation





# Data & Simulation Correlation

1D Models	A	B
Hot heat input	2.21 kW	2.15 kW
Cold heat lift	0.78 kW	0.84 kW
Warm heat output	3.07 kW	2.98 kW
Electrical input	34 W	23 W
COP	1.37	1.39

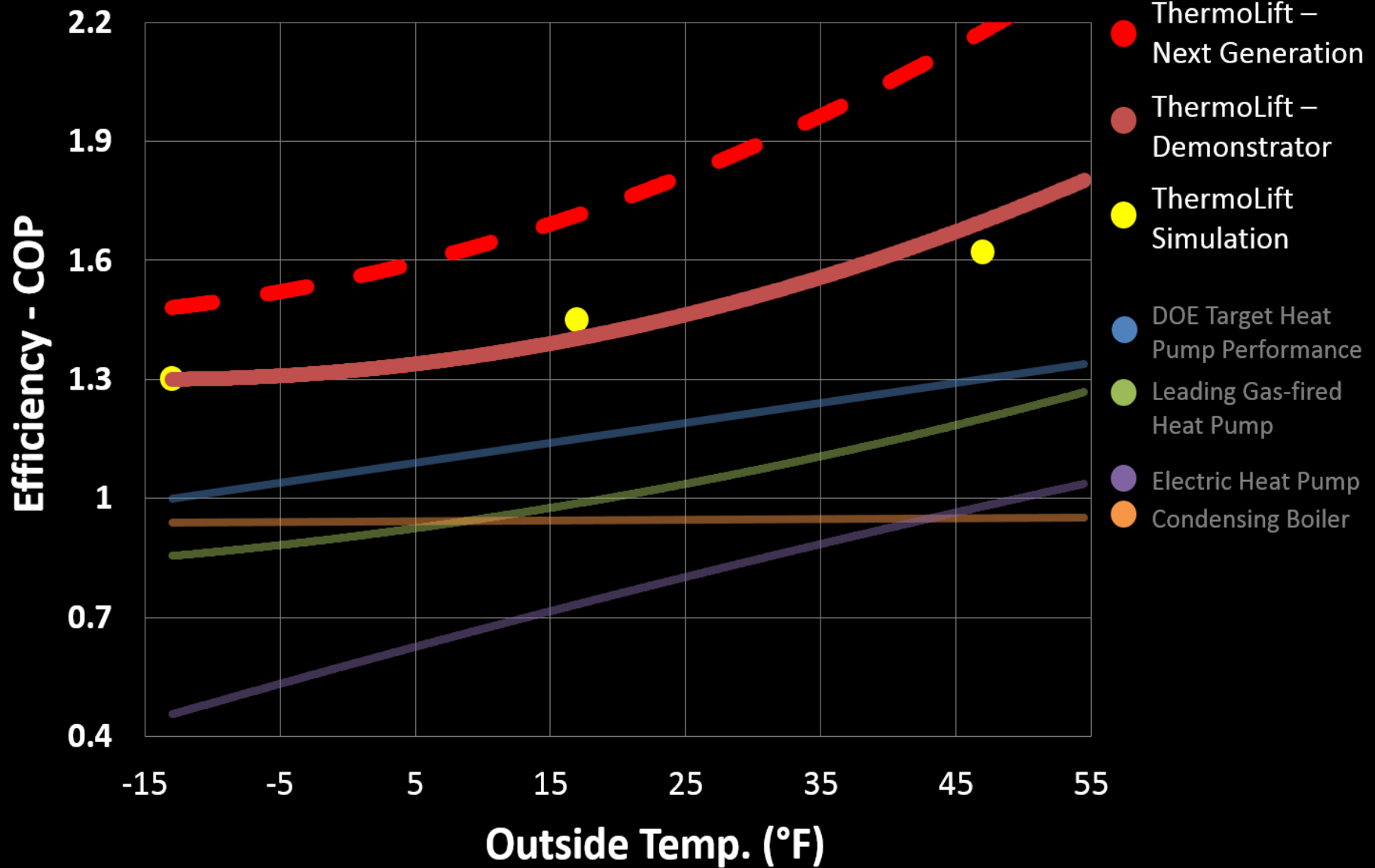
Multiple simulation tools show strong correlation with experimental data (left). Current testing conditions and results represent <10% of the anticipated capacity of the device.

*(Reduced heat, pressure & speed)*

Based on simulation results at full operating pressure (right), temperature and speed, the 1<sup>st</sup> generation ThermoLift prototype using natural gas will out-perform current SOA cold climate heat pumps leading to significant energy and emissions savings without refrigerants.

Temp °F	DOE Target	ThermoLift Simulation
-13	1	1.3
17	1.15	1.45
47	1.3	1.62

# Comparison to State-of-the-Art



# Team & Progress (5 PhDs & 9 Masters)

**Steve Winick\***

*TopSpin*

**Prof. Dr. Peter Hofbauer**

*Co-Founder & Inventor*

**Bob Catell\***

*Chairman, Advanced Energy Center (aertc.org)*

**Paul Schwartz\***

*Founder & CEO*

## Engineering

**Erik Kauppi**

*Program Manager*

## Business Development

**Jonathan Haas**

*Senior Vice President, M.S.  
Grants – DOE, NYSEDA, DOE*

## Intellectual Property

**Diana Brehob**

*Intellectual Property, M.S.*

**Jon Longtin, Ph.D.**

*SBU Mechanical Engr.*

**David Yates**

*Lead Designer*

**Rachel Falor**

*Admin / Accounting*

**Christina Convey**

*Research / Marketing*

**MBA Students**

*SBU Business*



**Seann Convey**

*Materials Sci. M.S.*

**Matt Duthie**

*CAD*

**Dennis Lin**

*Thermodynamic, M.S.*

**Andrew King**

*Mechanical Engineer*

**Wayne Oaks**

*CFD Expert, M.S.*

**Jim Cambell**

*Electronics Engineer*

**Hanfei Chen**

*Mechanical Ph.D. stu.*

**Lisa Thierbach**

*Software Controls*

**Ryan McGann**

*Mechanical Eng. M.S.*

**Tom Litow**

*Electronics, M.S.*

**Applied Thermodynamic Apparatus (ATA)**

## Advisors

**Michael Faltichek**  
*Long Island Angel Network*

**Michael Danaher**  
*WSGR*

**David Hamilton**  
*CEBIP*

**Bill Worek, Ph.D.**  
*Stony Brook University*

## Advocates

**Governor Cuomo**  
*New York State*

**Adm. Dennis McGinn**  
*Navy Energy*

**Mark Austin**  
*NYSEDA /  
Bright Capital*

**Col. Paul Roege**  
*Army Chief Operation Energy*

**Thomas Nowak**  
*European Heat  
Pump Association*

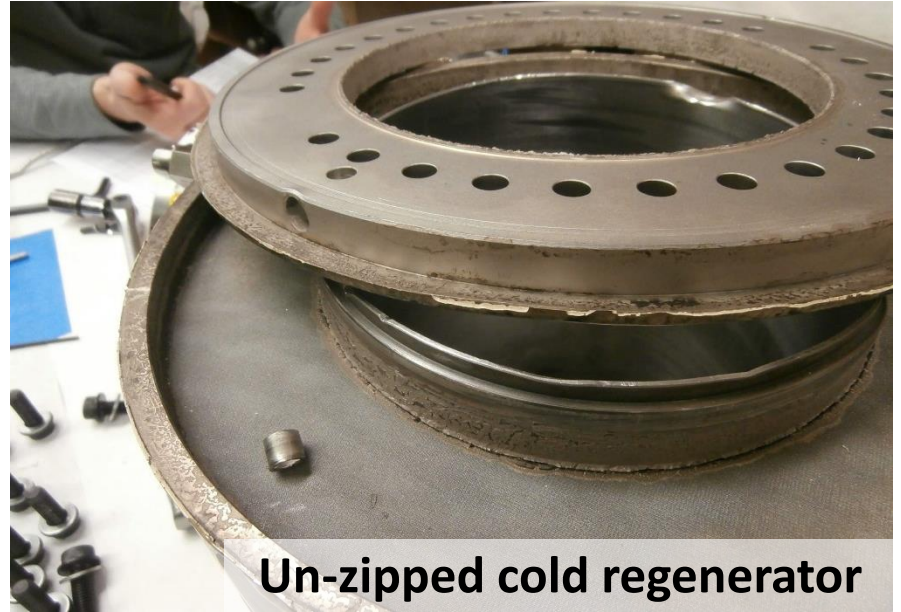
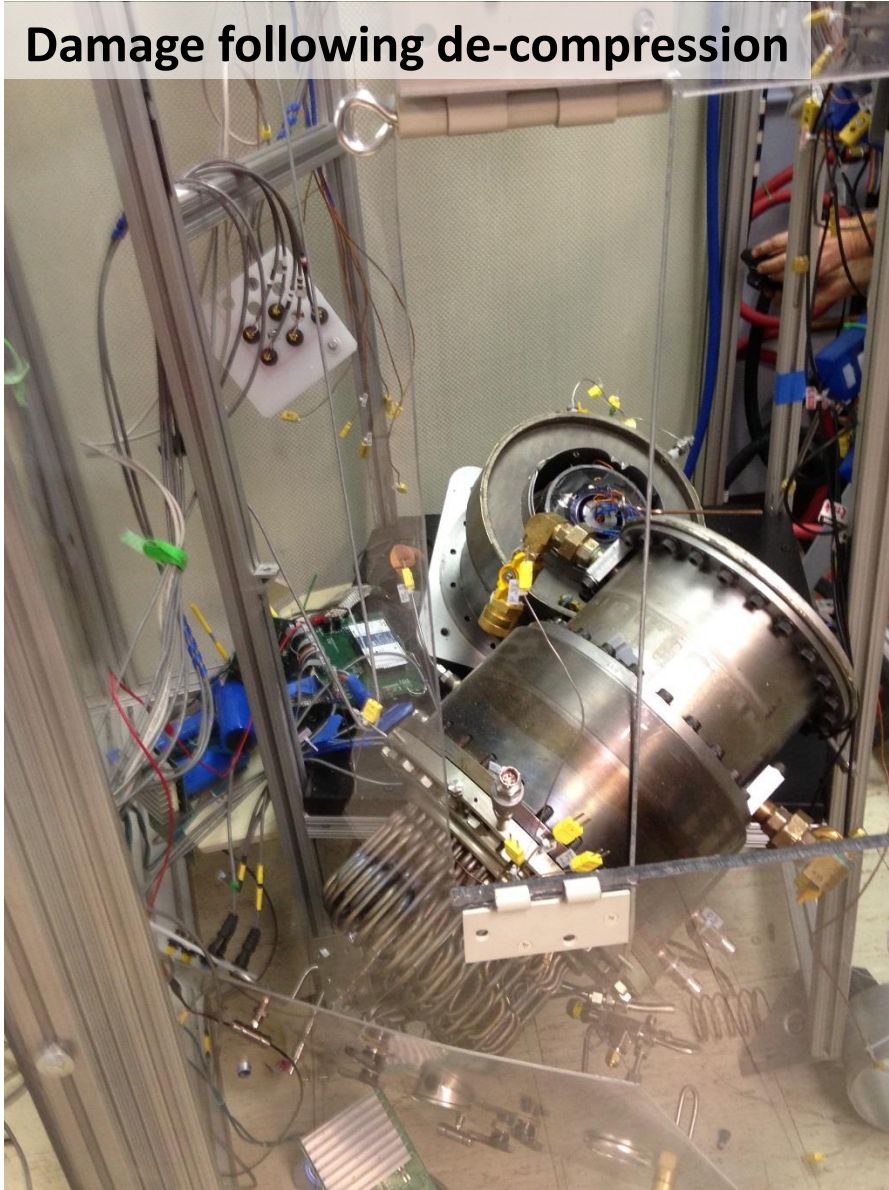
**Mark Stockbauer**  
*Director Energy Naval Air Warfare*

**Adm. Kevin Slates**  
*Navy Chief Operations Energy*

\* Board Member

# Manufacturing Failure → Timeline Setback

Damage following de-compression



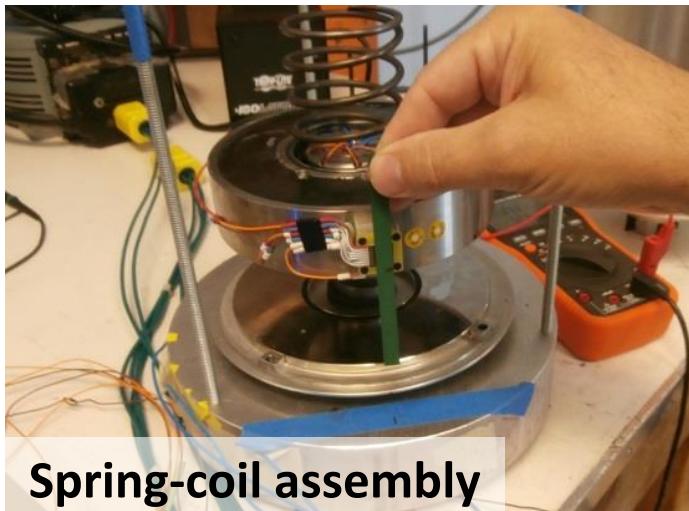
Un-zipped cold regenerator

Complete braze failure



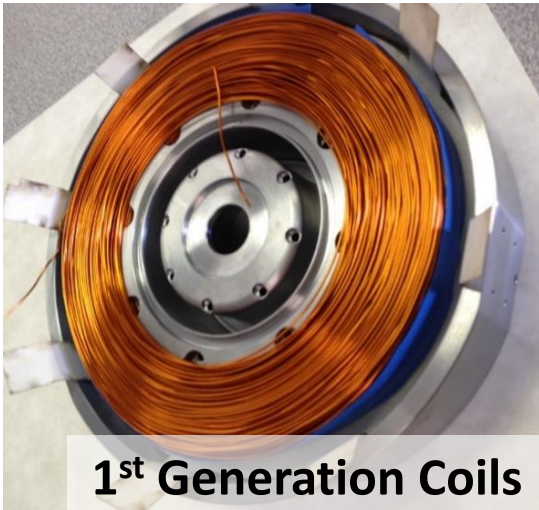


# Prototype Build & Improvements

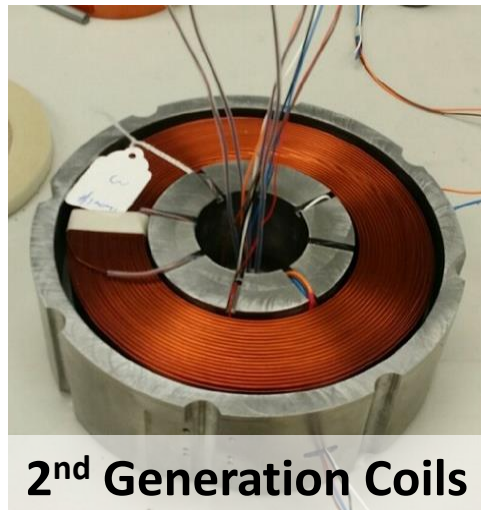


**Spring-coil assembly**

Through testing, enhancements to the spring-mass system and electromagnetic properties of our coils have led to reliable operation. Further refinement of the mechanical concepts and electronic controls will provide improved performance and enable field durability demonstration.



