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Package DX Units: Performance Optimization & Field Tests

Background

Unitary equipment is ubiquitous.

- 60% of US commercial space is cooled with RTUs (DOE)
- 54% of commercial building cooling primary energy consumption (EIA)
- Total annual installations over 300,000 units
- Estimated 1.6 million legacy units operating at low efficiency levels

100,000 units at DoD facilities / 20,000 buildings 100,000 units at USPS facilities / 30,000 buildings 500,000 units at 65,000 "big box" retail stores

RTU – Rooftop packaged air-conditioner Unit DOE – U.S. Department of Energy EIA – Energy Information Administration USPS – United States Postal Service









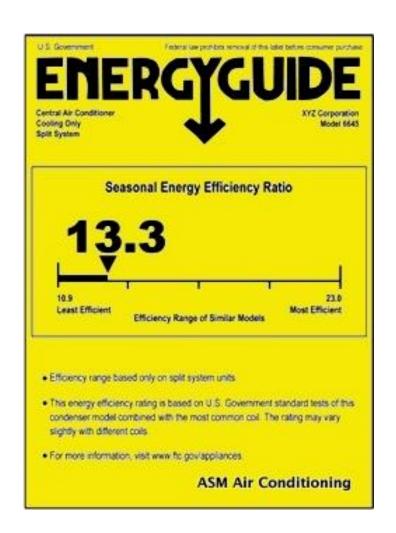
ROOFTOP PORTABLE PAD MOUNT SPLIT SYSTEM

Performance Ratings

"How much cooling you get for the electricity it uses"

Btuh per Watt (MBH per kW)

- EER Energy Efficiency Ratio
 Full Load @95F Ambient
- IEER Integrated Energy Efficiency Ratio
 Weighted Full Load & Part Load @95, 81.5, 68, and 65F
- SEER Seasonal EER
 Part Load @82F Ambient x 0.875
 Cyclic Performance Load Factor
- IPLV Integrated Part-load Value
 Legacy rating (no longer standard)



ANSI/AHRI Standard 340/360-2007

Drivers

1. Energy saving goals continue to rise

- > IEER Integrated Energy Efficiency Ratio 10.0 to 13.0, EER Energy Efficiency Ratio 9.7 to 11.7
- Upcoming DOE 10% and 30% increases in efficiency minimums
- ➤ Single-zone VAV and DDC requirements of Energy Standard 90.1-2013 and Green Standard 189.1-2014

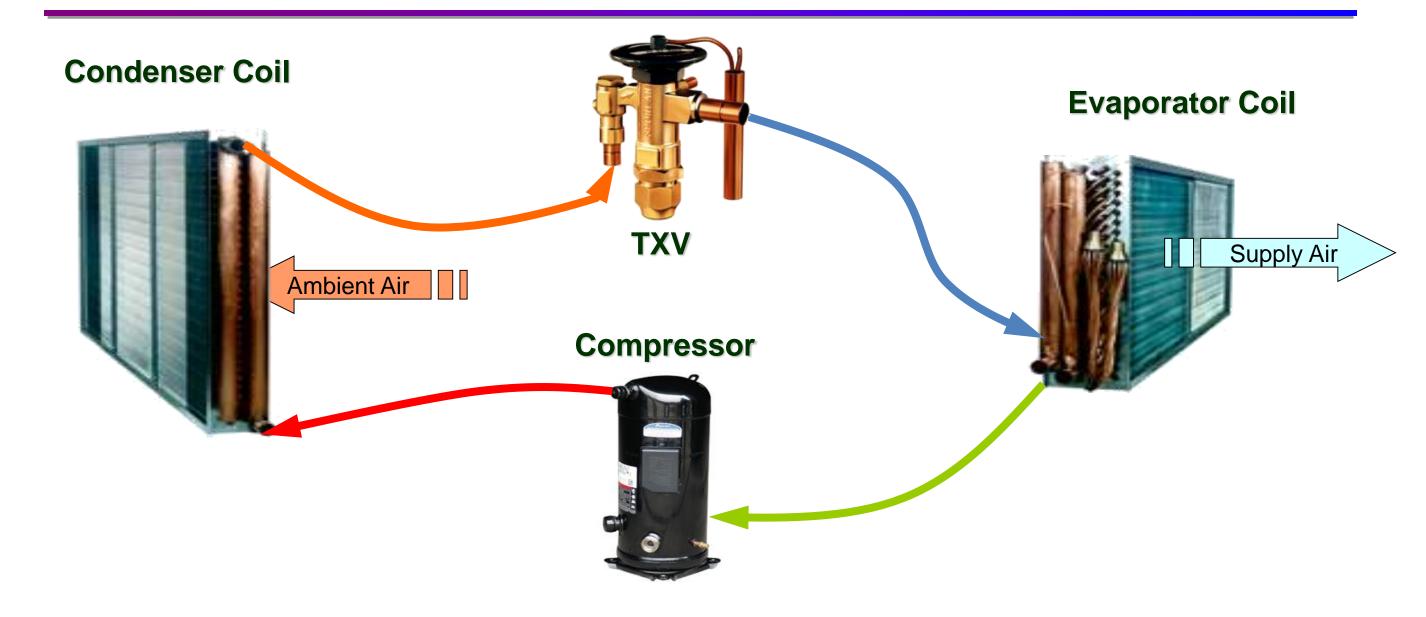
2. Dehumidification needs are increasing

