

Advanced Lean-Burn DI Spark Ignition Fuels Research

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Sandia National Laboratories
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Project ID: FT006

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Overview

Timeline

- Project provides science to support industry to develop advanced lean/dilute-burn SI engines for non-petroleum fuels.
- Project directions and continuation are reviewed annually.

Barriers

- Inadequate data and predictive tools for fuel property effects on combustion and engine efficiency optimization.
- Evaluate new fuels and fuel blends for efficiency, emissions, and operating stability with advanced SI combustion.
 1. Lean, unthrottled DISI with spray-guided combustion.
 2. Dilute and mostly premixed charge with advanced ignition.

Budget

- Project funded by DOE/VT via Kevin Stork.
- FY12 - \$750 K
- FY13 - \$700 K

Partners / Collaborators

- PI: Sandia (M. Sjöberg)
- 15 Industry partners in the Advanced Engine Combustion MOU.
- General Motors - Hardware.
- D.L. Reuss (formerly at GM).
- W. Zeng (post-doc, Ph.D. on spray diag.)
- Sandia Spray Combustion (Pickett).
- LLNL (Pitz *et al.*) – Mechanisms and Flame-Speed Calculations.
- USC-LA (Egolfopoulos *et al.*) - Flame Measurements.
- USC-LA (Gundersen *et al.*) – Corona Ignition.



Objectives - Relevance

Project goals are to provide the science-base needed for:

- Determining fuel characteristics that enable current and emerging advanced combustion engines that are as efficient as possible.

DISI with spray-guided stratified charge combustion system

- Has demonstrated strong potential for throttle-less operation for high efficiency.
- Overall lean operation prevents easy aftertreatment reduction of exhaust NO_x .
- High-EGR operation can reduce NO_x formation, but can also lead to partial burns.
- Stratified charge can easily cause soot formation.
- Hence, mastering NO_x / Soot / Combustion Stability trade-off is key to success.
- These processes are strongly affected by fuel properties (*e.g.* ethanol content).

- Develop a broad understanding of spray-guided SI combustion (*i.e.* conceptual model, including fuel effects).
 - For highest efficiency, cyclic variability needs to be minimized.
 - Help develop engineering tools that go beyond ensemble-averaged combustion, and incorporate cyclic variability.
- Current focus is on E85 and gasoline, and blends thereof.
 - Latest E85 specifications allow 51-85% ethanol by volume.
 - Flex-fuel vehicles need to function with 0 – 85% ethanol in the fuel tank.

Approach

- Combine metal- and optical-engine experiments and modeling to develop a broad understanding of the impact of fuel properties on DISI combustion processes.
- First, conduct performance testing with all-metal engine over wide ranges of conditions to identify critical combinations of operating conditions and fuels.
 - Speed, load, intake pressure, EGR, and stratification level.
- Second, apply a combination of optical and conventional diagnostics to develop the understanding needed to mitigate barriers.
 - Include full spectrum of phenomena; from intake/compression flows, fuel injection, fuel-air mixing, spark development and ignition, to flame spread and burn-out.

Supporting modeling and experiments:

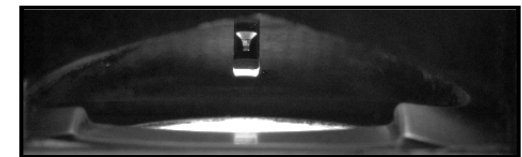
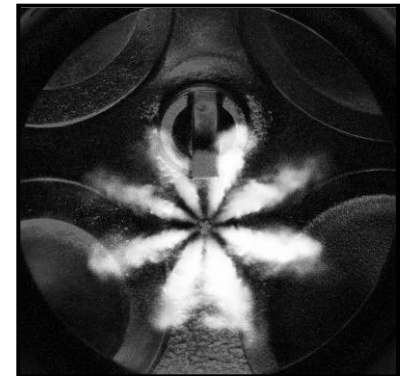
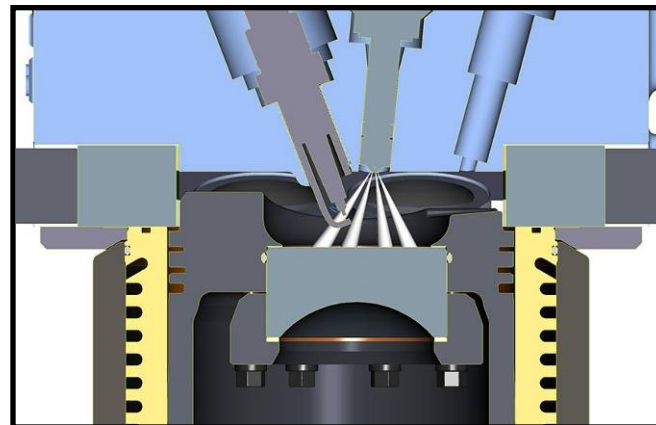
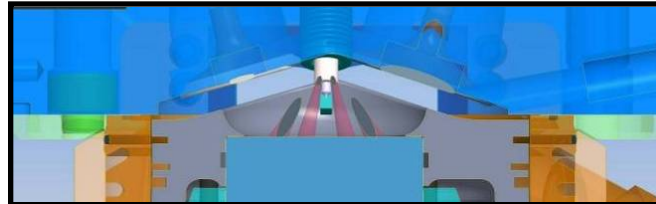
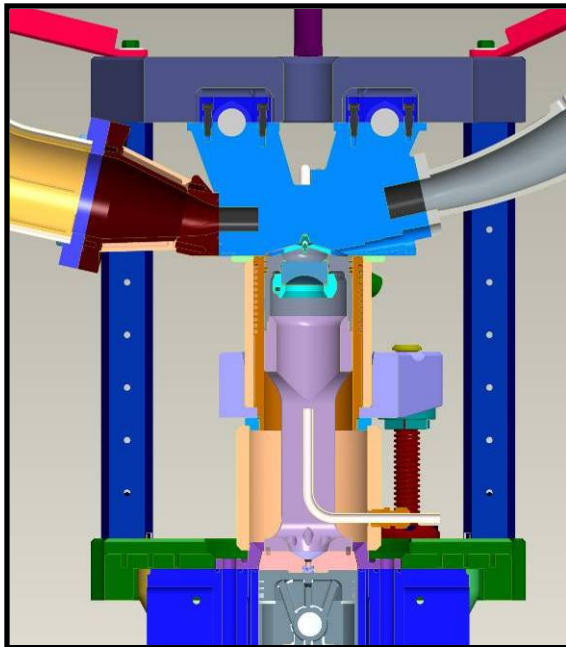
- Conduct chemical-kinetics modeling of flame-speed and extinction for detailed knowledge of governing fundamentals.
 - Collaborate on validation experiments and mechanism development.
 - CFD modeling of spray penetration and mixing.
- Addresses barriers to high efficiency, robustness, and low emissions by increasing scientific knowledge base and enhancing the development of predictive tools.

Approach / Research Engine

Two configurations of drop-down single-cylinder engine.

Bore = 86.0 mm, Stroke = 95.1 mm, 0.55 liter swept volume.