**1<sup>st</sup> Generation Coils**



**2<sup>nd</sup> Generation Coils**



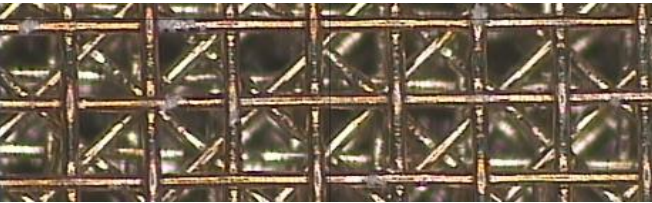
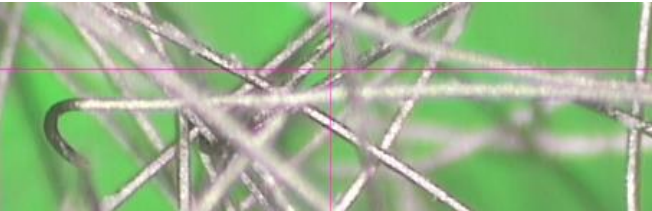
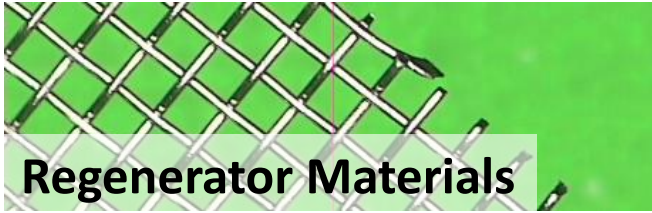
**Improved Guides**



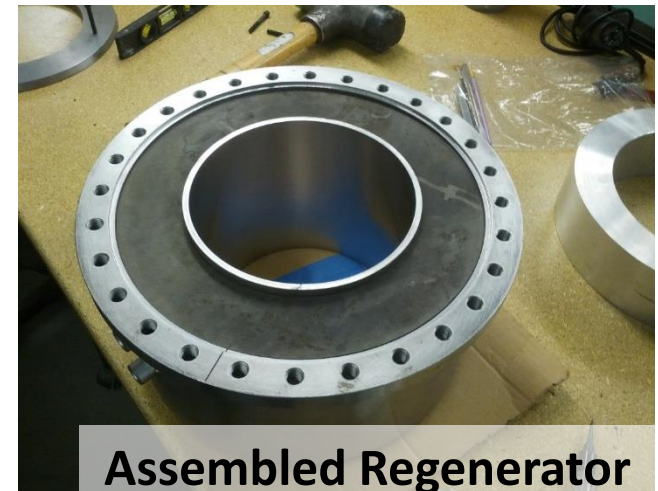
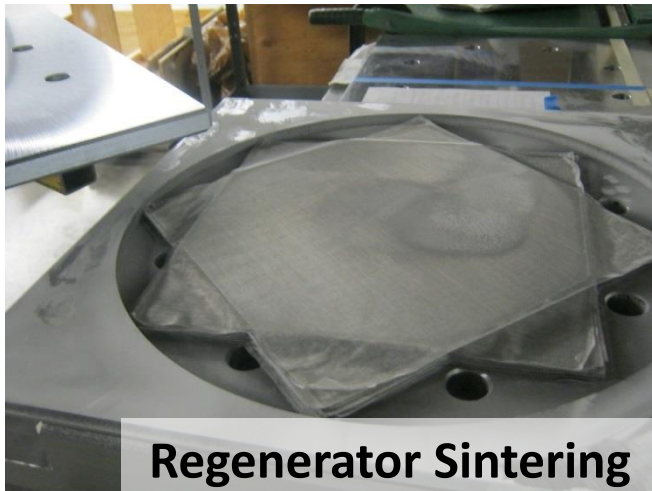
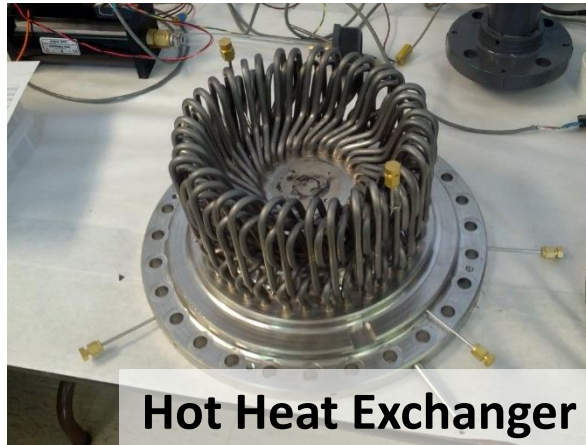
**Displacer Testing**



# Thermodynamic Components (6 Patents Pending)

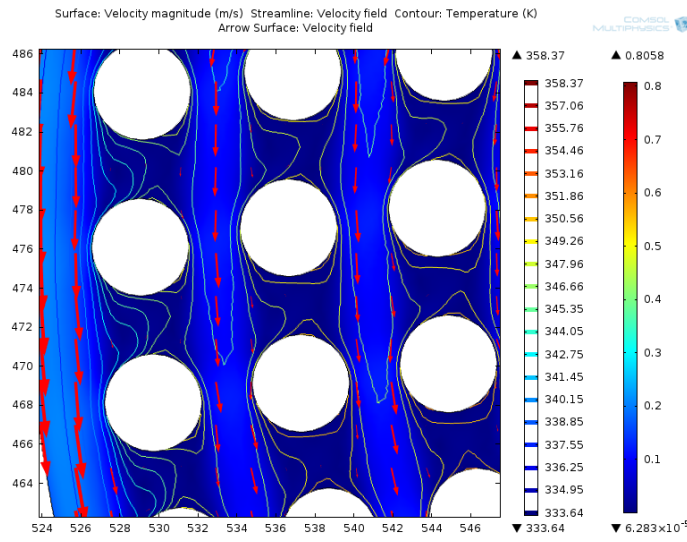
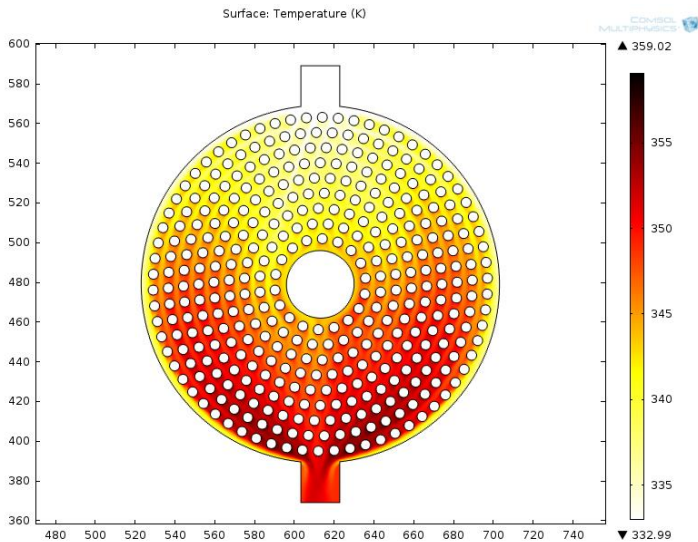


ThermoLift's 1<sup>st</sup> generation prototype integrated a simple off-the-shelf burner and basic heat exchanger and regenerator concepts. Future development will focus on novel optimized thermodynamic components.

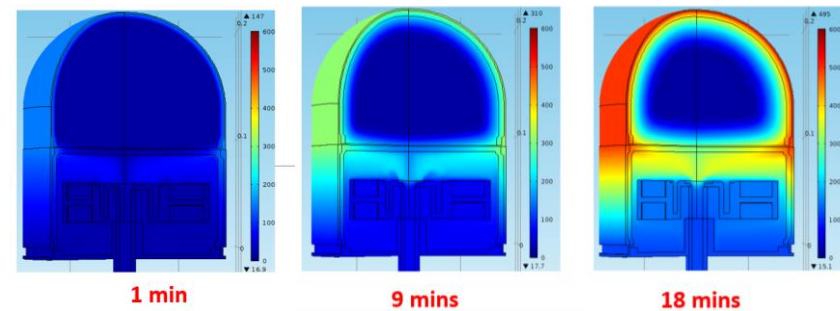
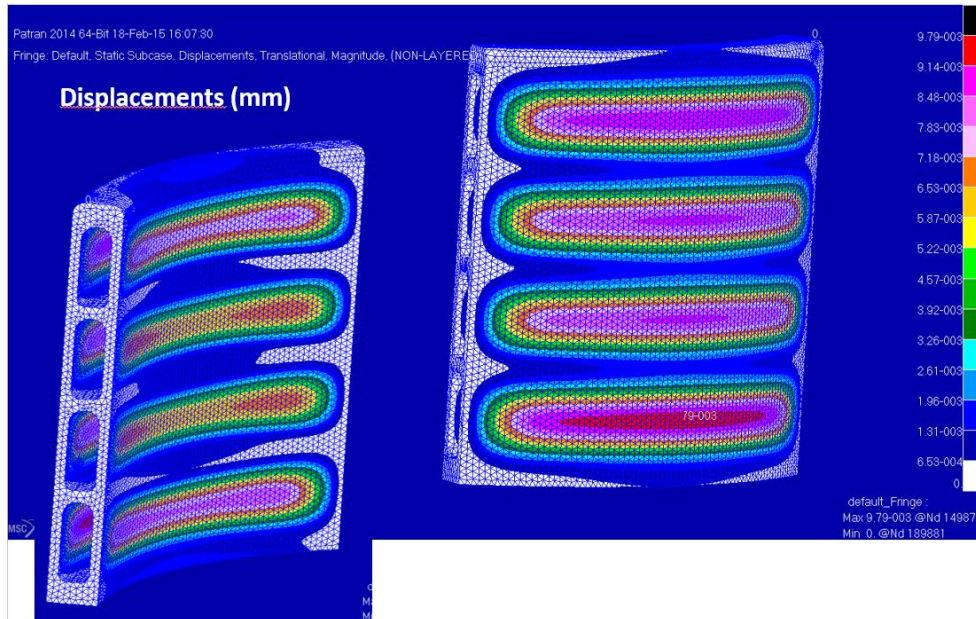




# Simulation Improvements

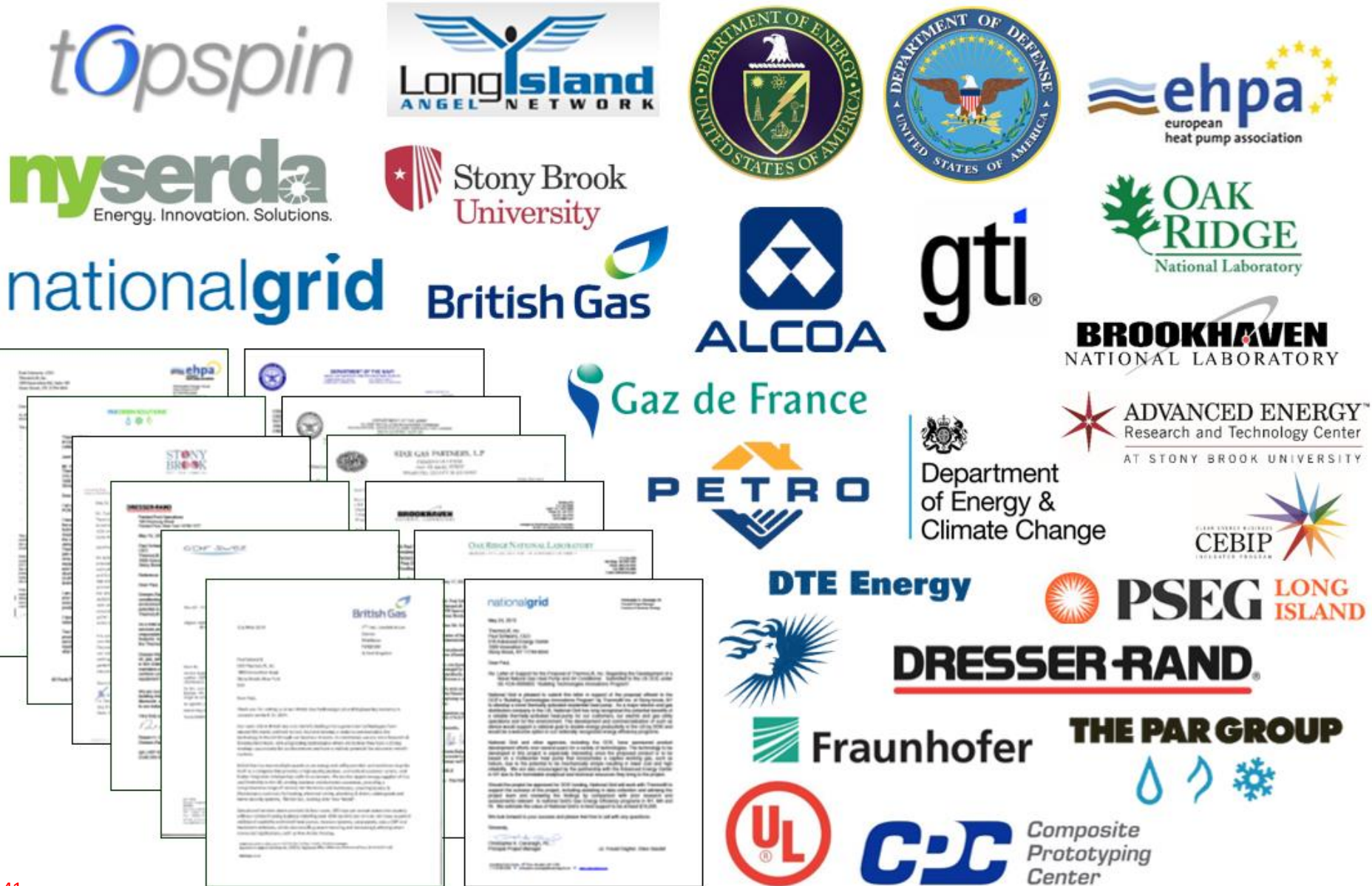


Previously, thermodynamic simulation tools and team expertise were limited to 1D & 2D for simplistic cases.



In the past 12 months, we have been able to dramatically improve our modeling robustness and agility with more sophisticated 3D simulation.

# Letters of Support / Interest in Commercialization





# Progress and Accomplishments

## Lessons Learned:

- Mechatronics have a high sensitivity to flow resistance, dead volume, and thermal considerations.
- Refinement of mass-spring system and coil geometry can dramatically improve reliable operation and performance.
- Manufacturing process engineering and precision are crucial to helium temperature and pressure concerns.

## Accomplishments:

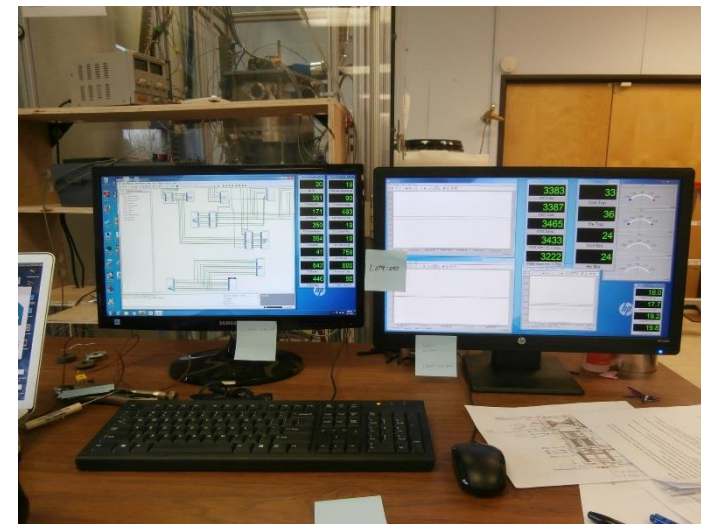
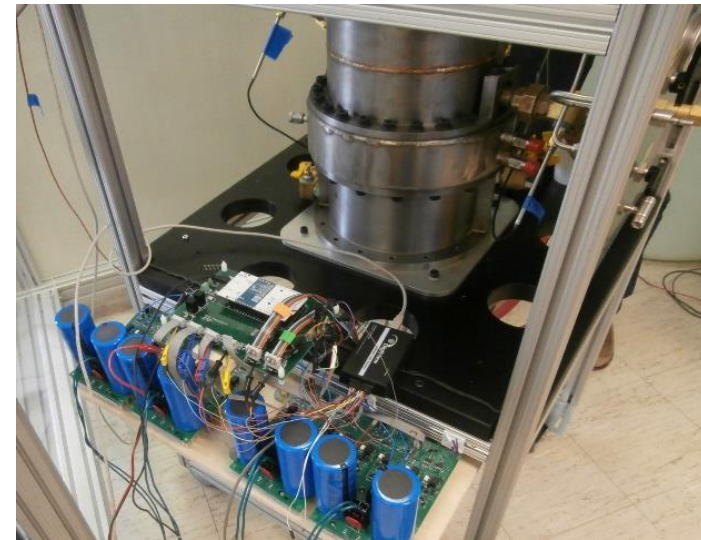
- Built and began testing 1<sup>st</sup> generation prototype.
- Re-designed numerous improved components.
- Developed baseline simulations for tuning, validation, and future enhancements.

## Market Impact:

- Assembled an experienced organization.
- Built extensive network of advocates, partners, suppliers, and qualified customers.

## Awards/Recognition:

- 2015 Long Island's Hottest Startups
- 2014 CleanTech Open NE People's Choice Award
- 2014 Fraunhofer 4 Startups to Watch at NY Energy Week
- 2014 ARPA-E Future Energy – *Winner*
- 2014 & 2013 Defense Energy Summit – *Winner*
- 2013 New England Venture Summit – *Best Presenter*



# Project Integration and Collaboration

## Project Integration:

- Stony Brook University - AERTC
- Engineering expertise in Detroit
- Gas utilities domestically and abroad
- Established advocates in numerous organization (DOD, European Heat Pump Assoc.)