- Reduced sensible loads means lower SHR
 - Lower lighting Watts / sqft
 - Higher insulation R-values
 - Heat reflective / low-e glass
 - often addressed with energy intensive reheat
- > Part-load requirements of IAQ Standard 62.1-2013 s.5.9 (< 65%rh)

often "solved" with energy intensive reheat



Basic DX Cycle



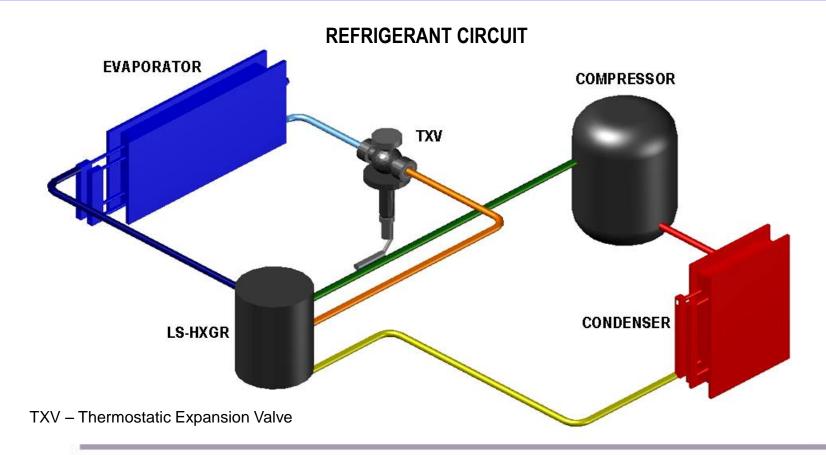
Modified DX Cycle

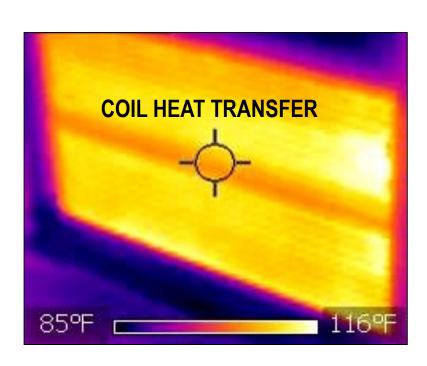


Revises the traditional refrigeration cycle at a fundamental level.

Improvement of evaporator refrigerant / two-phase heat transfer. Increased suction density improves compressor volumetric efficiency.