- All-metal: Metal-ring pack and air/oil-jet cooling of piston.
- Optical: Pent-roof window, piston-bowl window, and 45° Bowditch mirror.
- Identical geometry for both configurations, so minimal discrepancy between performance testing and optical tests.
- 8-hole injector with 60° included angle \Rightarrow 22° between each pair of spray center lines.
Spark gap is in between two sprays.





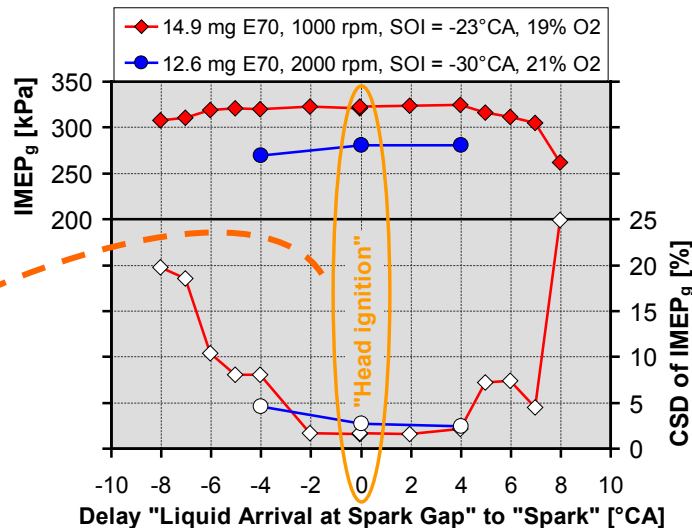
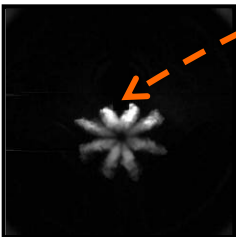
Technical Accomplishments

- ➔ ● Examined E85 operation with near-TDC fuel injection for ultra-low NO and soot.
 - ➔ — Spectroscopic characterization of the various stages of ignition and combustion.
 - Effects of intake O₂ on exhaust soot emission.
 - Spark-plasma stretch analysis and dual-camera high-speed combustion luminosity imaging for understanding partial burn cycles.
- ➔ ● Performed PIV measurements of in-cylinder flows during compression, fuel injection and combustion.
- ➔ ● Compared NO formation for E85 and gasoline.
 - ➔ — PIV measurements to understand mixing rates of hot combustion gases.
- ➔ ● Investigated effects of air flow (rpm & swirl) on well-mixed & stratified E70 oper.
 - Determined how the combustion rate scales with engine speed, and the effects of cyclic flow variability.
- Initial examination of effects of fuel blend (E0 to E100) on stratified operation.
 - ➔ — Spark-timing requirement for stable ignition and low soot emissions.
 - Soot and NO exhaust emissions across load ranges for operation with "head ignition".
- Examined the potential of PLIF imaging of E85 using intensified high-speed camera.
- Set up and validated FORTÉ CFD-code to study fuel-jet penetration and mixing.
- For well-mixed operation, initialized study of fuel effects on endgas autoignition (knock).



Parameter Space

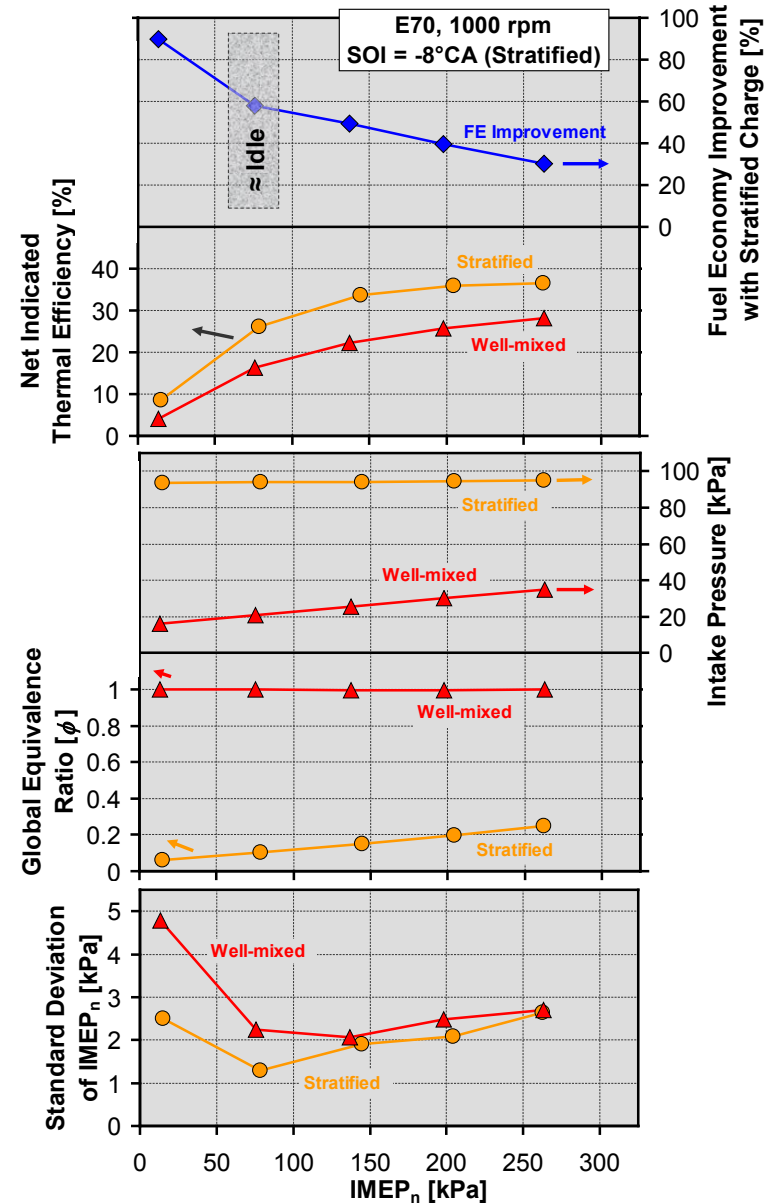
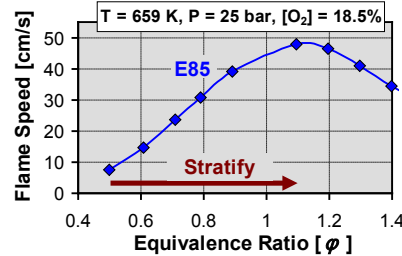
- The parameter space is huge.
- Grouped as hardware, static parameters & operating variables.
- Stratified operation for E70 and E85 often used spark timing (ST) for “head ignition”.
- Stable combustion with good CA50 control.
- Head ignition can easily lead to unacceptable soot for gasoline.
 - Later spark is then needed (*i.e.* tail ignition).