## Partners, Subcontractors, and Collaborators:

- Applied Thermodynamic Apparatus (ATA)
- Oak Ridge National Labs
- National Grid, Star Gas, Par Group
- Fala Technologies, LoDolce, MicroTube, HandyTube, Bruce Diamond

## Communications:

### Exhibits:

2014-15 ARPA-E, 2013-14 New England Venture Summit, 2013-14 Defense Energy Summit, 2013-14 Advanced Energy Conference, 2012-14 National Academy of Sciences

### Publications:

2014 American Gas Magazine, DOE Non-Vapor Compression HVAC Technologies, China Lake High Tech. Association Journal, Newsday, LI Business News, CBS News Affiliate



# Next Steps and Future Plans

Project Schedule										
Project Start: 9/1/2013		Completed Work								
Project End (originally planned): 9/1/2014		Active Task (in progress work)								
Project End (anticipated): 3/31/2016		Milestone/Deliverable (Originally Planed)								
		Milestone/Deliverable (Actual)								
	FY 2013	FY 2014				FY 2015				FY 2016
Task	Q4 (Oct - Dec)	Q1 (Jan - Mar)	Q2 (Apr - Jun)	Q3 (Jul - Sep)	Q4 (Oct - Dec)	Q1 (Jan - Mar)	Q2 (Apr - Jun)	Q3 (Jul - Sep)	Q4 (Oct - Dec)	Q1 (Jan - Mar)
<b>Past Work</b>										
Q4 Milestone: Concept Definition	◆									
Q1 Milestone: Simulation and Design		◆								
Q2 Milestone: Procurement and Build			◆		◆					
<b>Current/Future Work</b>										
Q3 Milestone: Testing				◆			◆			
Q4 Milestone: Testing ORNL					◆				◆	

- Large scale demonstration
- Further optimized design:
  - Analysis of flow resistance and dead volume of critical components
  - Simplified manufacturing design
  - Analysis of innovative technologies and materials
- Focus on cost reduction and durability to decrease time to production / commercialization



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# REFERENCE SLIDES



# Project Budget

**Project Budget:** Total budget of \$1,103,810 including \$200,000 allocated to ORNL for 3<sup>rd</sup> party testing and validation (\$903,810 operating budget).

Total DOE / ThermoLift cost share of 60.85% (\$550,000) / 39.15% (\$353,810).

**Variiances:** Development program was extended from proposed completion date of September 2014 to March 2016. No changes to the budget were undertaken.

**Cost to Date:** Total spent = \$863,280 / \$1,103,810 ≈ 78.1% of total budget.

**Additional Funding:** ThermoLift has been awarded an additional \$482,000 in NYSERDA funding and approximately \$4,500,000 in private capital (TopSpin Partners, Long Island Angel Network, and others).

## Budget History

10/1/2013 – 9/30/2014 (past)		10/1/2014 - Current (current)		Current – 3/31/2016 (planned)	
DOE	Cost-share	DOE	Cost-share	DOE	Cost-share
\$525,331	\$337,949	\$0	\$0	\$24,669	\$15,861
				+ \$200,000 (ORNL)	

# Project Plan & Schedule

## Project Schedule

Project Schedule										
Project Start: 9/1/2013		Completed Work								
Project End (originally planned): 9/1/2014		Active Task (in progress work)								
Project End (anticipated): 3/31/2016	◆	Milestone/Deliverable (Originally Planed)								
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	FY 2013	FY 2014				FY 2015				FY 2016
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<b>Past Work</b>										
Q4 Milestone: Concept Definition	◆									
Q1 Milestone: Simulation and Design		◆								
Q2 Milestone: Procurement and Build			◆		◆					
<b>Current/Future Work</b>										
Q3 Milestone: Testing				◆			◆			
Q4 Milestone: Testing ORNL					◆					◆

# ARPA-E 2015 Showcase

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Home » Photo Gallery: 2015 ARPA-E Innovation Summit Technology Showcase

## Photo Gallery: 2015 ARPA-E Innovation Summit Technology Showcase

February 11, 2015 - 2:16pm



### THERMOLIFT: EFFICIENT NATURAL GAS HEATING & COOLING 4 of 10

ThermoLift, a recipient of a grant from the Energy Department's Buildings Technology Office, uses thermal energy from natural gas to heat and cool efficiently – reducing energy costs by up to 50 percent.

Image: Photo courtesy of Matty Greene, Energy Department.



**Allison Lantero**  
Digital Content Specialist, Office of Public Affairs

### RELATED ARTICLES

 [Changing What's Possible at the 2015 ARPA-E Energy Innovation Summit](#)

 [The Future of Manufacturing Takes Shape: 3D Printed Car on Display at Manufacturing Summit](#)

### Do New Technologies Matter?

Each year the Technology Showcase at the ARPA-E Innovation Summit presents cutting-edge energy technologies, and this year was no different. From a heating and cooling system that works on thermal energy powered by natural gas to the first ever 3D-printed car, breakthrough technologies were on display around every corner. Along with these next-generation technologies, experts were on hand to answer questions about how things work and when these new products and systems will be available to the public.

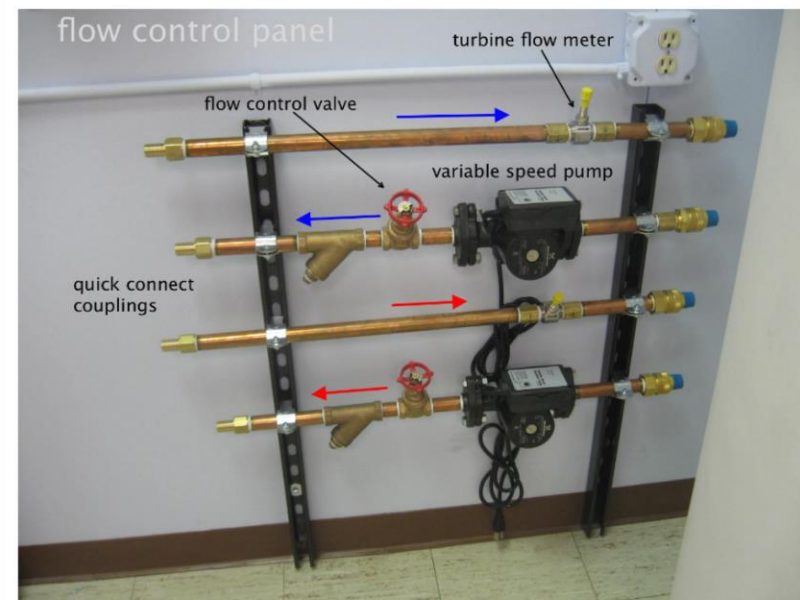
Check out the photo gallery above to learn about some of the amazing technological advances on display at the showcase.

If you'd like to find out more about ARPA-E or the Summit, visit [arpa-e.energy.gov](http://arpa-e.energy.gov).



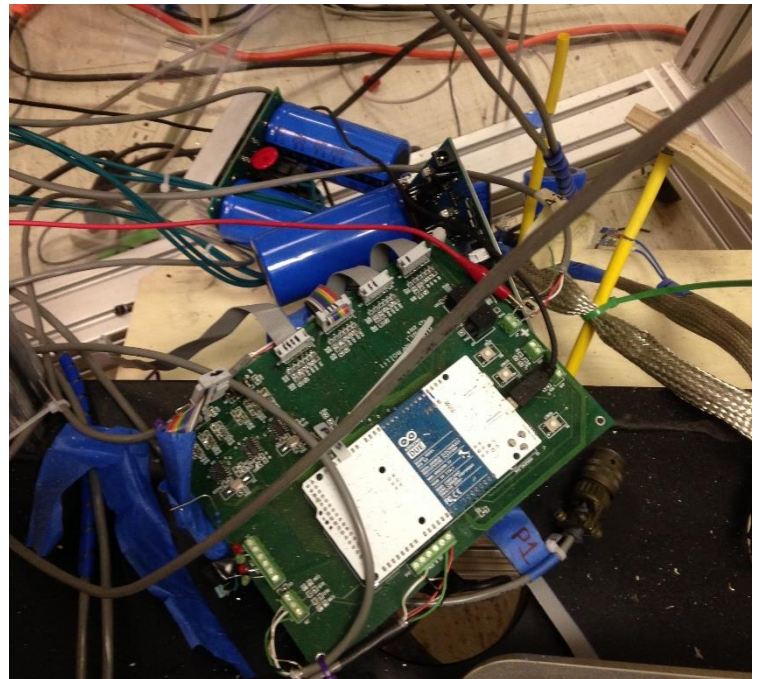
**Matty Greene**  
Videographer

# Test Cell



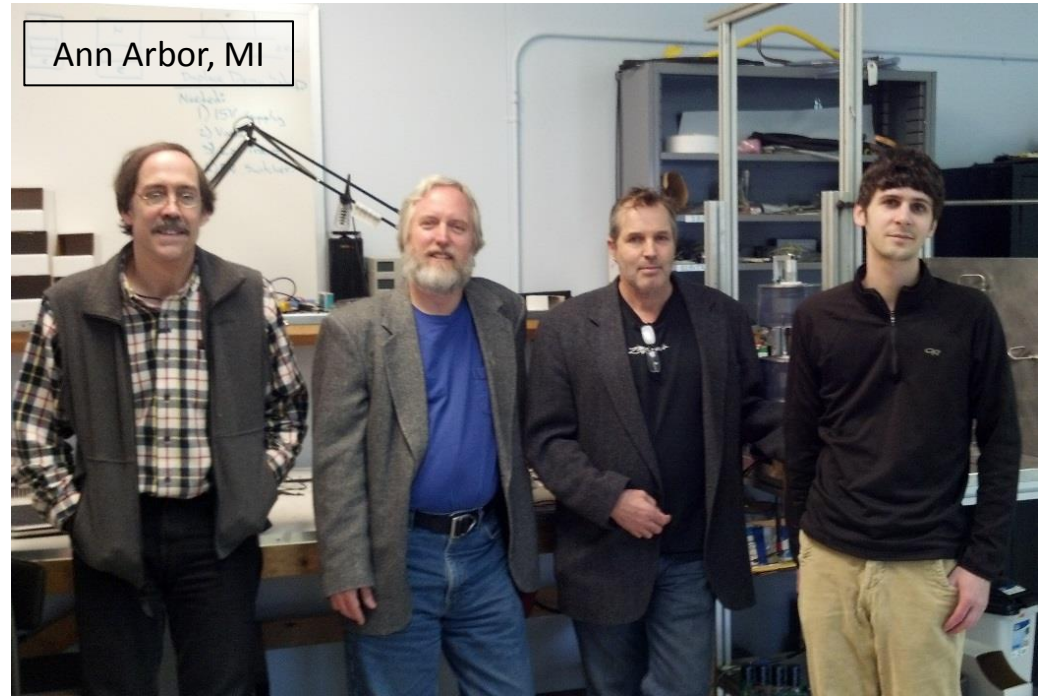
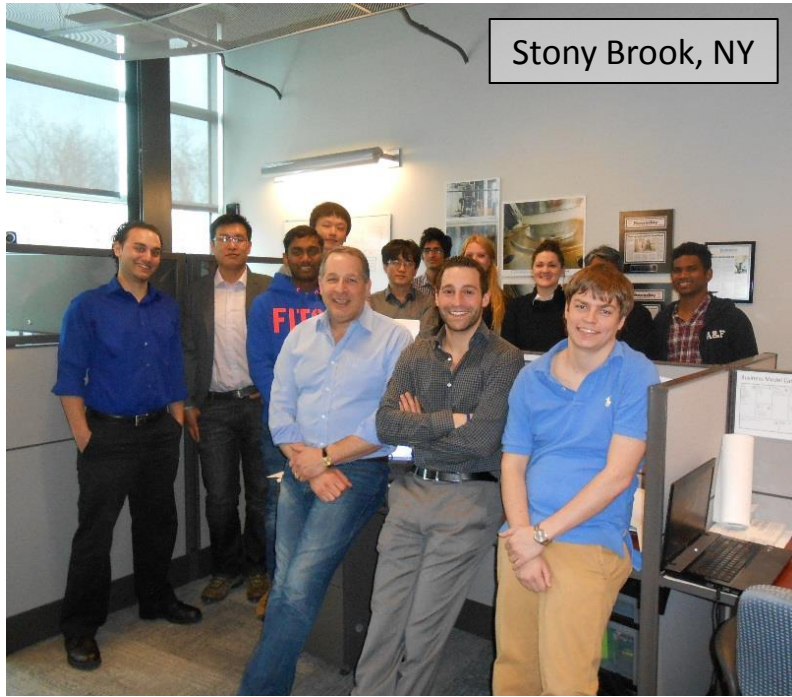


# Manufacturing Failure

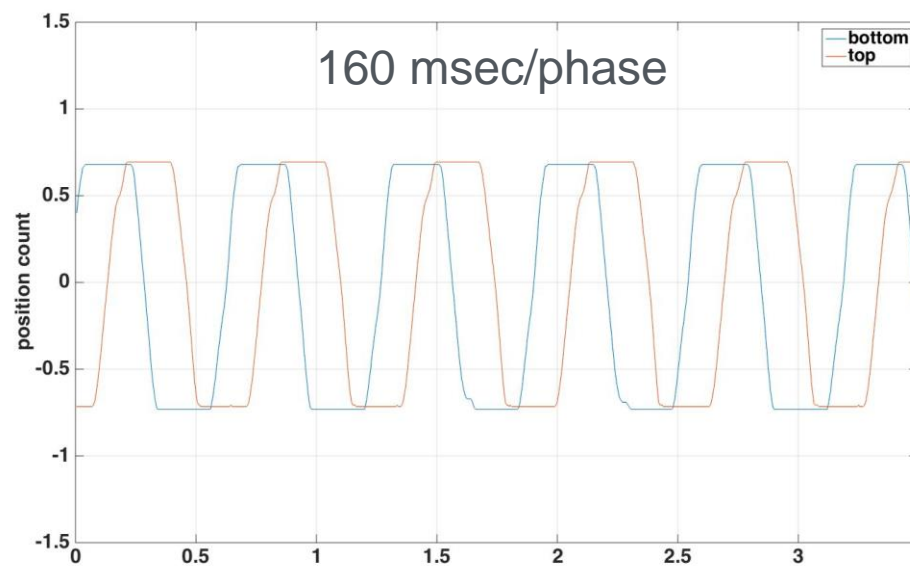
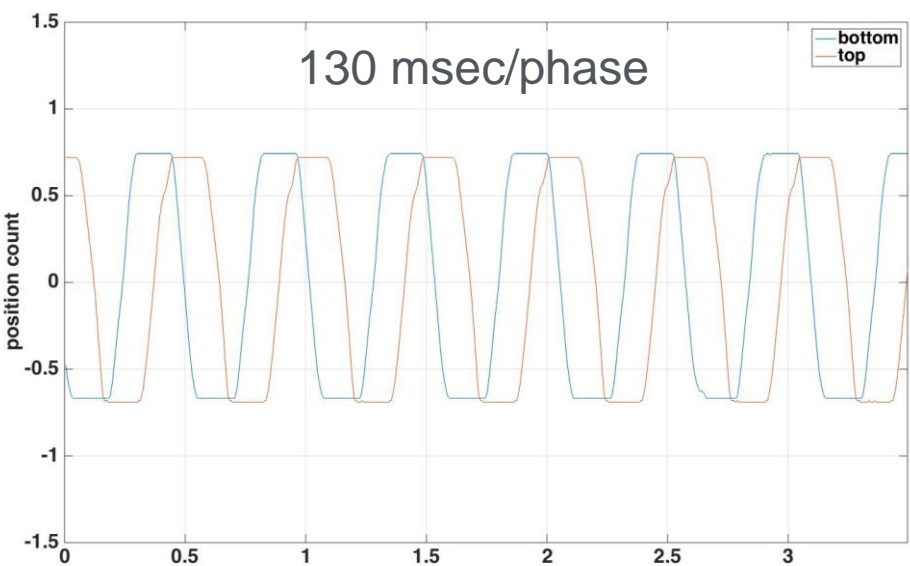
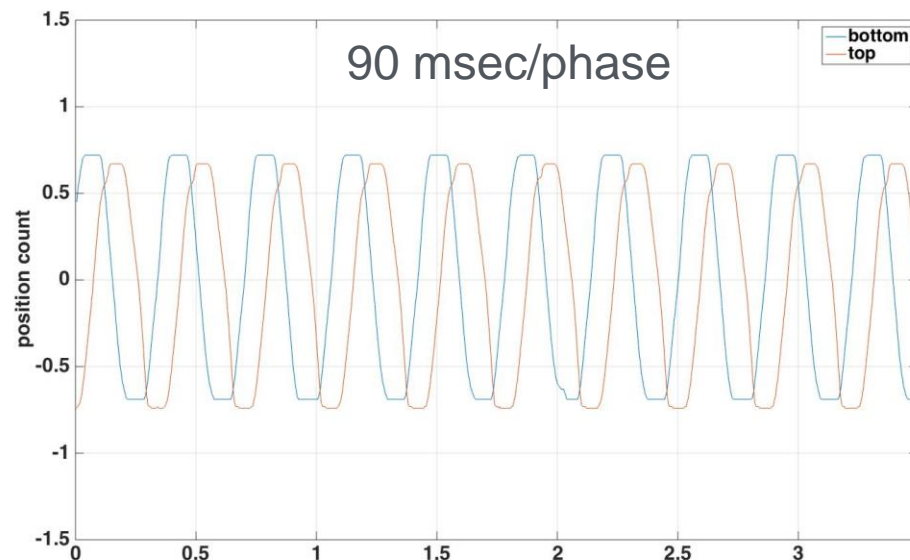
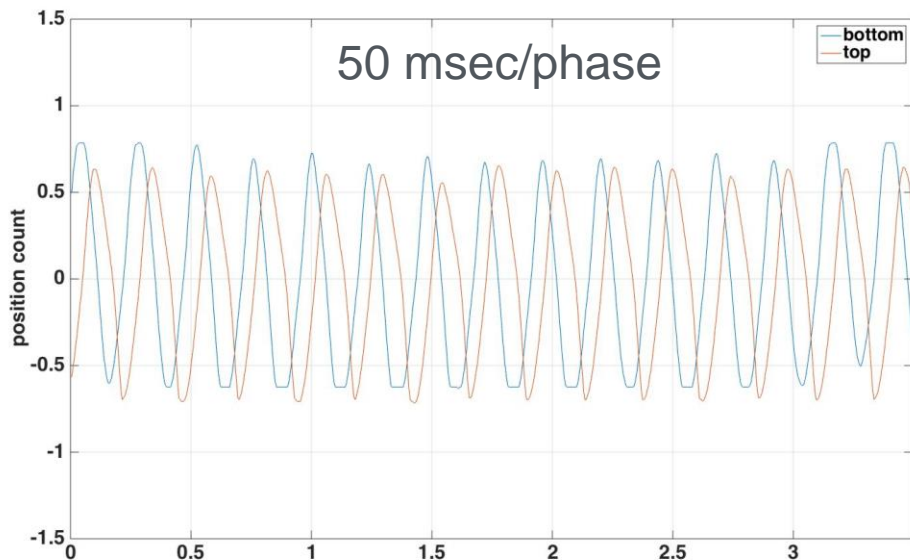