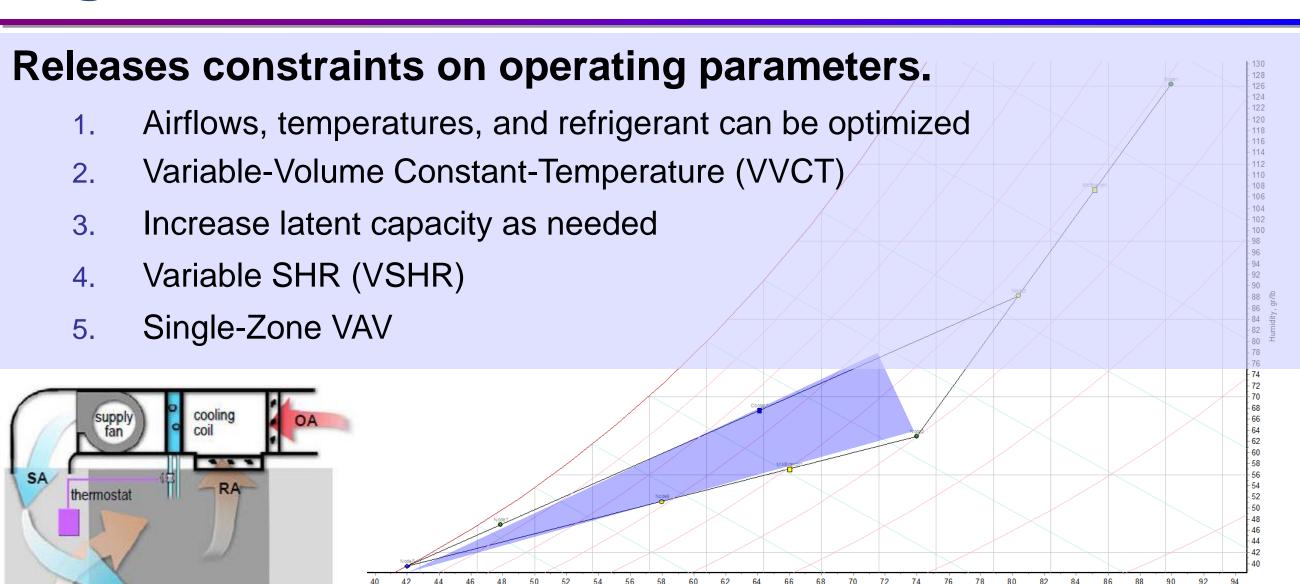
Variable sensible heat ratio optimizes airside performance.