Parameter	This Presentation
CR	12
Piston Bowl	Ø 46 mm
Valve Timings	For Minimal Residual Level
Injector & Spray Targeting	Bosch 8 x 60° Straddling Spark
Swirl Index	2.7
Tumble Index	0.62
Injection Pressure	170 bar
# of Injections	Single
Spark Energy	106 mJ
T _{coolant}	60°C
T _{in}	26-28°C
P _{exhaust}	100 kPa
Fuel Type	Gasoline (E0) – E100
Engine Speed	1000 - 2000 rpm
Intake Pressure	18 - 105 kPa
IMEP _n	20 - 637 kPa
Start of Injection	-310 to -6°CA
Spark Timing	-36 to -5°CA
EGR / [O ₂] _{in}	21 – 14.5% O ₂

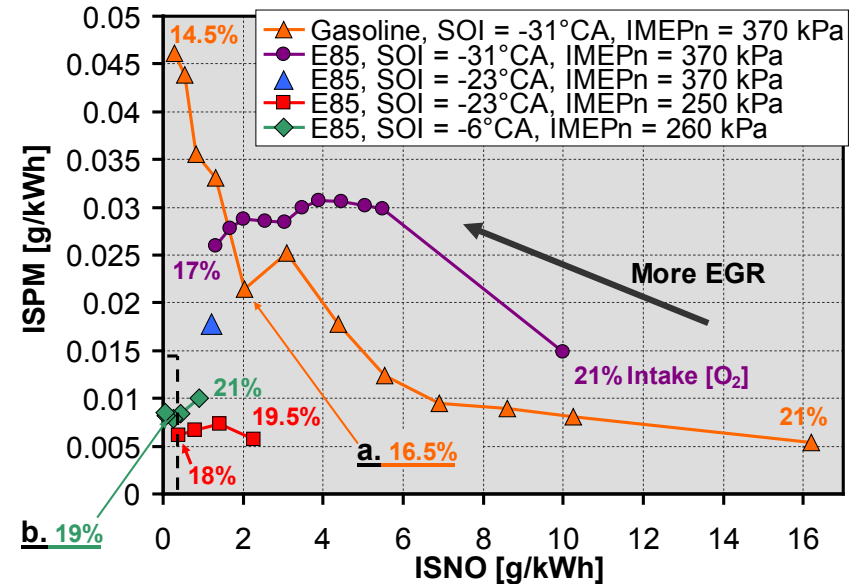
Fuel Economy Potential with Stratified Comb.

- Well-mixed lean mixtures burn too slowly for stable SI operation.
- Overcome this with fuel stratification to raise local ϕ .
- Allows lean and throttle-less engine operation.
 - High γ , and no pumping losses.
 - \Rightarrow High efficiency.
- Example for E70 fuel.
- Strongest gain of fuel economy for low loads.
 - 30% FE gain at 1/4 load to 60% near idle.
- Overall lean operation prevents easy exhaust aftertreatment of NO_x .
- This example used “head ignition” of fuel jets.
- Head ignition allowed very small fuel injections to be combusted stably.



Previous Results - Reaching Inside NO/PM Box

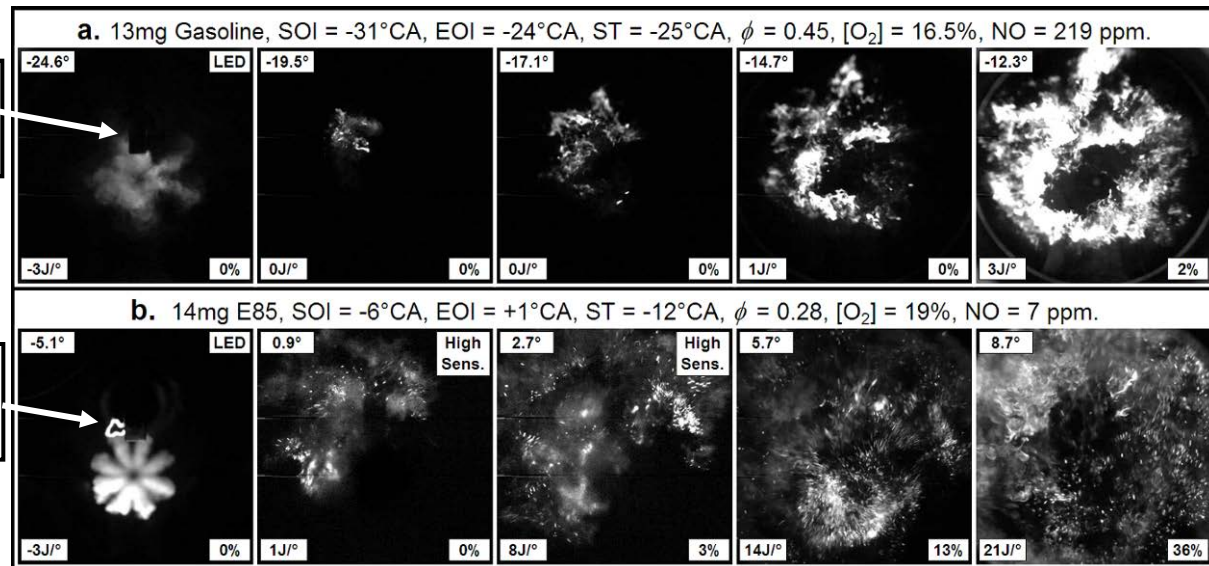
- With E85, can reach inside the US2010 NO/PM box, using near-TDC injection.
- E85 responds favorably to SOI retard.
 - Lower peak temperatures, and less residence time, \downarrow NO formation.
- Oxygenated fuel, and strong vaporization cooling of ethanol.
 - Suppresses soot formation.



- Less flame-like combustion for E85 warrants further investigation.
- Use spectrograph.
- Nature of early faint flames?
- Presence of soot?

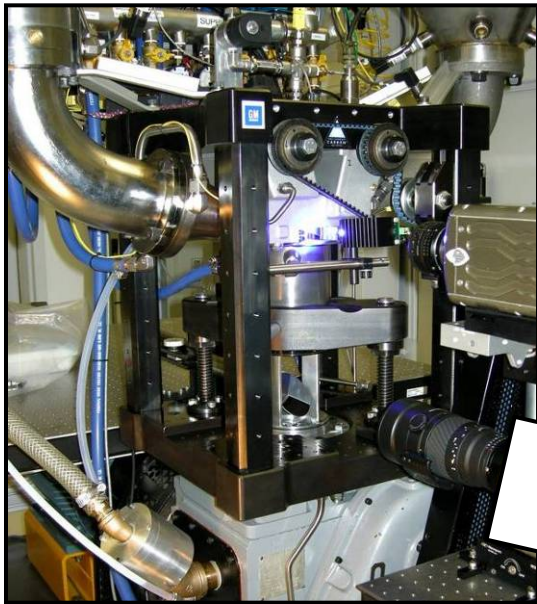
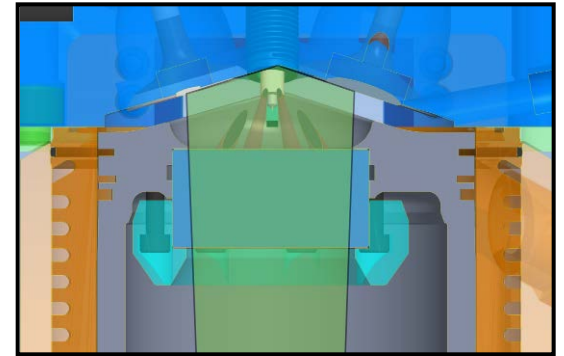
Gasoline Tail Ignition