# Team & Progress

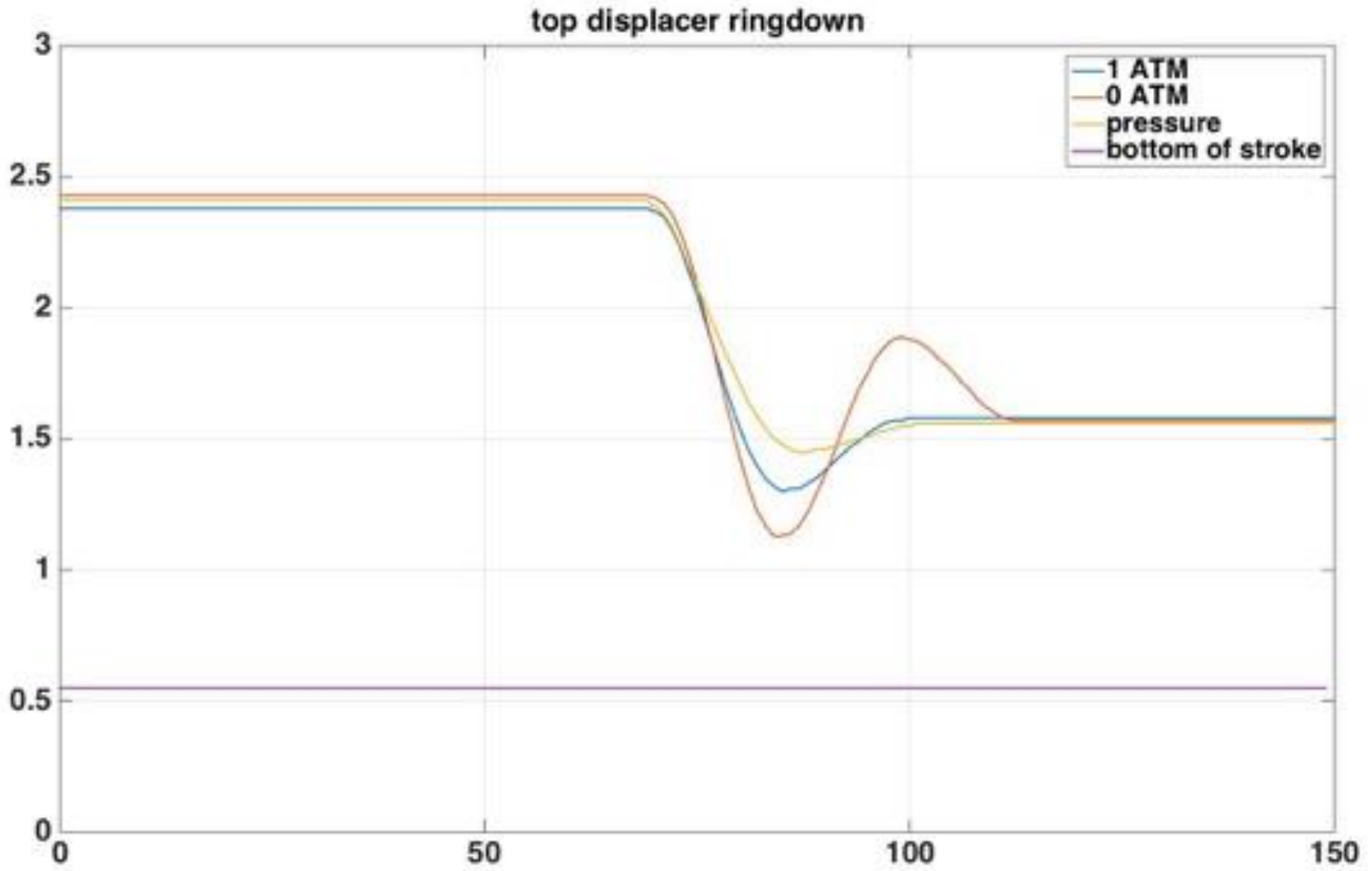


# Variable Frequency & Waveform

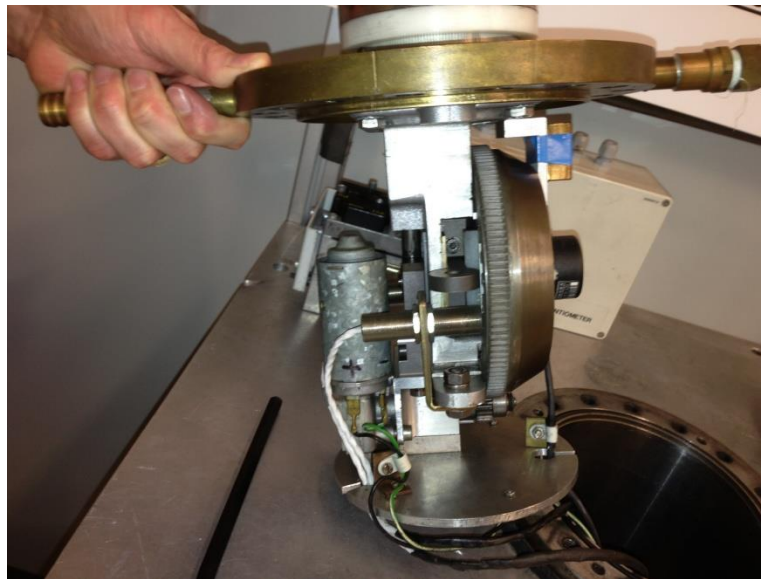
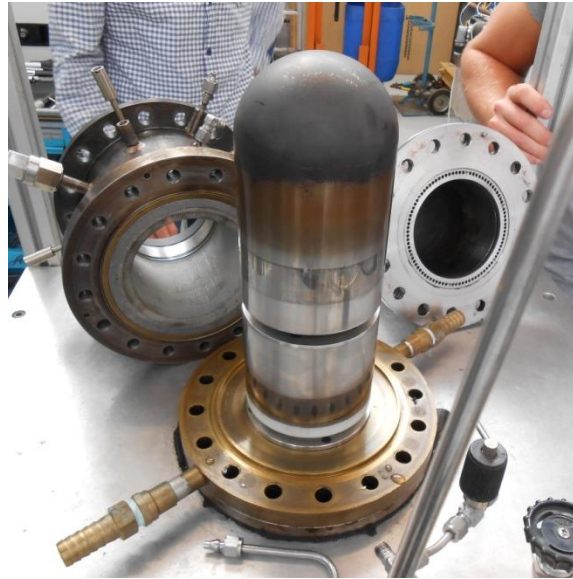




# Ringdown Testing → Reduce Friction

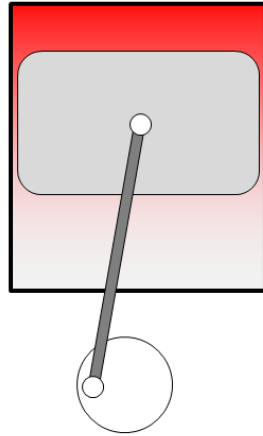


# BVE Energy Demonstrator 1991

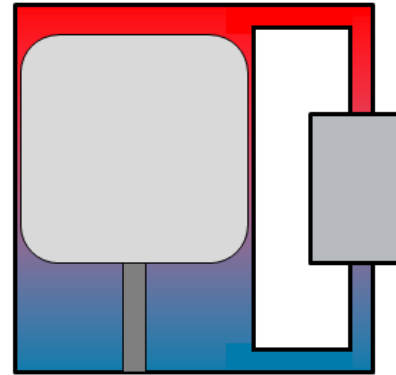


# Thermally Driven Cycles (Vuilleumier & Stirling)

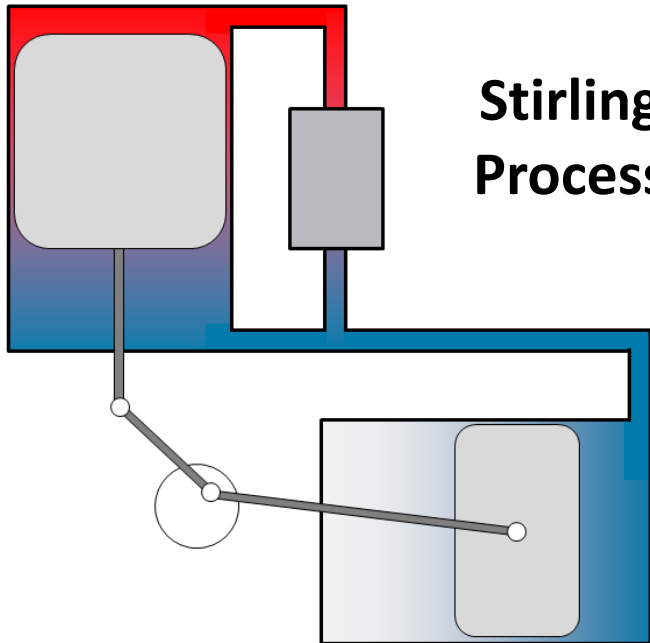
**Mechanical  
Compressor**



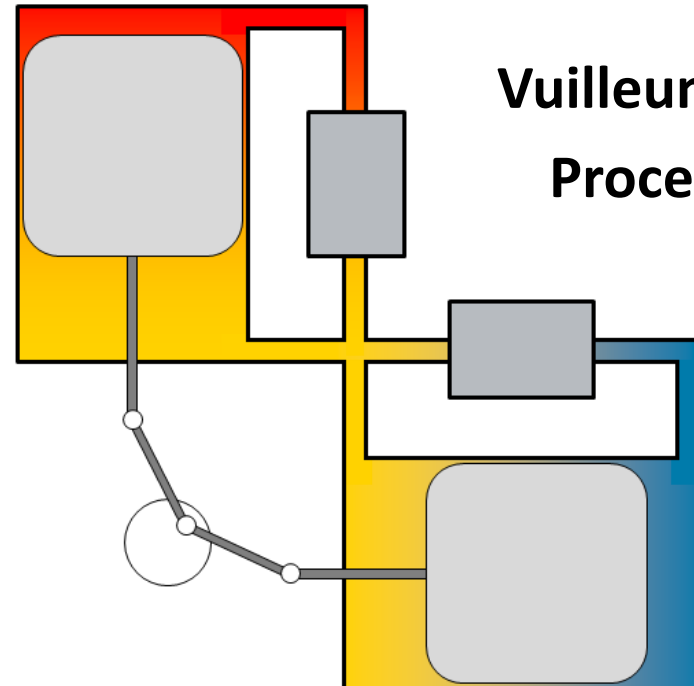
**Thermal  
Compressor**



**Stirling  
Process**



**Vuilleumier  
Process**

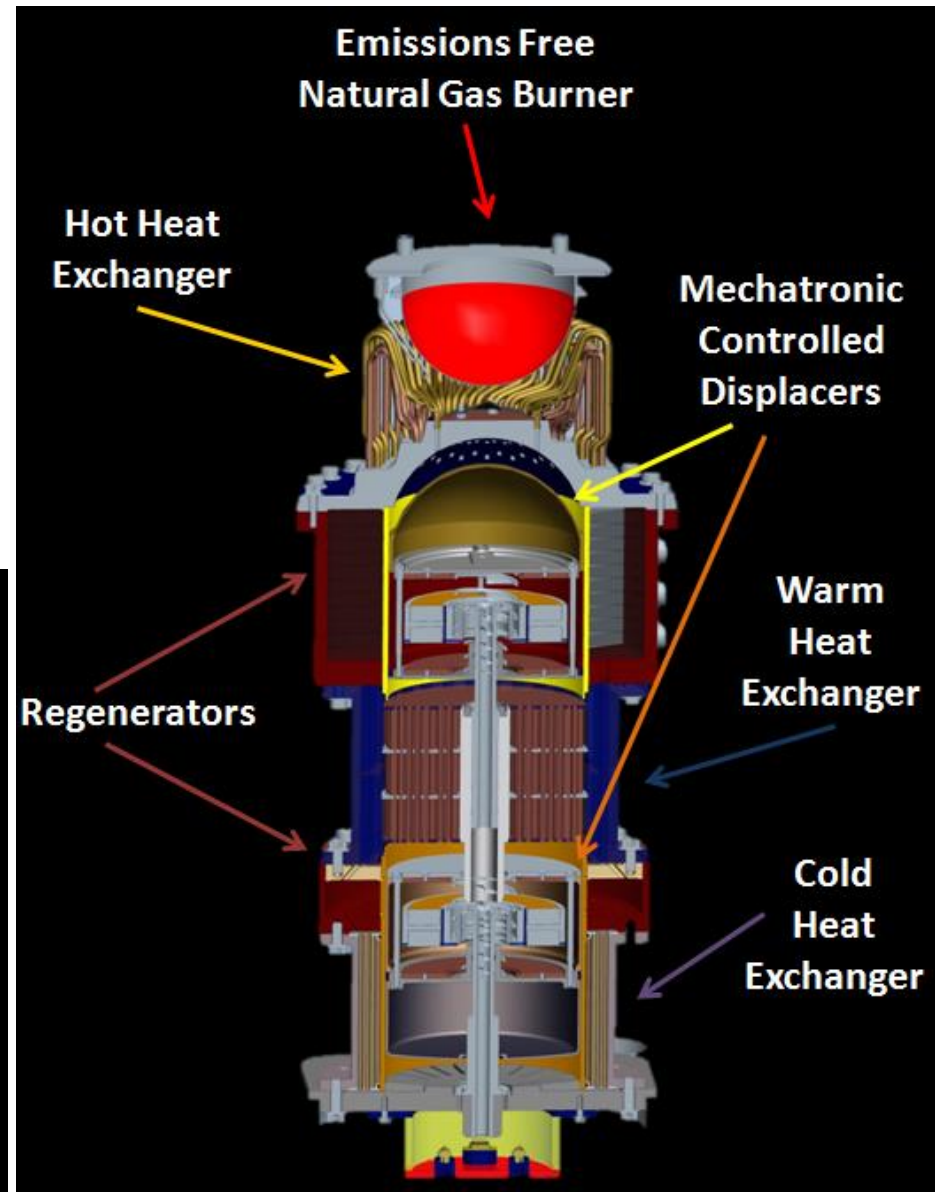
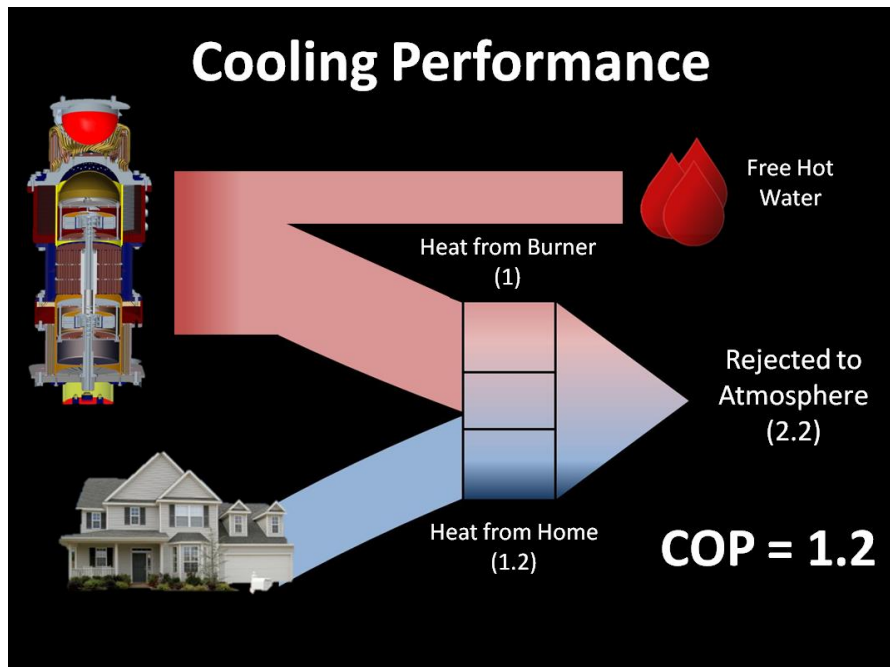