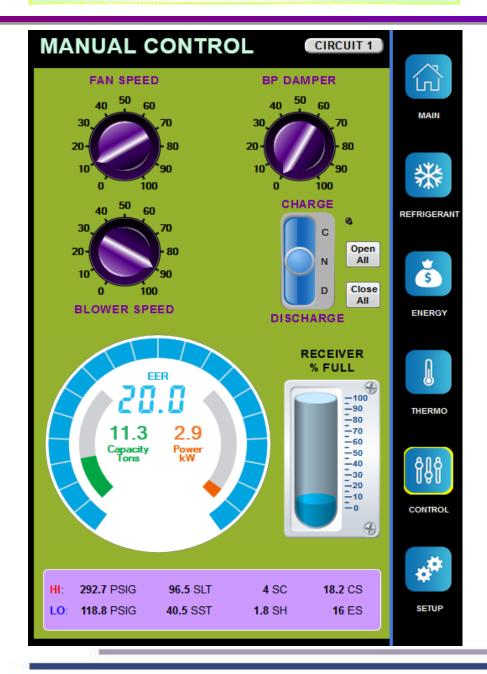
Significance of DX Modification

space



EER Optimizer

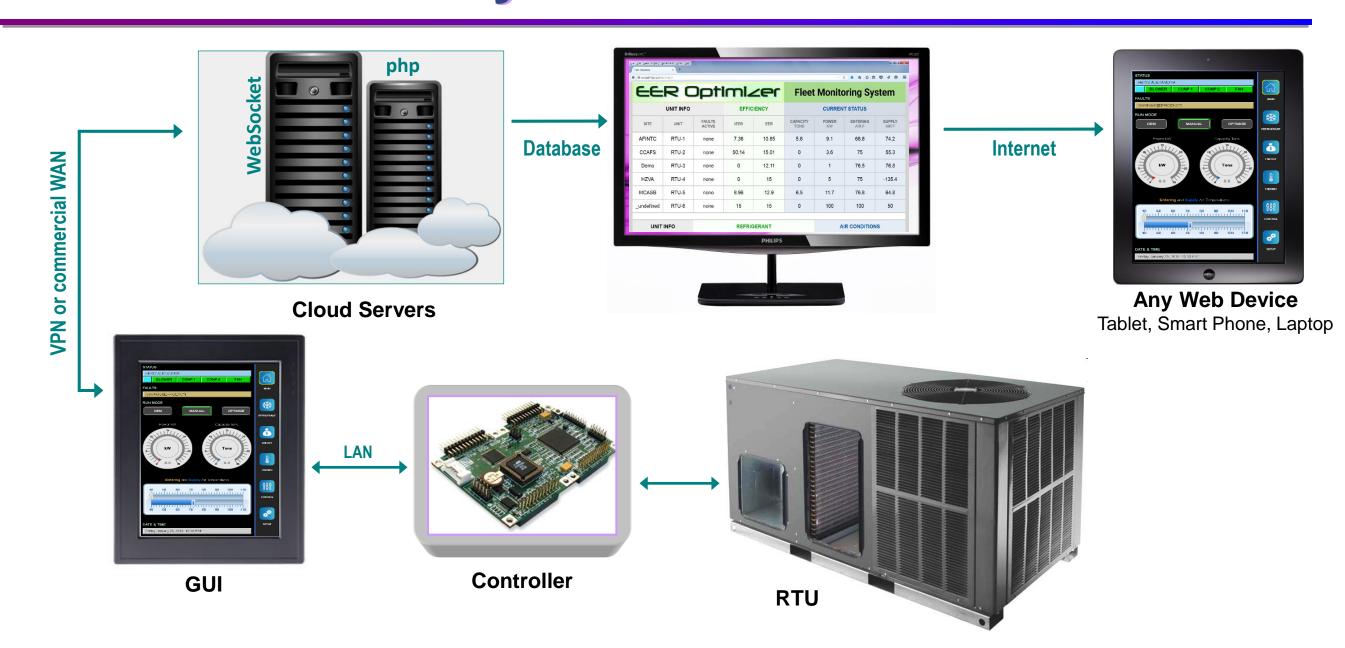
http://www.EERoptimizer.com/



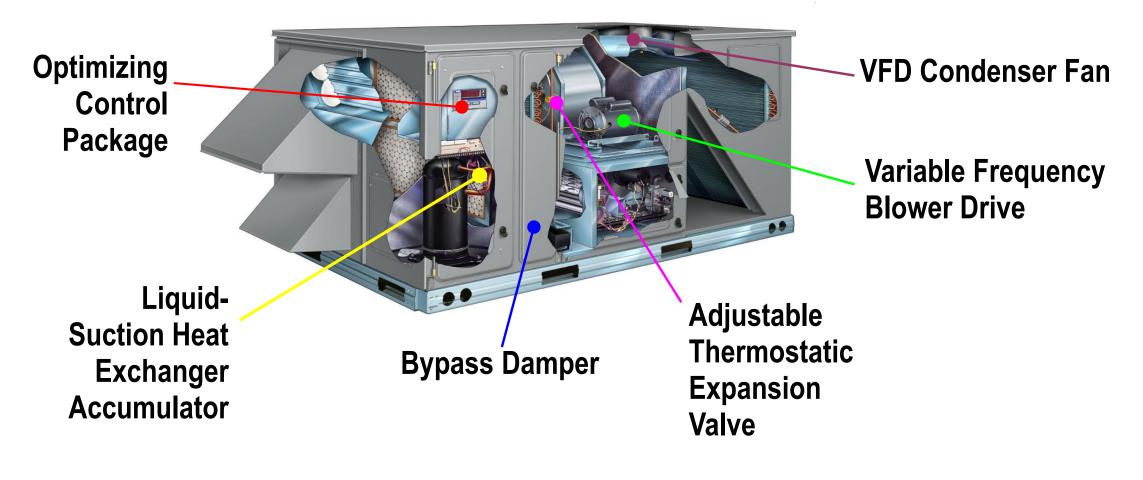
- Controls all operating parameters
- Target is maximum EER while precisely meeting sensible and latent loads
- Continuous performance tuning
 - Supply Blower Speed
 - Condenser Fan Speed
 - Refrigerant Charge
 - Supply Air Temperature
 - Coil Temperature
 - Economizer Damper
- Continuous web reporting
 - EER, IEER, Tons Capacity
 - Faults, such as low refrigerant or fouled coil
 - Diagnostics detects issues before problematic