E85 Head Ignition

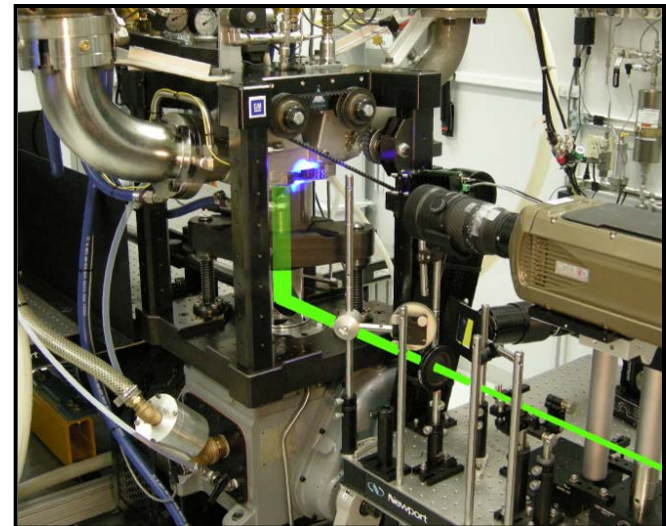
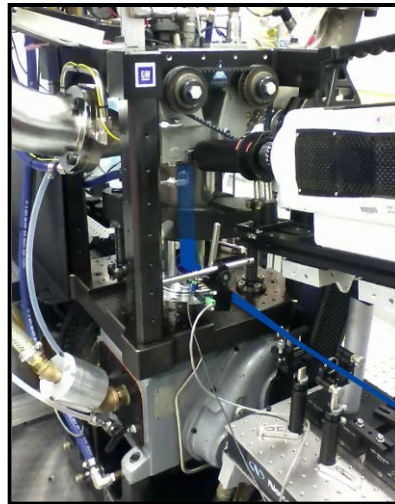


Optical Diagnostics Setups

- PLIF high-speed 355 nm laser – Quantronix HP-UV. Intensified Phantom v311.
- In-house developed pulsed high-intensity LED for Mie-scattering.
- PIV high-speed 532 nm laser - Quantronix Dual Hawk.
 - Vertical laser sheet near spark-plug gap.
- Mie & natural luminosity imaging via Bowditch mirror.
 - Notch filters to reject 532 nm laser light.
- Dual-camera setup or Spectrograph.

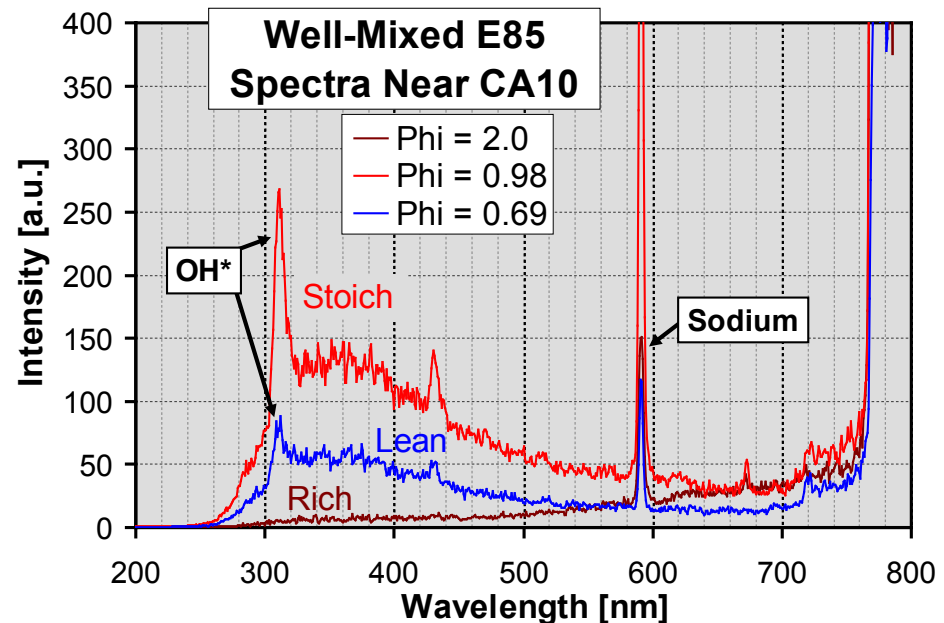
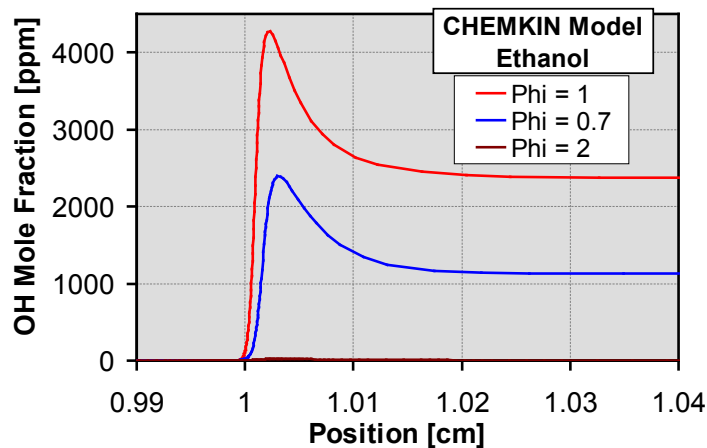
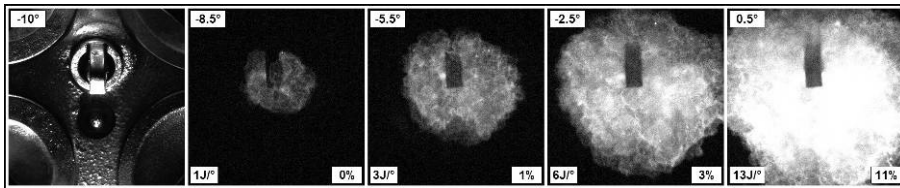