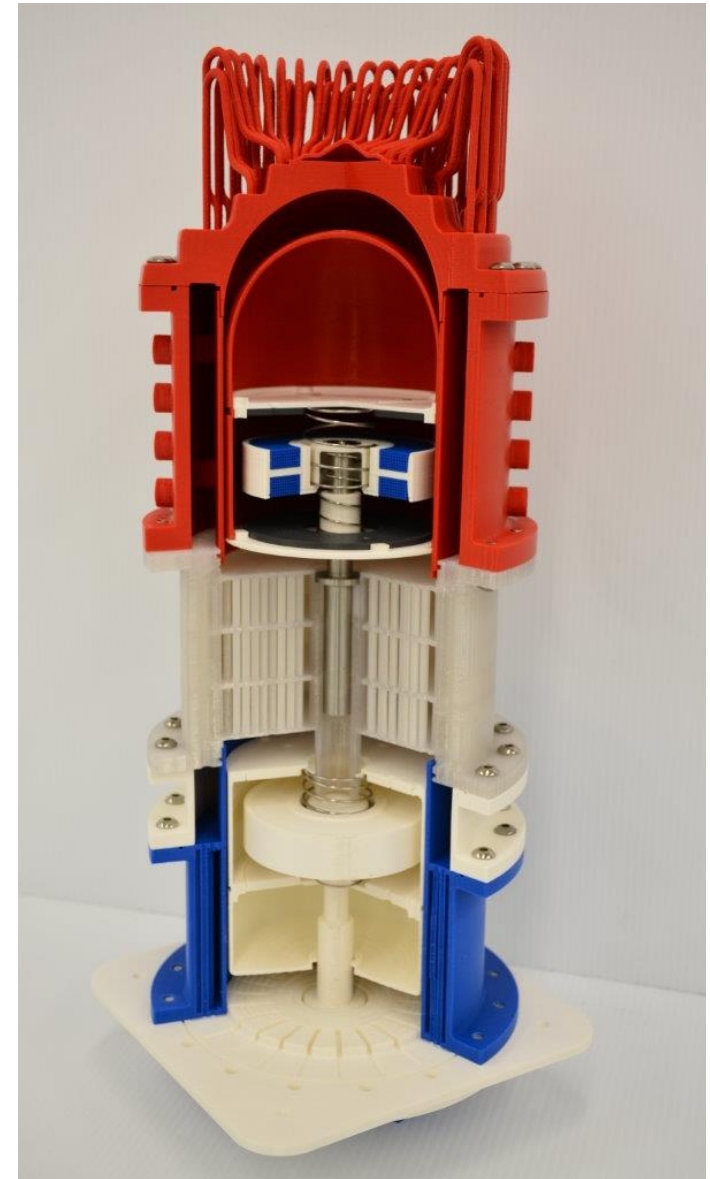
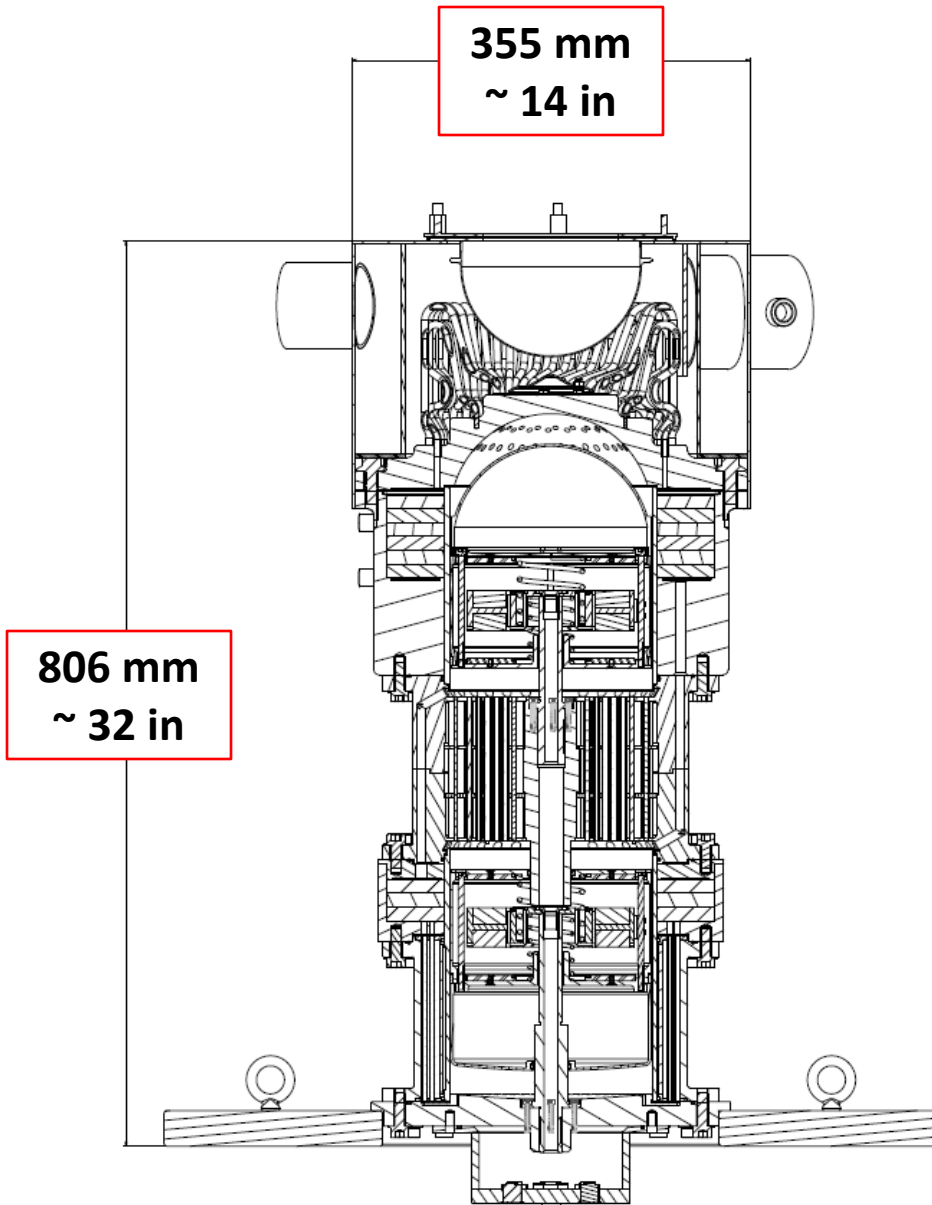
# ThermoLift First Generation

## Benefits:

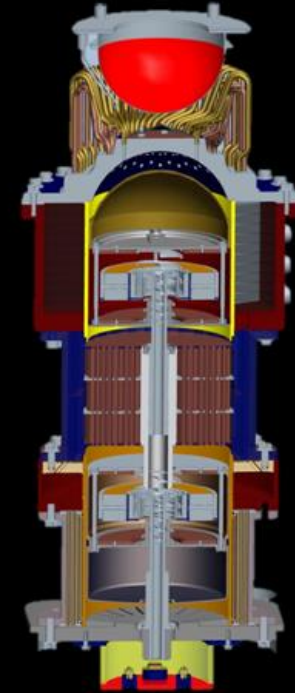
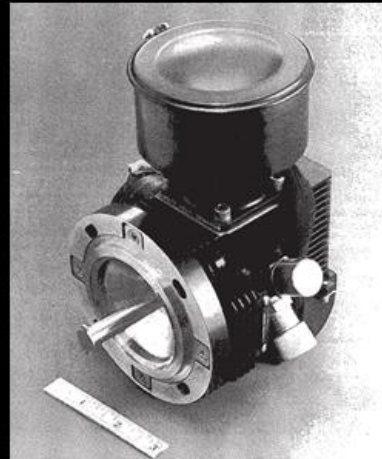
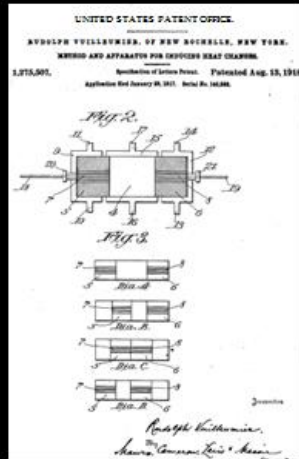
- Single device for complete HVAC
- 30-50% reduction in energy used
- Minimal electricity / Grid Independent
- No refrigerants
- Cold climate performance
- Fuel agnostic
- Simple retrofit & installation
- Smaller footprint / Fully scalable units
- Smart grid enabled controls
- Full efficiency at partial loads



# 3D Printed Prototype



## Proven & Demonstrated



1975-76: USAF, NASA

2012: ThermoLift

1920

1930

1940

1950

1960

1970

1980

1990

2000

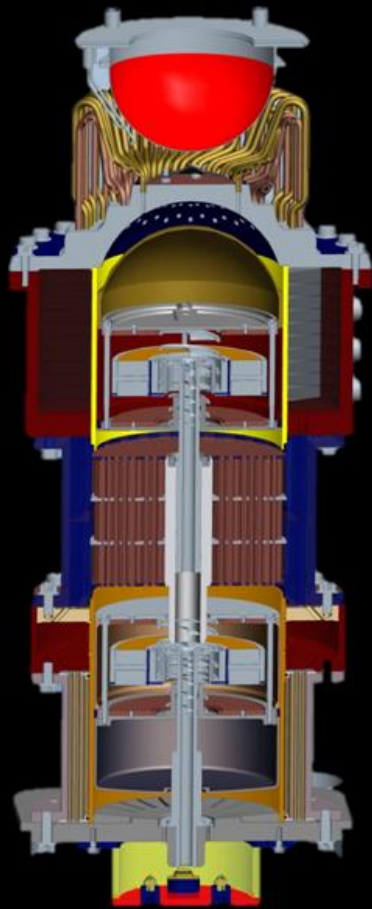
2010

1918: First Patent

1991-98: Demonstrator



## Operational Advantage



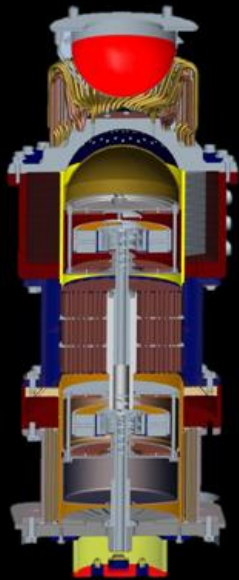
Heat from  
Burner  
(15 kW)

**COP = 1.65**

Combined  
Heat Delivered  
(25 kW)

FREE Heat from  
Ambient (10 kW)

## Cooling Performance



Free Hot Water

Heat from Burner  
(1)

Rejected to  
Atmosphere  
(2.2)

Heat from Home  
(1.2)

**COP = 1.2**



# Thermodynamic Cycle Analysis & Simulation

Two different 1D simulation tools

## Tested effects of:

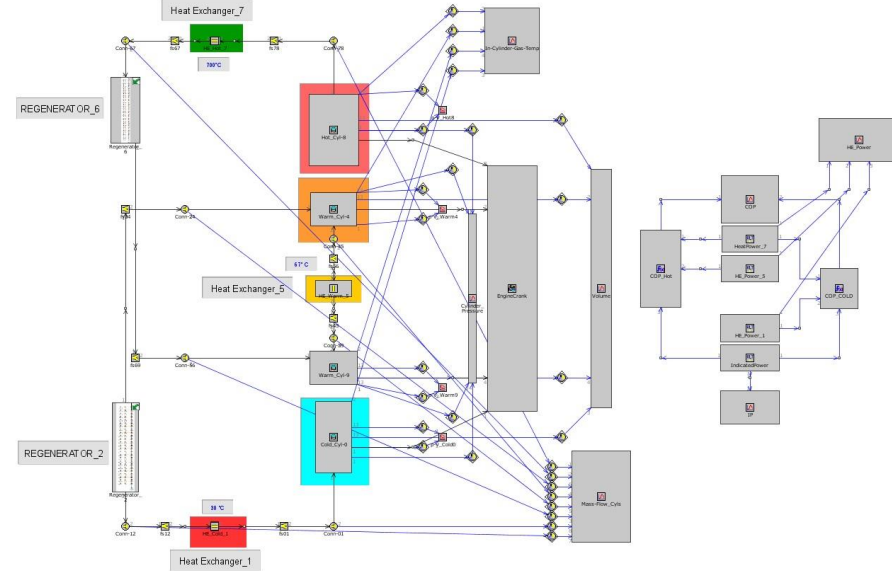
- Non-traditional displacer motion
- Geometry changes
- Different temperatures

## Results:

- Found potential issues with flow resistance leading to improved design

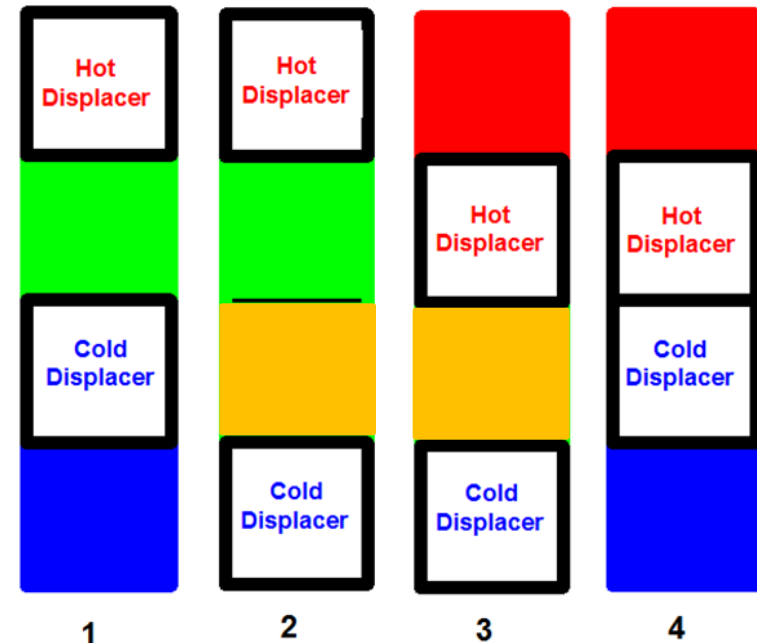
## Next steps:

- Integrate and validate simulation with CFD and prototype test data
- Define critical loss mechanisms



Case	Combine d Energy Rate Out of Fluid; Part HE_Hot_ 7	Combine d Energy Rate Out of Fluid; Part HE_Hot_ Warm_5	Combine d Energy Rate Out of Fluid; Part HE_Cold_ Warm_3	Combine d Energy Rate Out of Fluid; Part HE_Cold_ 1	Output; Part COP_Hot
No Unit	kW	kW	kW	kW	-
4	-13.102	11.140	12.378	-9.492	1.80
5	-14.136	11.646	15.933	-12.120	1.95
6	-15.131	12.132	19.697	-14.518	2.10
7	-16.028	12.819	23.409	-16.332	2.26