Data Connectivity





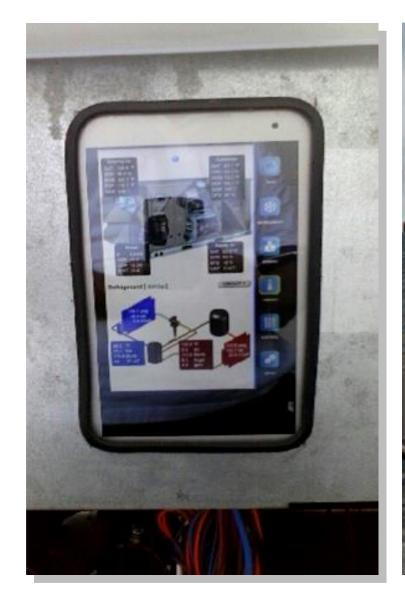
Supercharged Package Unit







Installation

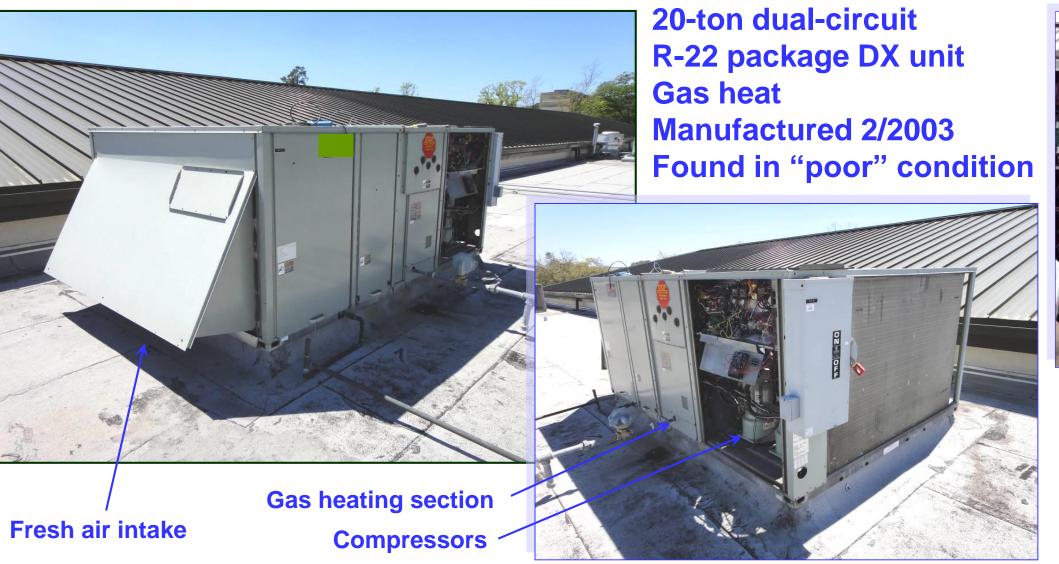






Site 1: Retail Store
Beaufort, South Carolina
2627 cooling degree-days
Climate zone 3A Warm-Humid







Site 2: Classroom Building Mojave, California Elevation 2500 feet

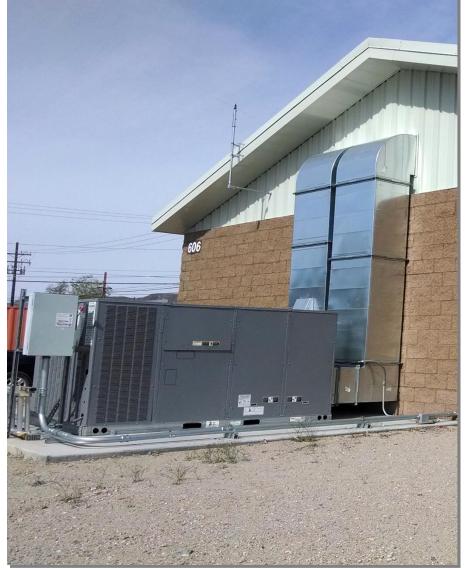
Climate zone 3B Hot-Dry 3225 cooling degree-days 2597 heating degree-days







12½-ton dual-circuit R410a package DX unit Heat Pump Installed 2010





Site 3: Electronics Development Laboratory Cape Canaveral, Florida

3633 cooling degree-days, Climate zone 2A Hot-Humid



8½-ton dual-circuit R410a package DX unit 9 kW-heat Installed 1/2012







Performance Analysis

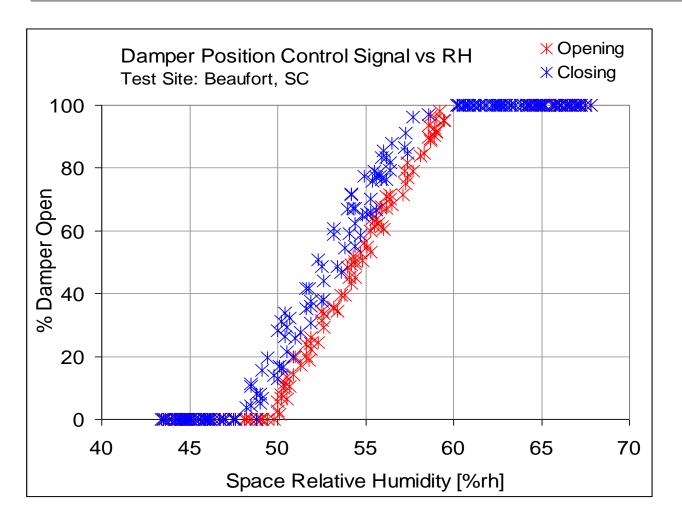
Performance measurement data available live via web links http://www.tinyurl.com/CCAFS-EDL