v7.10 or
Spectrograph



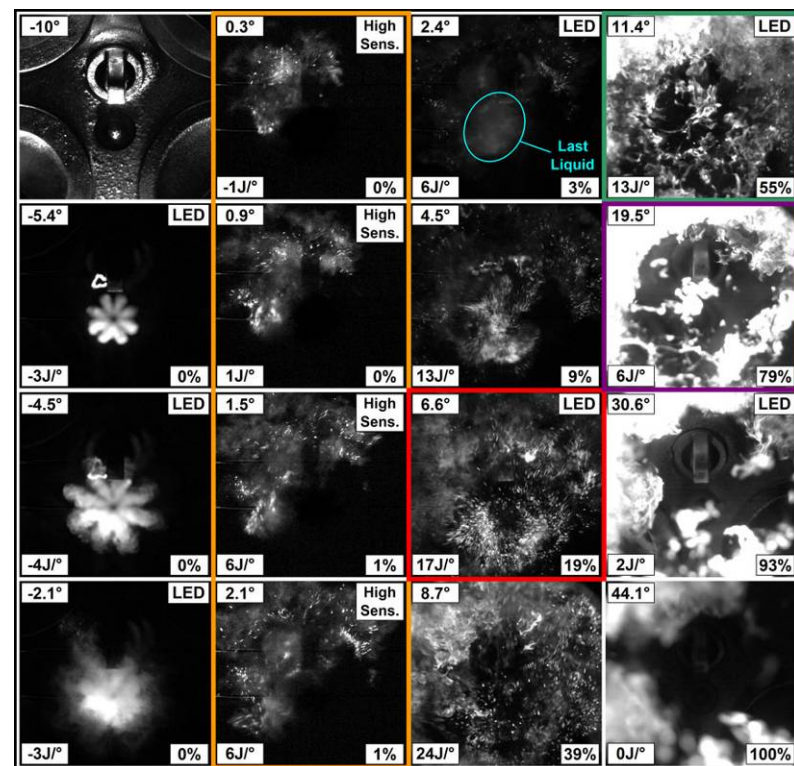
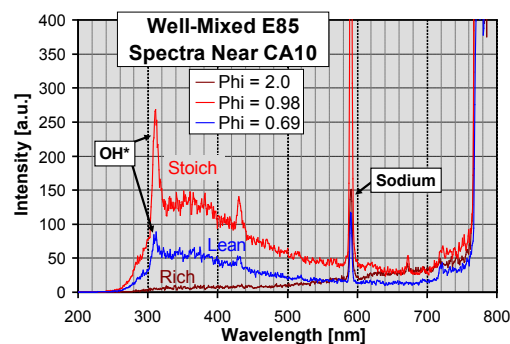
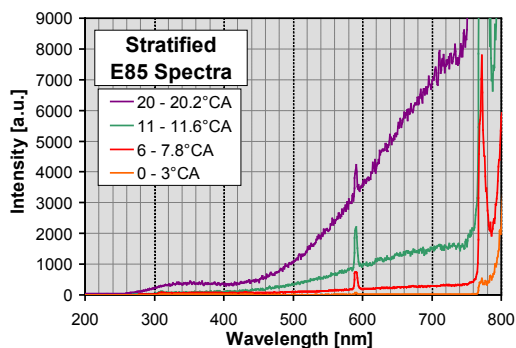
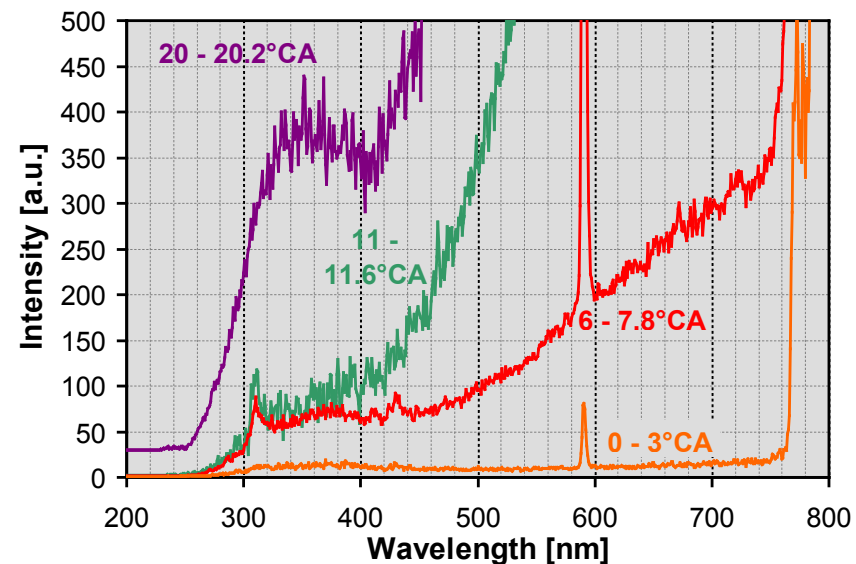
Well-mixed Spectral Response

- Spectrograph had coarse grating with 122 lines/mm.
 - Low resolution, but useful for obtaining an overview of the light characteristics.
- Emissions lines near 590 nm indicate high sodium content in fuel.
- Stoichiometric and lean operation show emissions peak near 308 nm.
 - Indicative of high levels of excited OH^* .
- Spectra are consistent with CHEMKIN flame-modeling results.
- Rich combustion has weak luminosity and no peak near 308 nm.



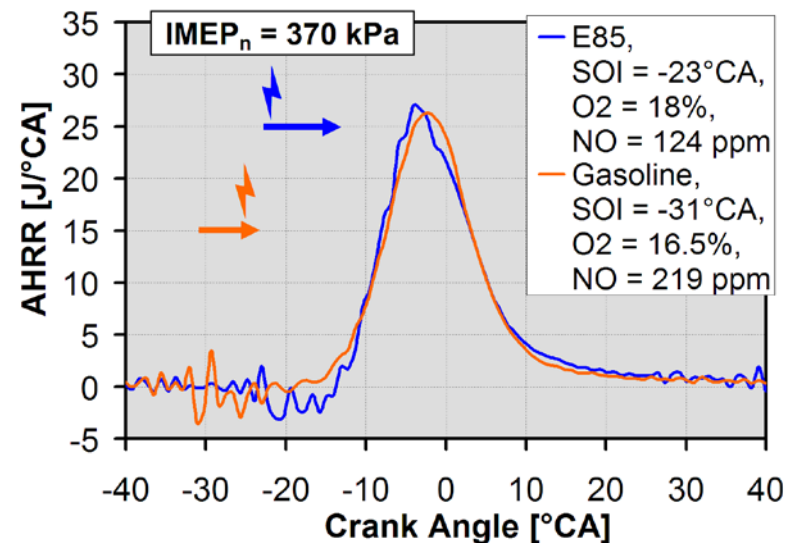
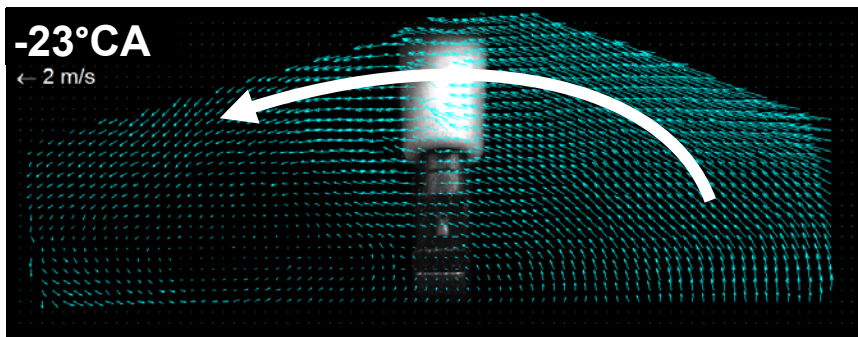
Stratified Spectra

- E85. SOI = -6°CA . Spark = -12°CA .
- Early luminosity is weak, and shows no peak around 308 nm.
 - Indicative of exclusively rich combustion.
- Hypothesis: Early flame is strained along fuel jets. Avoids extinction by existing in ϕ - regions with highest robustness.
- From 6° to 11°CA , distinct peak near 308 nm indicates stoichiometric and lean combustion.
- Late luminosity is dominated by black-body radiation, indicative of soot.