**COP**  
**2.26**





# Heat Exchanger CFD

## Using 2D CFD:

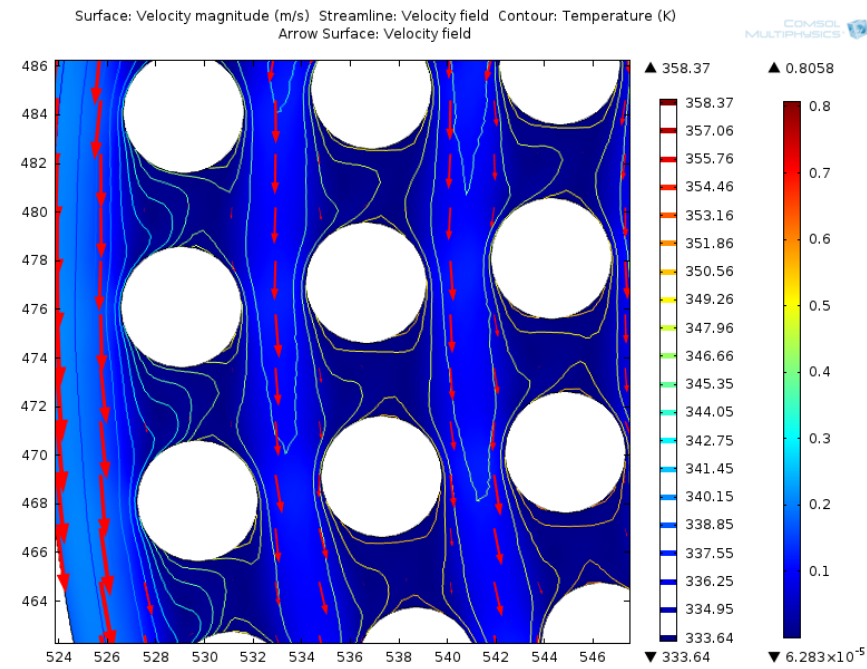
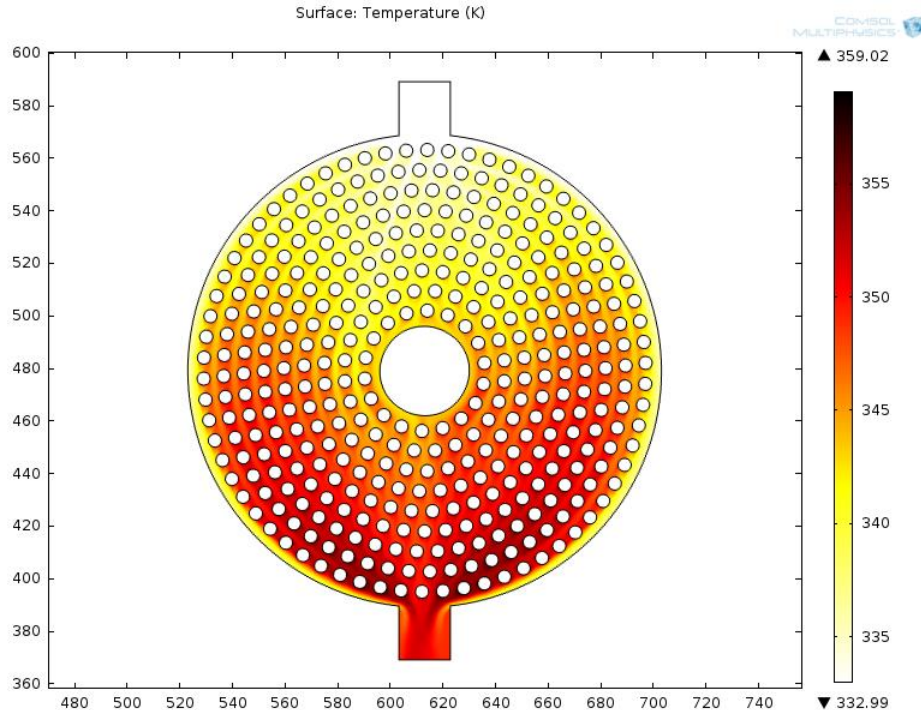
- Simulated fluid flow and heat transfer between helium working gas and hydronic distribution system

## Results:

- Required design changes (length, # tubes, baffles) to ensure sufficient heat transfer with acceptable pressure drops

## Next steps:

- Validate simulation data including losses (e.g. entrance / exit effects)
- Further analyze flow resistance versus dead volume performance tradeoff
- Develop more effective HX designs for future generations including alternative fuels



## Simulation:

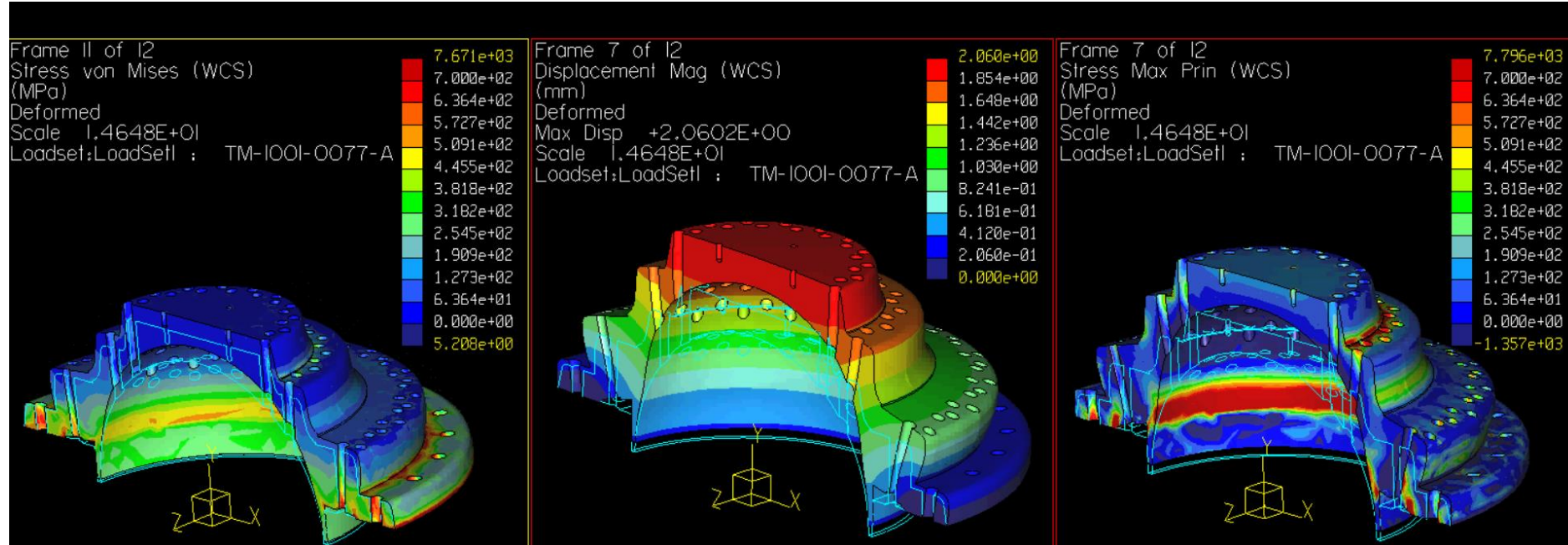
- Found unacceptable stress and strain on the heater head

## Results:

- Designed thicker plate and changed flange locations for proper sealing
- Developed method for better creep evaluation

## Next steps:

- Evaluate materials with superior mechanical properties at high temperatures
- Investigate alternative design structures for thermal stress mitigation

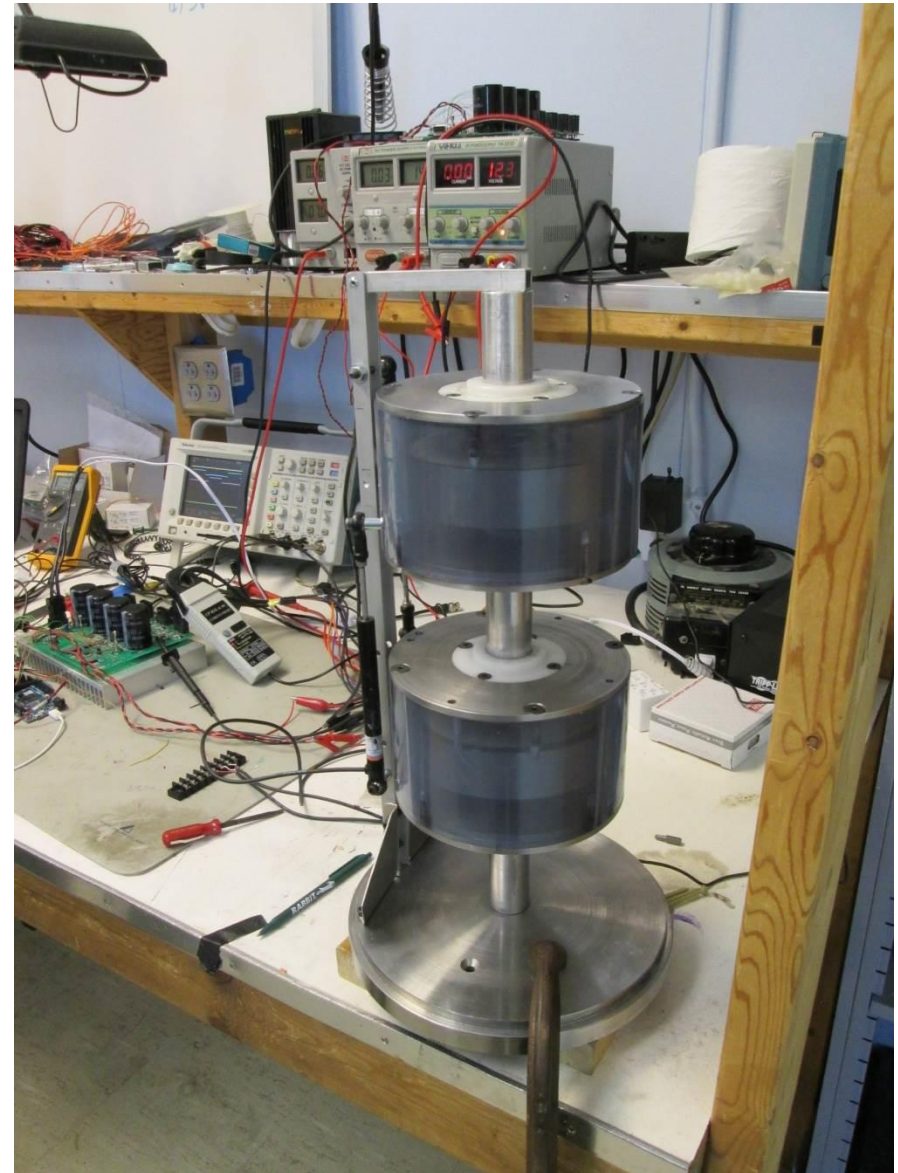


## Development:

- Determined frequency and novel displacer motion
- Built test apparatus
- Tuned spring based on masses and flow resistance
- Developing control software
- Added position sensors for increased feedback
- Tested in ambient air

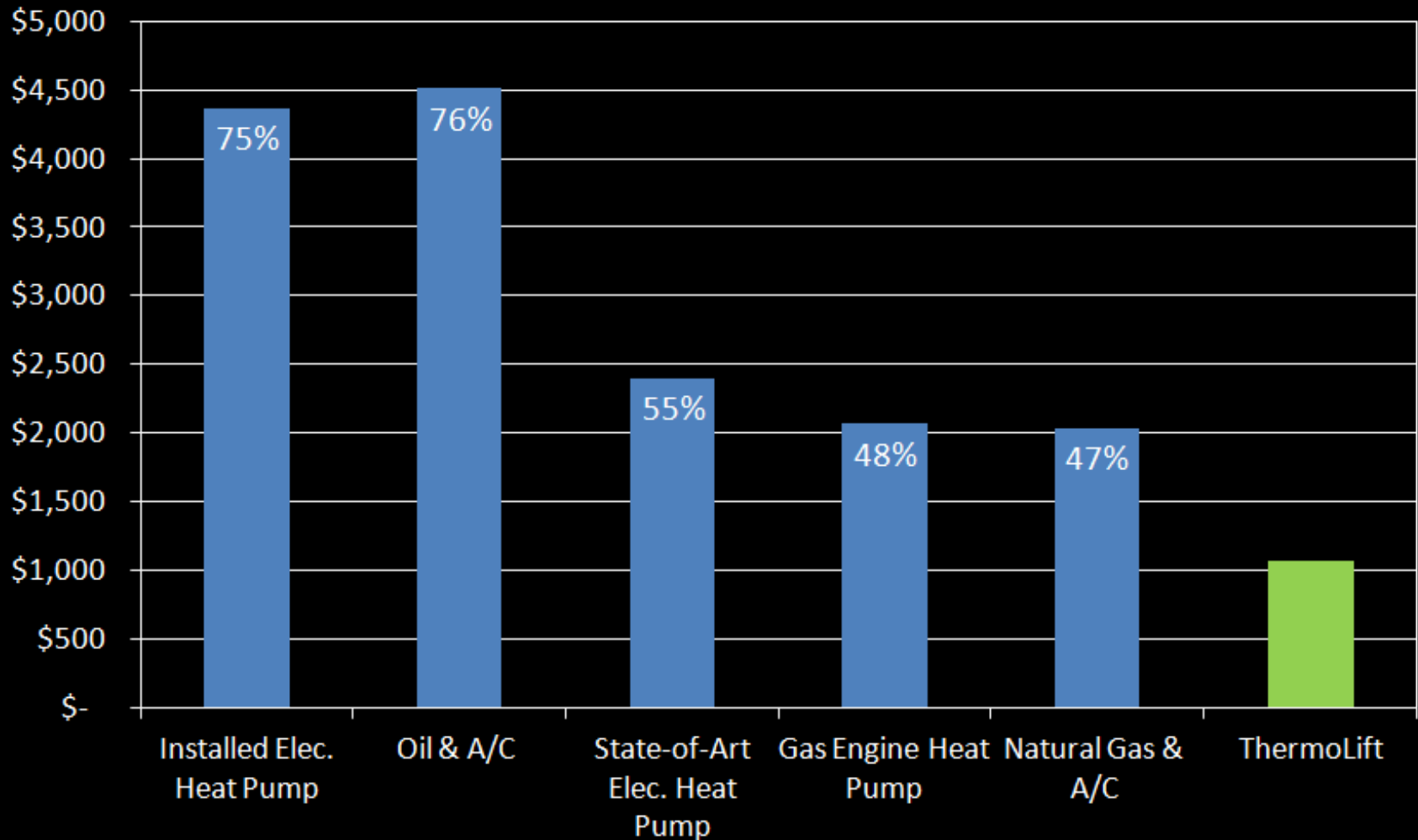
## Next steps:

- Testing in operational environment

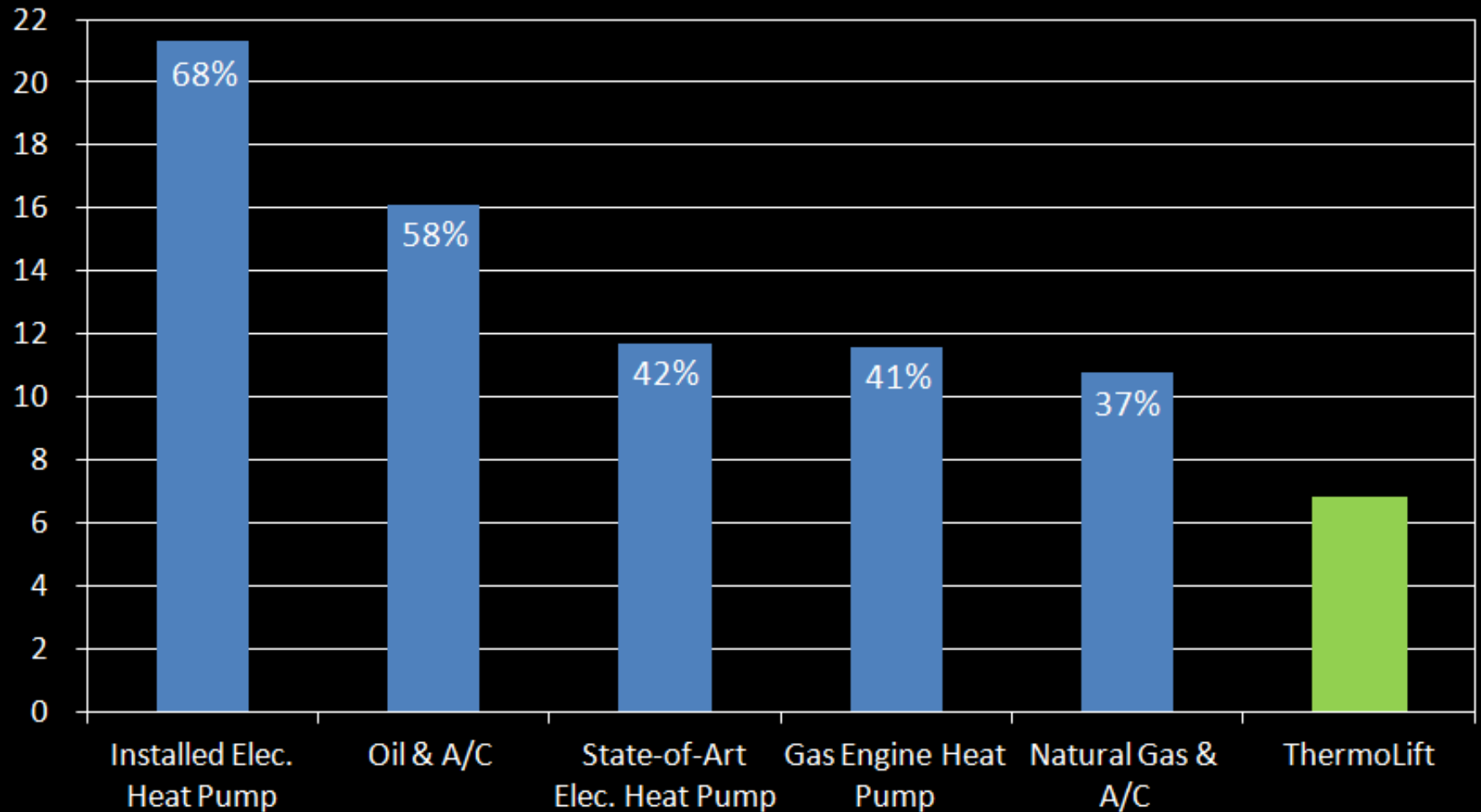




## Annual Cost of Energy (Heating, Cooling and Hot Water)



## Annual CO<sub>2</sub> Emissions (Tons)



## Customer Value Proposition

### Lower Capital Expense:

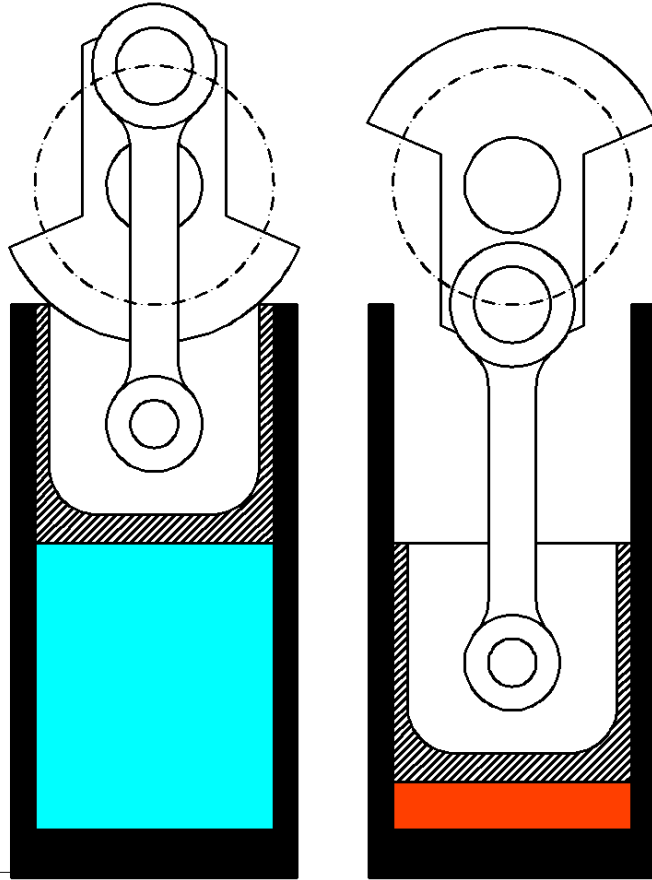
System Function	Efficiency	State of the Art Equipment Cost	ThermoLift Equipment Cost
Space Heating	0.96 (AFUE)	\$2,446	\$5,500
Water Heating	0.98 (EF)	\$1,625	
Space Cooling	16 (SEER)	\$3,162	
<b>TOTAL</b>		<b>\$7,233</b>	<b>\$5,500</b>

### Lower Life Cycle Cost:

System	Efficiency (AFUE)	State of the Art Operational Cost	ThermoLift Operational Cost	Annual Cost Savings	20 Year Savings
Gas Furnace	0.96	\$1,220	\$732	\$488	\$9,760
Oil Boiler	0.86	\$2,824	\$1,129	\$1,694	\$33,888



## Mechanical Compression

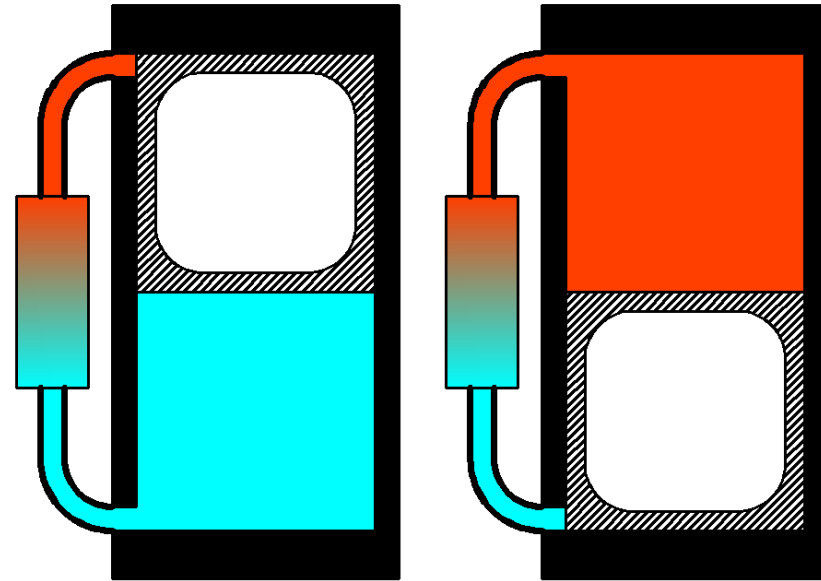


**Polytropic Process:**

$$P_2 = P_1 * (V_1/V_2)^n \quad T_2 = T_1 * (V_1/V_2)^{(n-1)}$$

$V_1 = 600, V_2 = 100, n = 1.4$   
 $T_1 = 273 + 50 \text{ K} = 122^\circ\text{F}$      $T_2 = 273 + 388 \text{ K} = 730^\circ\text{F}$   
 $P_1 = 1 \text{ bar} = 14.5 \text{ psi}$      $P_2 = 12 \text{ bar} = 174 \text{ psi}$

## Thermal Compression

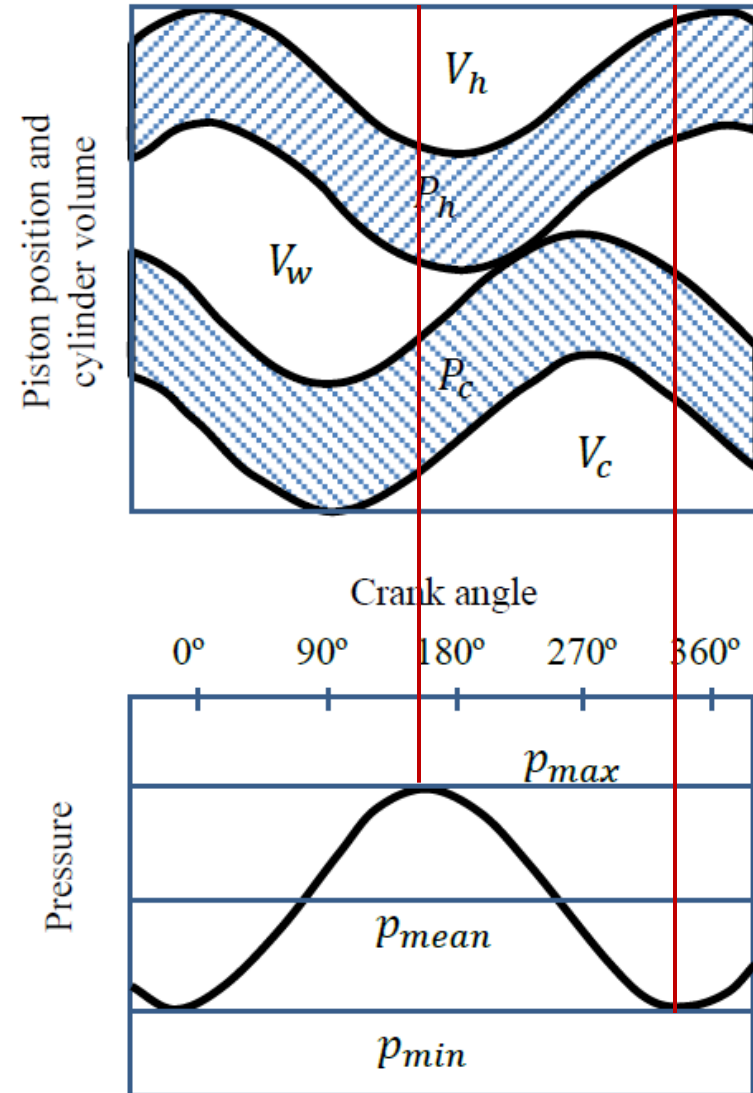
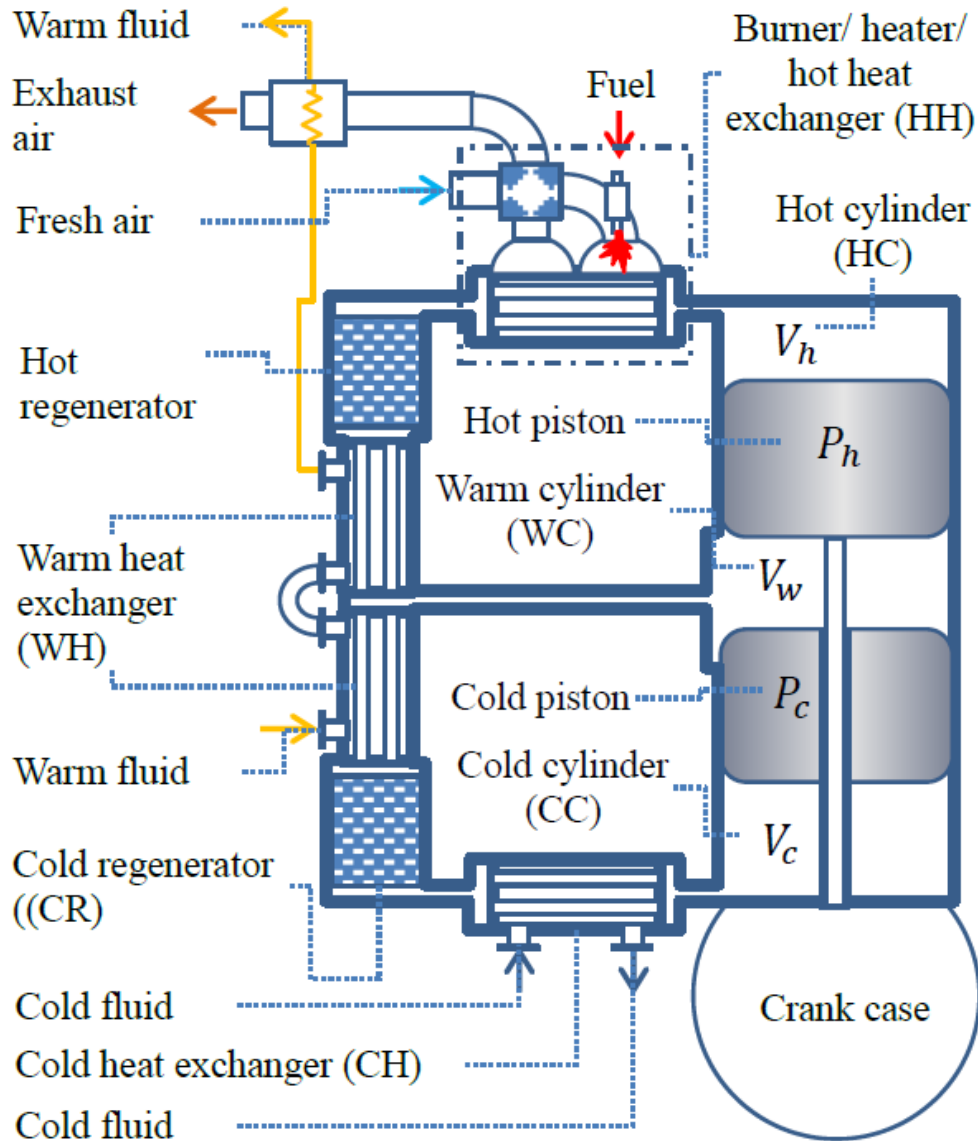


**Isochoric Process:**

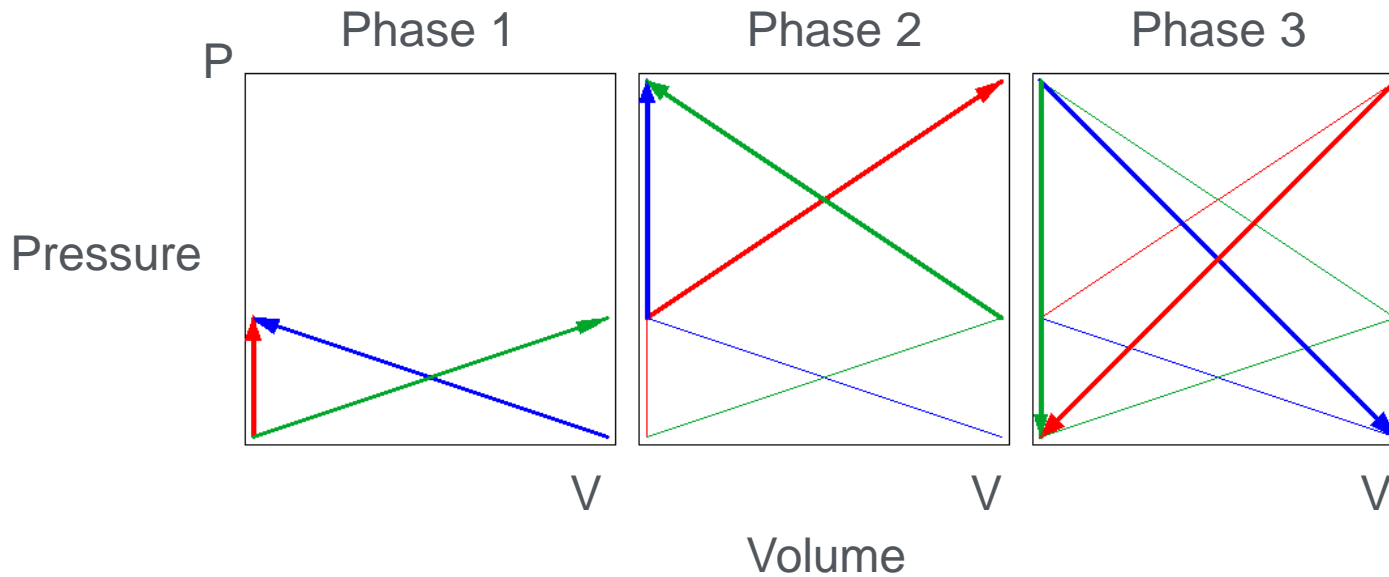
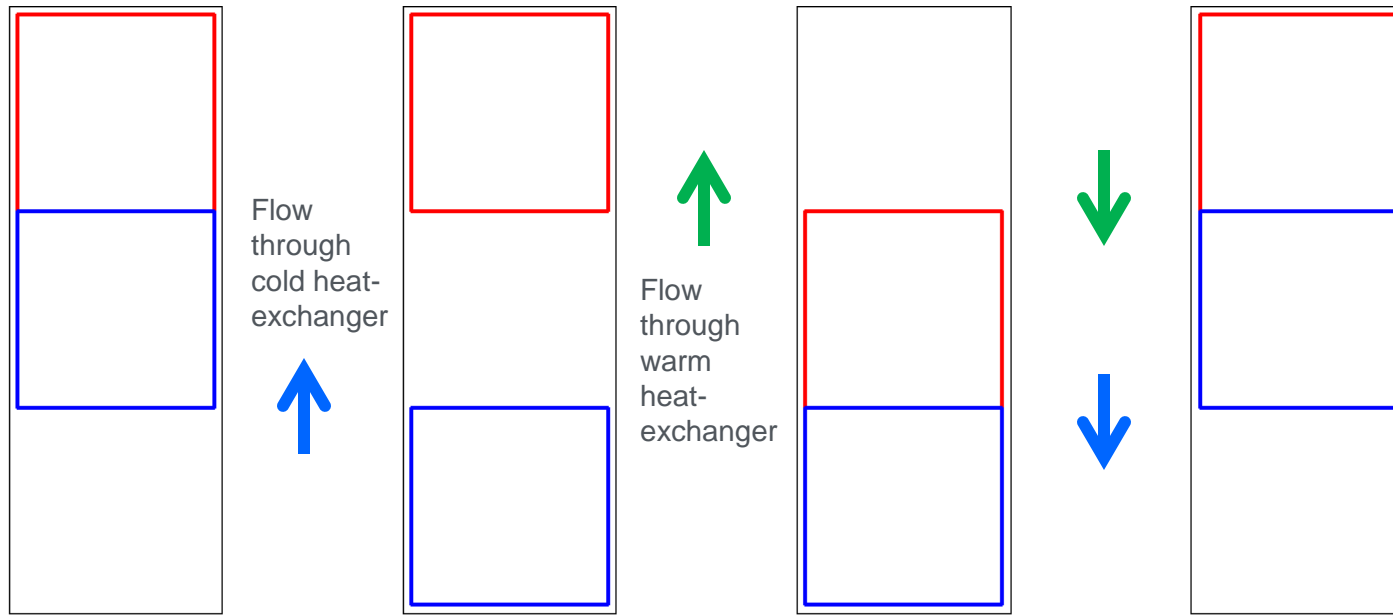
$$P_2 = P_1 * T_2 / T_1$$