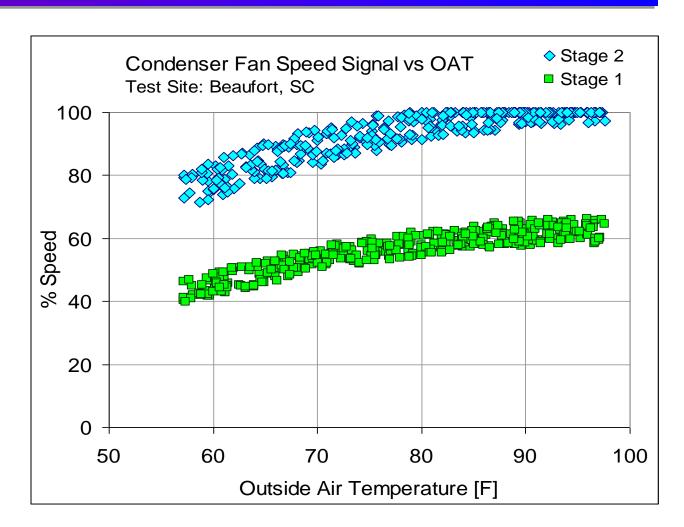


45 Sensors on each RTU

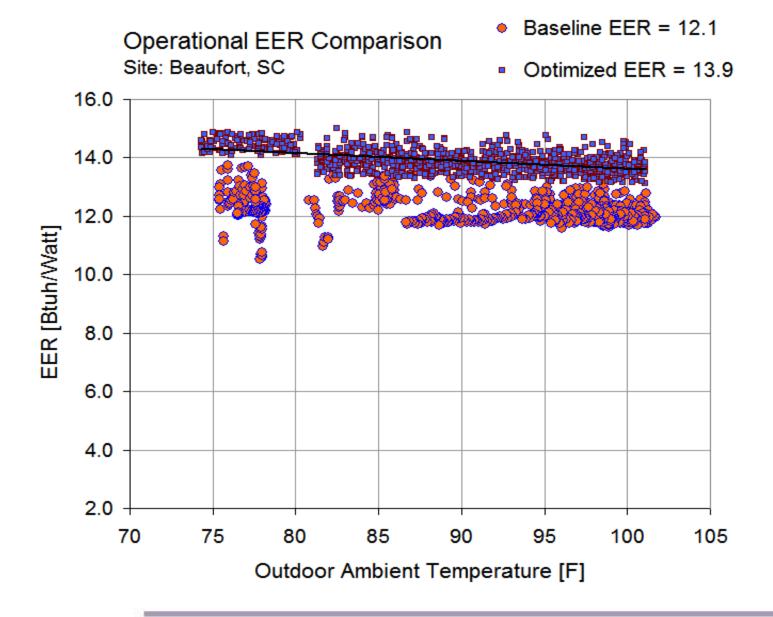
- Compressor Amps (2)
- Fan and Blower Power
- Total Unit Power
- Refrigerant Pressures (4)
- Refrigerant Temperatures (12)
- Refrigerant Flows (2)
- Air Temperatures

 at thermostat, return, outdoor, coil, entering
 & leaving coil, unit discharge
- Air Humidity
 coil entering, unit discharge, at thermostat,
 outdoor
- Space and Ambient CO₂ level
- Control point status





The optimizing controller tuned the RTU's operation according to varying conditions as expected. Shown is control of damper position and condenser fan speed with change in humidity and temperature.