NO Emissions for Gasoline and E85

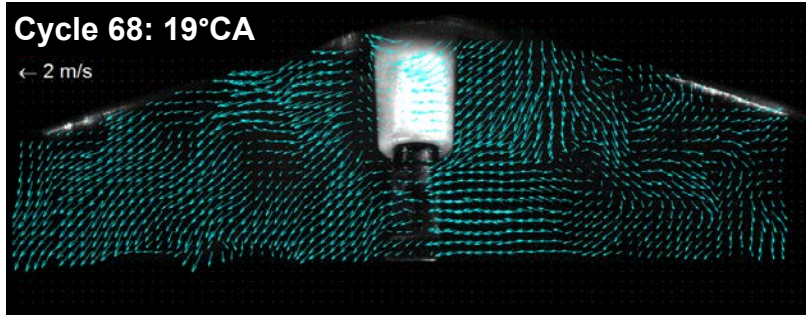
- Gasoline with SOI = -31°CA , and E85 with SOI = -23°CA have very similar AHRRs.
- Yet, NO emissions are 77% higher for gasoline (**219 vs. 124 ppm**). Why?
 - A. Intake $[\text{O}_2]$ is 1.5% lower for gasoline, so goes wrong way.
 - B. Spray model shows 60K more vaporization cooling for ethanol (at $\phi = 0.8$).
- With these factors, detailed gasoline/E85 surrogate mechanism by Dr. Marco Mehl at LLNL predicts **26K higher flame temperature** at $\phi = 0.8$ for E85.
- Hence, **other factors** must come into play as well to **limit NO formation**.
 - C. EOI to CA50 delay is 23°CA for gasoline but only 12°CA for E85. (Tail vs. Head Ignition).
 - D. E85 has 52% more fuel injected because of its lower heating value.
- **C & D** implications on in-cylinder mixing rates?
- Perform PIV measurements with and w/o fuel injection.
- Average non-DI PIV shows development of tumble flow in bowl.



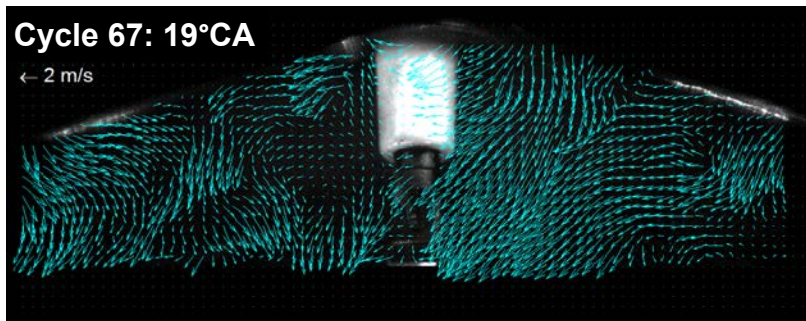
Mixing Rates Vs. NO Emissions

- PIV shows that in-cylinder turbulent kinetic energy is higher during burn-out for E85.
 - Lower heating value of E85 \Rightarrow 52% more fuel injected \Rightarrow More fuel-jet momentum.
 - More closely-coupled injection and combustion.

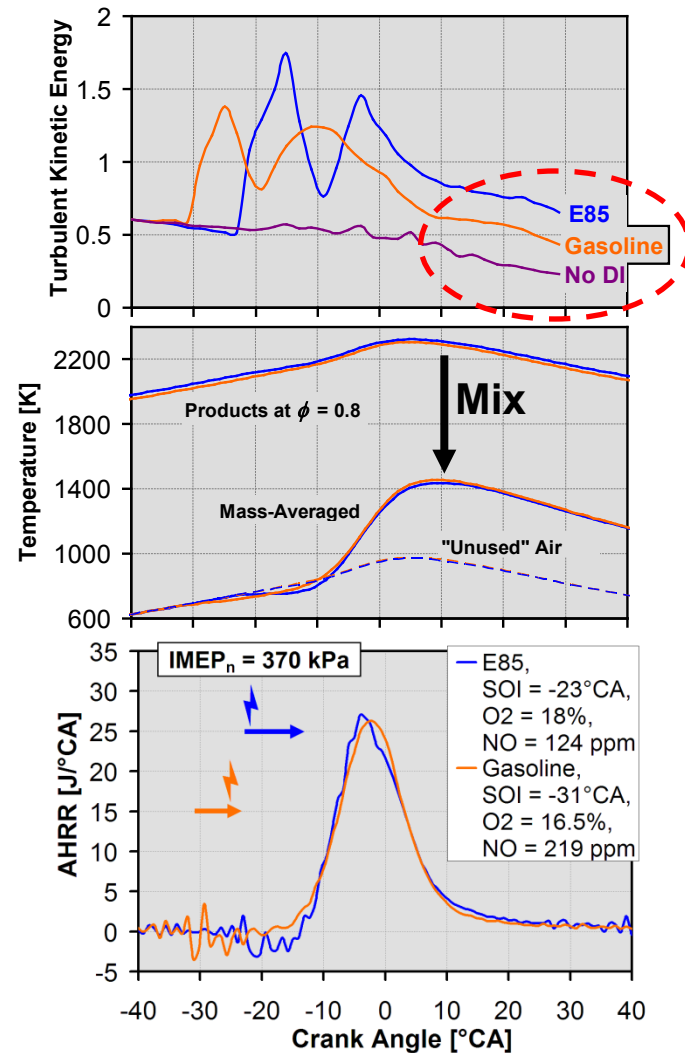
Gasoline



E85



- Global $\phi = 0.43-0.45$, so **more rapid mixing** of hot combustion products with cooler unused air has potential to stop thermal NO production.
- Consistent with E85's observed lower NO emissions.