$T_1 = 273 + 50 = 122^\circ\text{F}$      $T_2 = 273 + 800 = 1472^\circ\text{F}$   
 $P_1 = 100 \text{ bar} = 1450 \text{ psi}$      $P_2 = 332 \text{ bar} = 4815 \text{ psi}$

# BVE Architecture

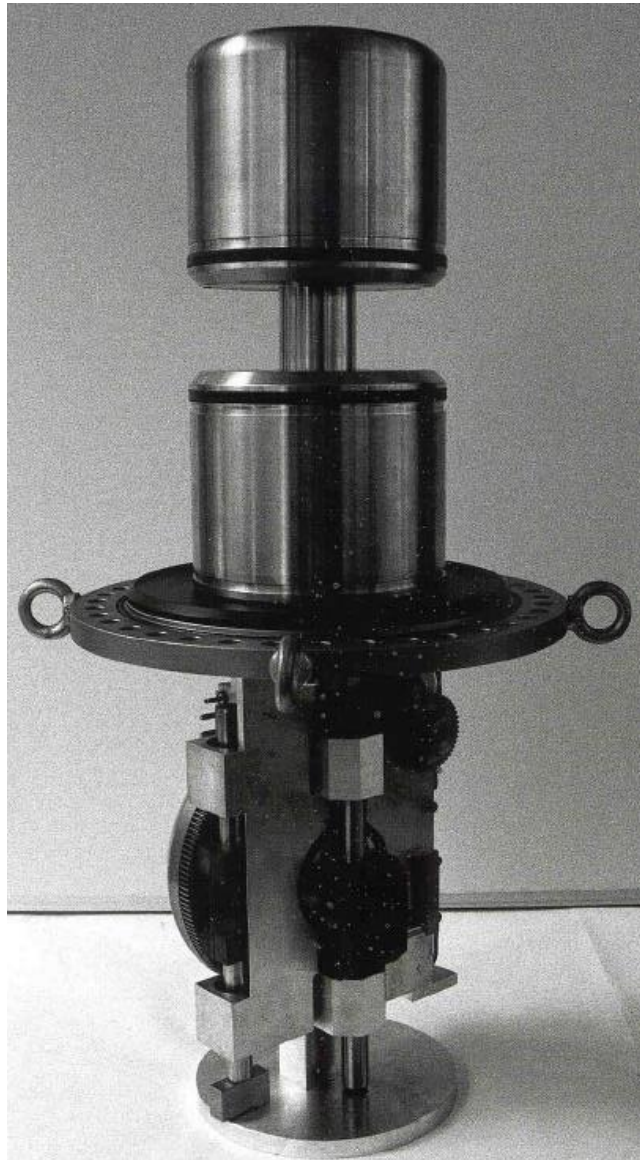


# ThermoLift 3 Phase Process





# 1. First BVE Heat Pump



Schnittzeichnung 20/26  
kW Prototyp

**BVE**

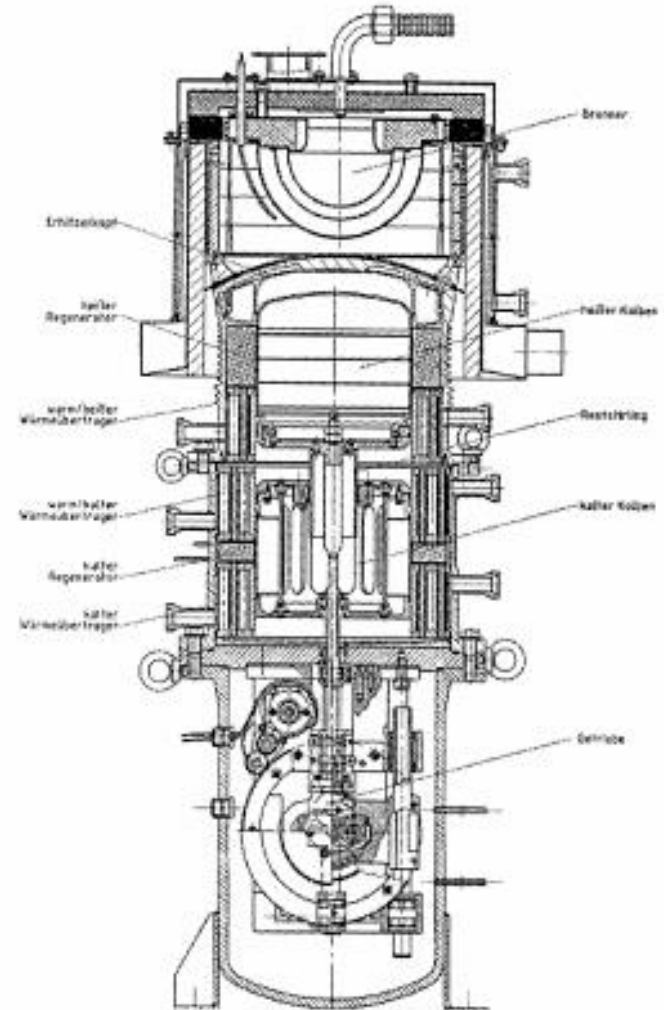
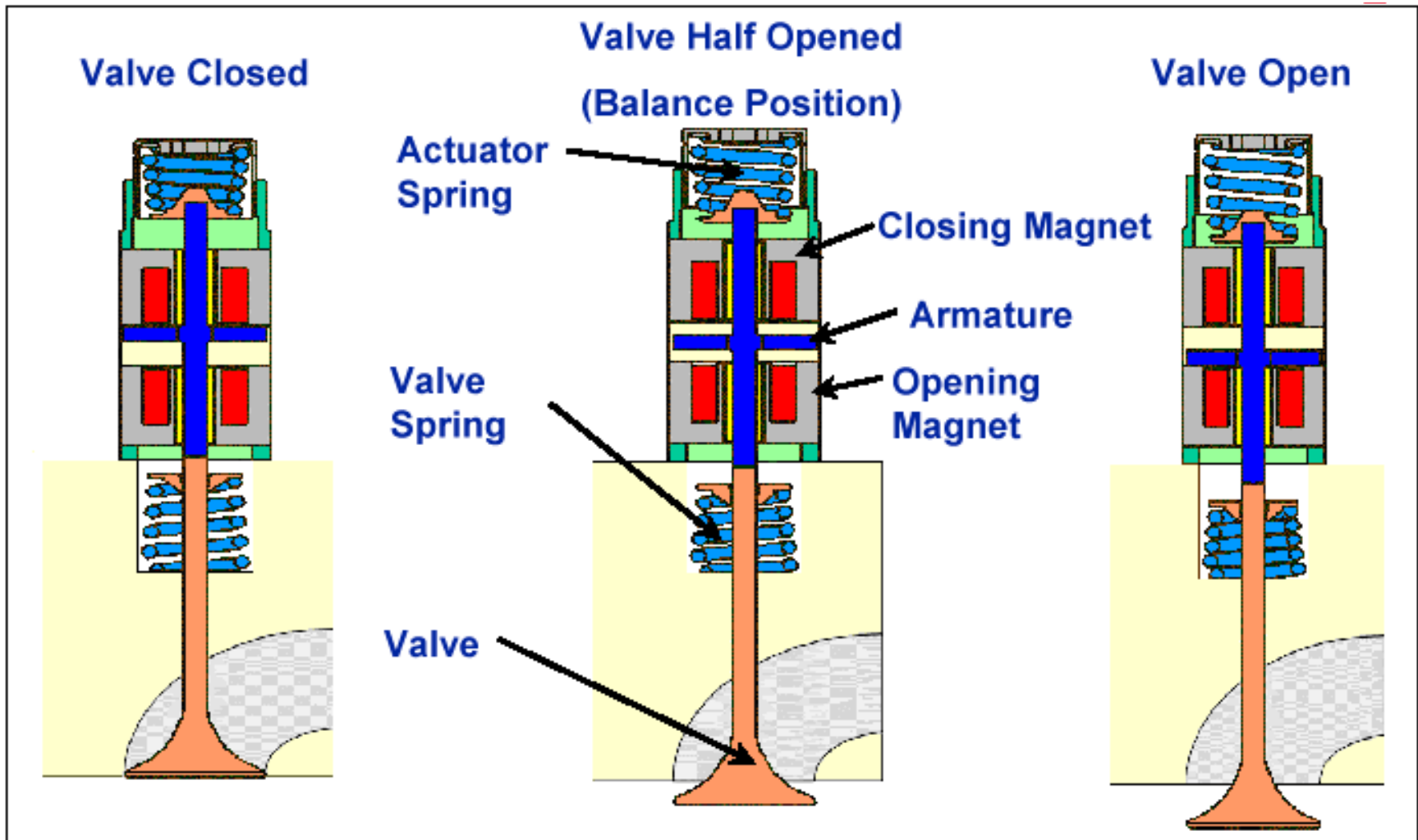


Abb. 4.1 20 kW Prototyp der Vuilleumier-Wärmepumpe

## 2. Electromechanical Valve Actuator

### Operating Principle:





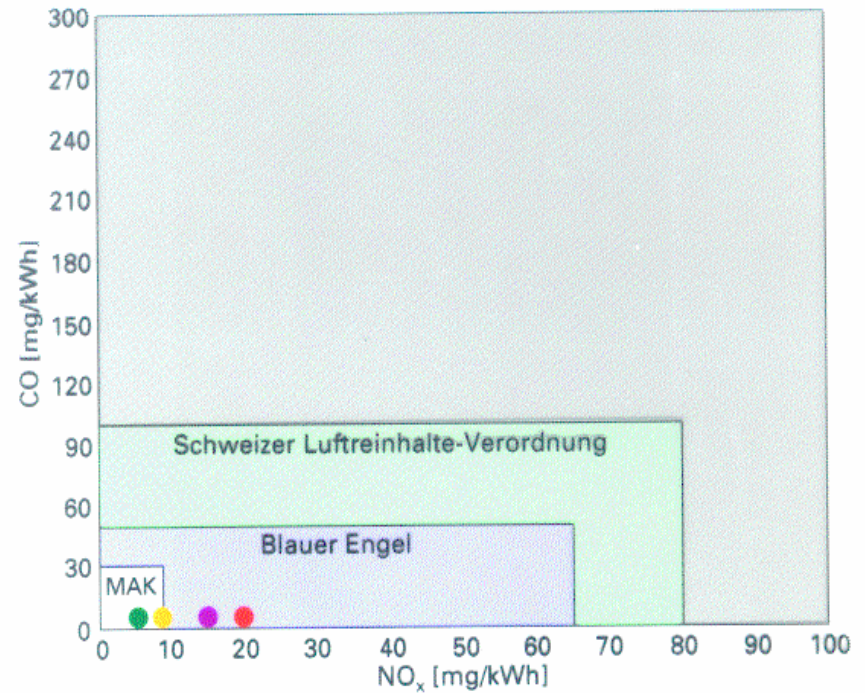
# 3. Emission-free Matrix Radiant Burner

European Environmental Award for  
the MatriX Radiant Burner



Emissionswerte Gas-Brennwertkessel

VIESSMANN



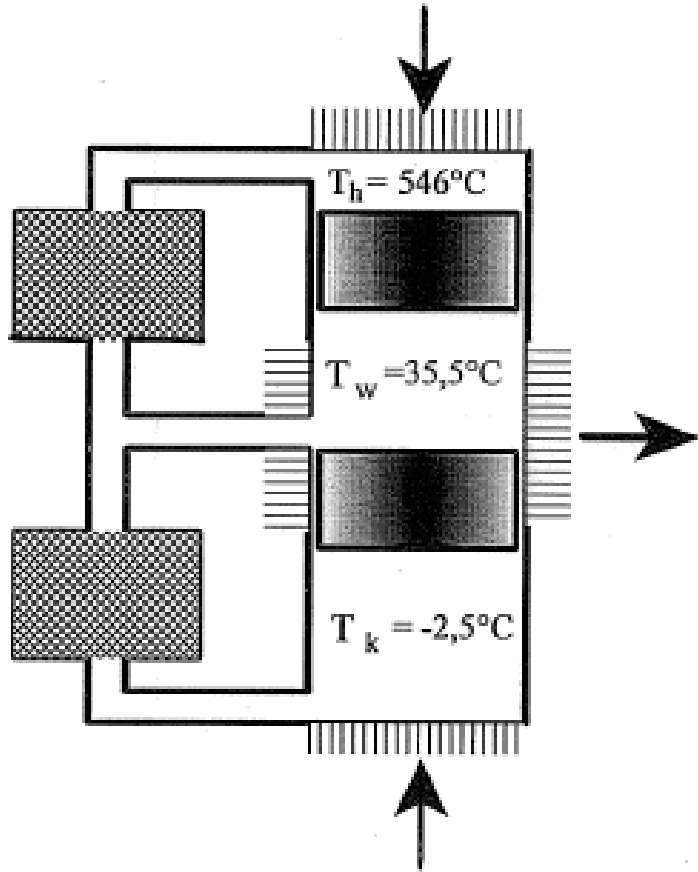
- Condensola-RN Unit mit MatriX-Katbrenner (22 kW)
- Eurola-CB
- Mirola-MB/-MA (18 kW)
- Condensola-RN Unit (22 kW)



# BVE Experimental Results

**general data:**

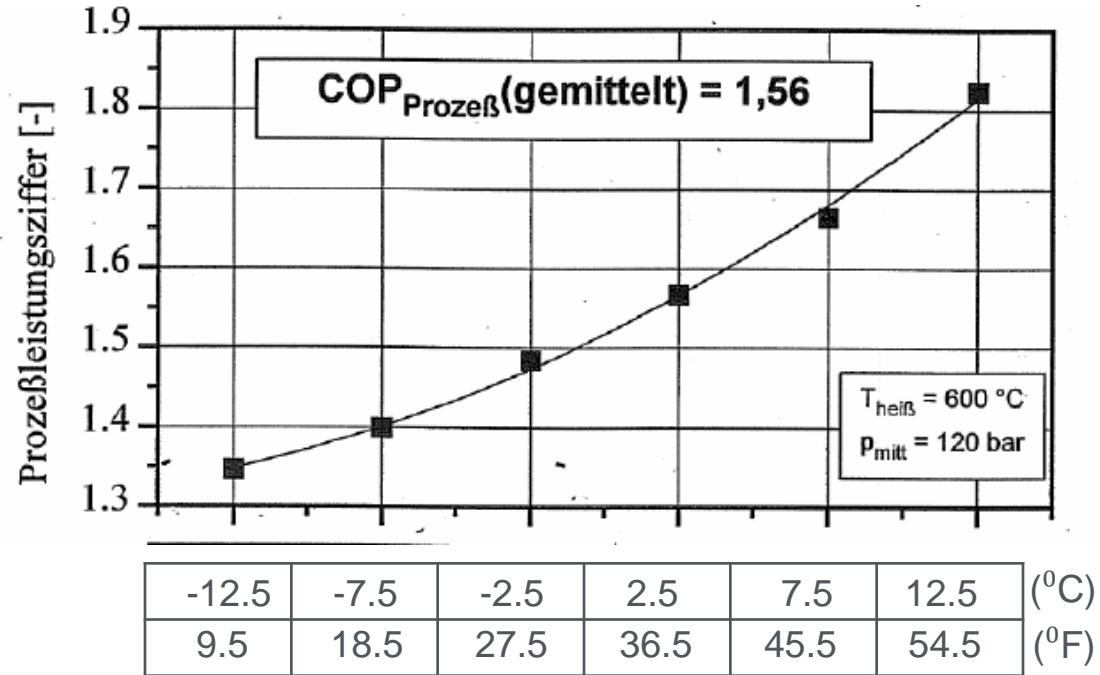
mean pressure 119,2 bar  
 pressure amplitude 8,87 bar  
 speed 456 rpm



**remark:**

- methan as burning gas in order minimize error of lower heat content

## Performance vs. Temperature



**hot:**

mean hot temp. 546,4°C  
 exhaustgastemp. 32,4°C  
 airpreheattemp. 395°C  
 burnerpower 2561 W  
 heat of condens. 168 W  
 $\text{COP}_{\text{burner/lower heat cont.}}$  0,78  
 insulationloss 239 W

**warm:**

mean fluidtemp. 35,5°C  
 heating power 3646 W

**cold:**

mean fluidtemp. -2,5°C  
 cooling power 1167 W