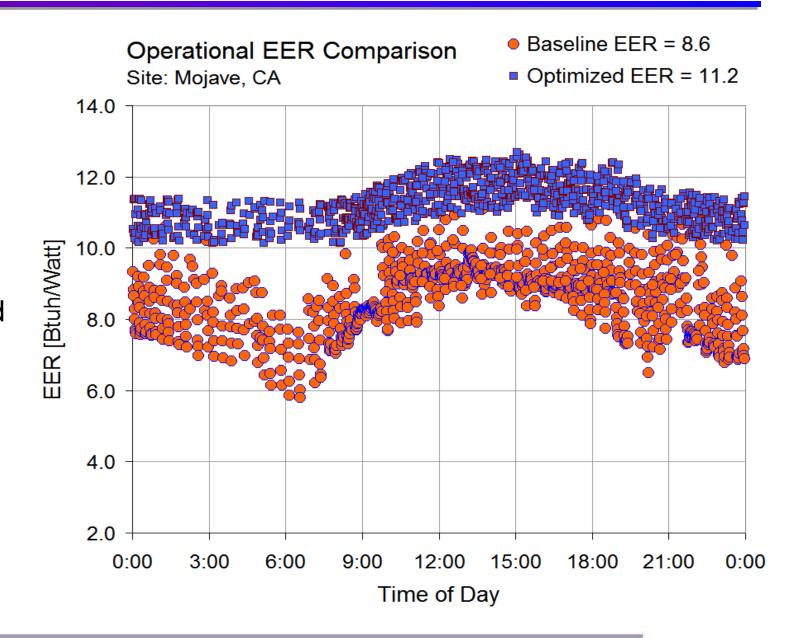


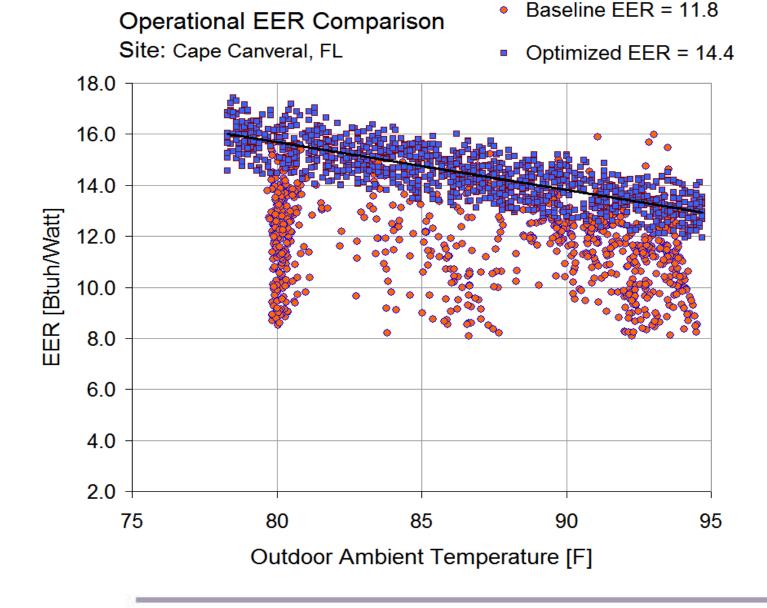
Beaufort SC Field Test Preliminary Result

- 15% IEER increase from 12.4 to 14.3
- 15% operational EER increase
- Elimination of startup efficiency loss
- Reduced compressor cycling
- 27% less energy kWh/CDD consumed

Mojave CA Field Test Preliminary Result

- 37% IEER increase from 7.8 to 10.6
- 31% operational EER increase
- Elimination of startup efficiency loss
- Reduced variation with temperature
- 40% less energy kWh/CDD consumed





Cape Canaveral FL Preliminary Result

- 22% IEER increase from 13.4 to 16.4
- 23% operational EER increase
- Elimination of startup efficiency loss
- Reduced compressor cycling
- 37% less energy kWh/CDD consumed
 - Not counting reheat energy savings
- Space humidity between 45 ~ 50%rh

Results Summary

Data Summary	Operational IEER		Average EER		Efficiency Gain	
Site	Baseline	Optimized	Baseline	Optimized	IEER	EER
Beaufort, SC	12.4	14.3	12.1	13.9	15%	15%
Mojave, CA	7.8	10.6	8.6	11.3	37%	31%
Cape Canaveral, FL	13.4	16.4	11.8	14.5	22%	22%



Beaufort, SC



Mojave, CA



Cape Canaveral, FL

Conclusion

- Web connection from anywhere
- Fault detection & diagnostics
- 15 to 37% operational IEER increase
- Elimination of startup efficiency losses
- 27% to 40% less energy kWh/CDD
- Improved dehumidification
- Cooler compressor operation
- Reduced compressor cycling