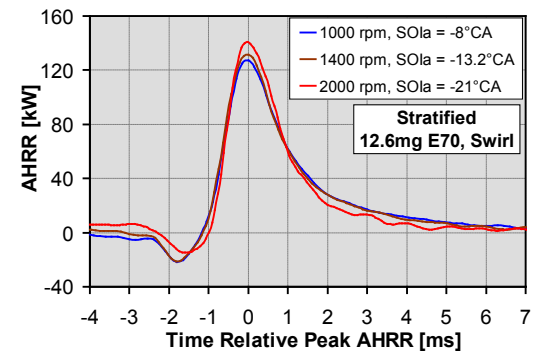
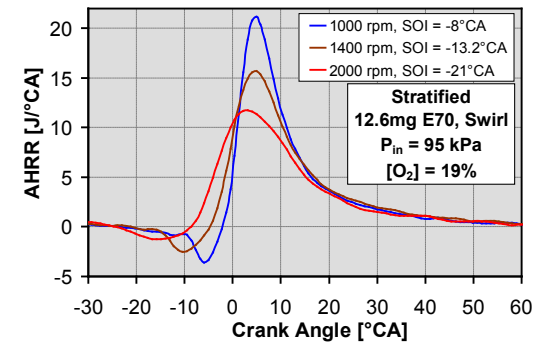
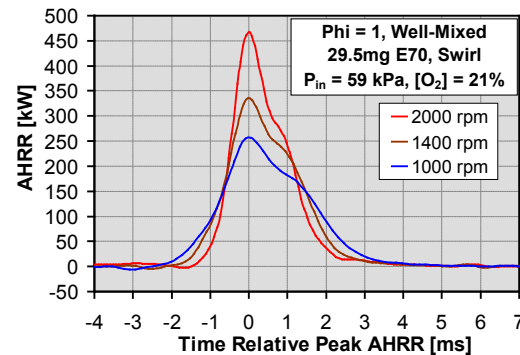
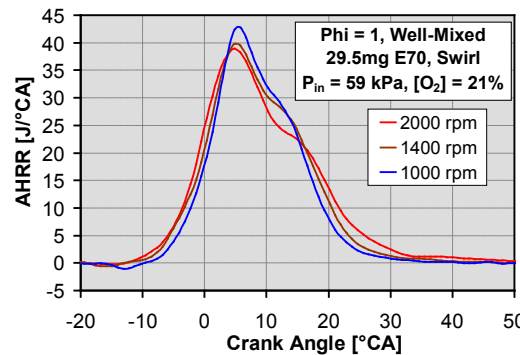
Role of In-Cylinder Flow Field

What is the role of the in-cylinder flow field for stratified-charge combustion?

— The flow generated by the intake and compression strokes.

- Change the flow by **changing engine speed**.
- Observe AHRR changes. Well-mixed (WM) and stratified combustion.
- Well-mixed AHRR constant in $J/^{\circ}CA$, stratified AHRR spreads out.
- WM-comb. speeds up in kW/ms. Combustion rate scales with turbulence level.

- Stratified combustion rate constant in kW/ms.
 - Combustion rate governed by fuel/air mixing.
 - On average, this mixing is dominated by fuel-jet penetration.
-
- This is for E70 “head-ignition”.
 - “Tail ignition” more controlled by flame propagation?

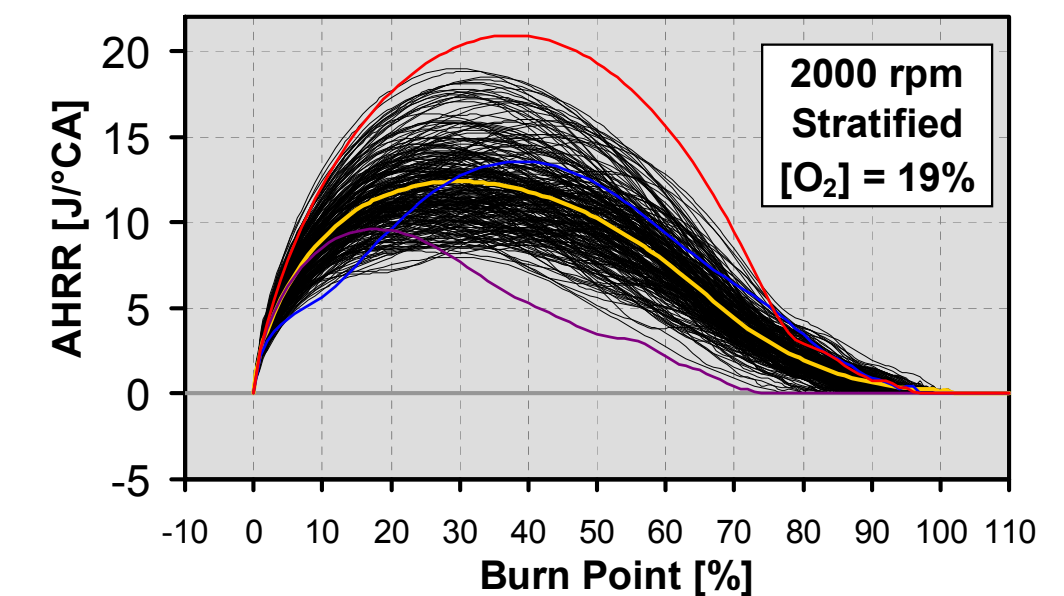
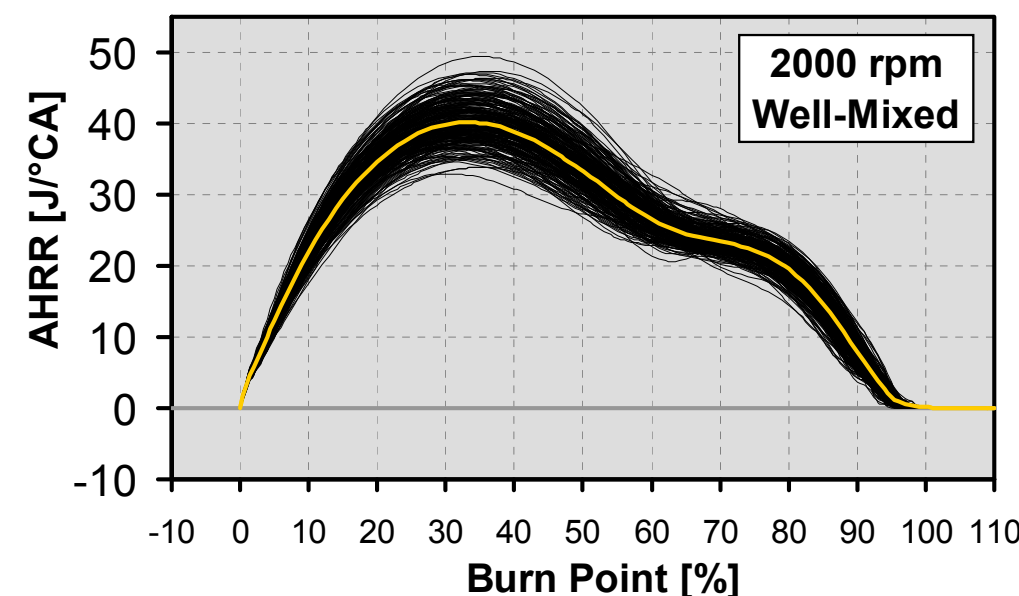
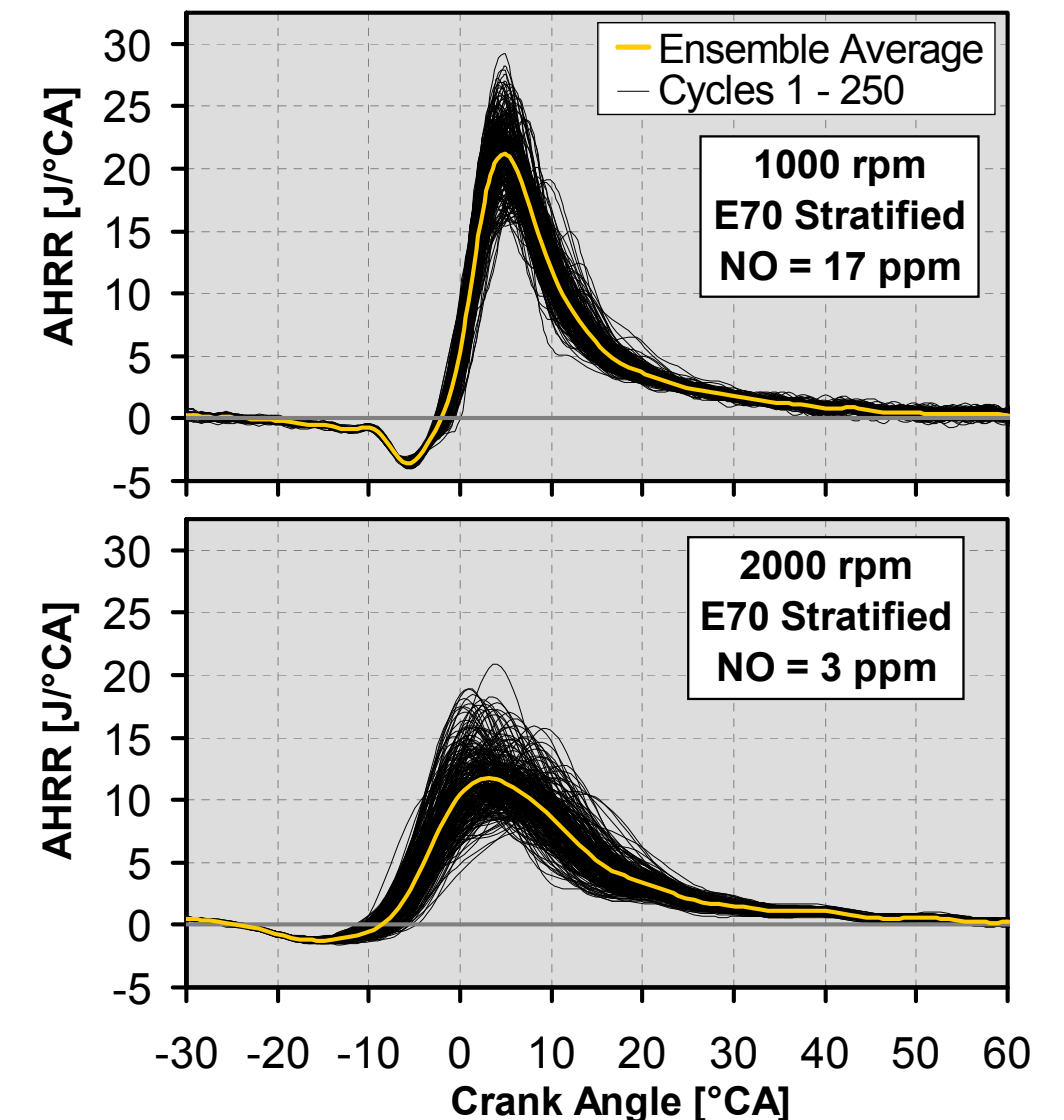
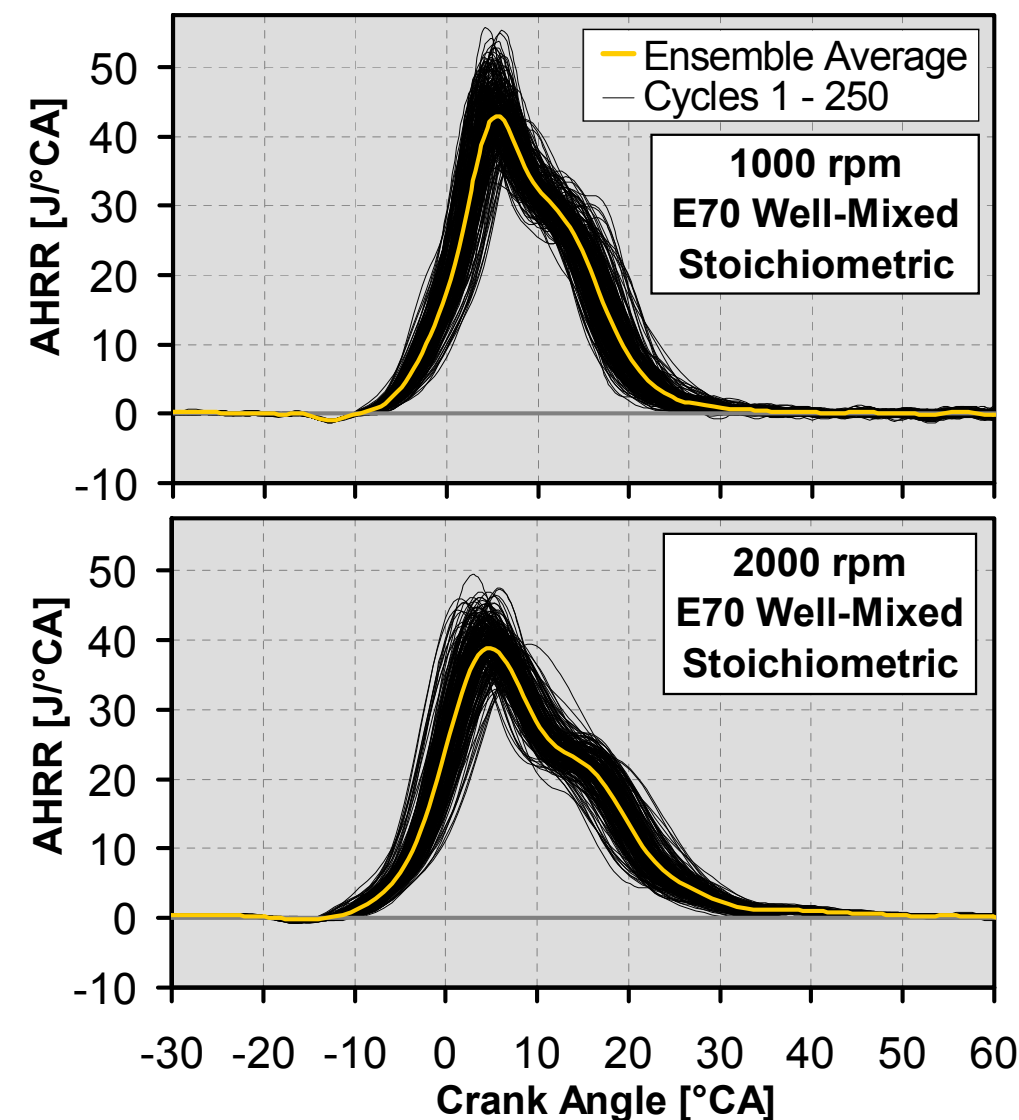


In-Cylinder Flow Field vs. Cyclic Variability

- Well-mixed operation: Relative cyclic variability does not change.
- Stratified combustion: rpm \uparrow , in-cylinder flow field becomes sufficiently strong relative to the fuel jets \Rightarrow increased variability of combustion.

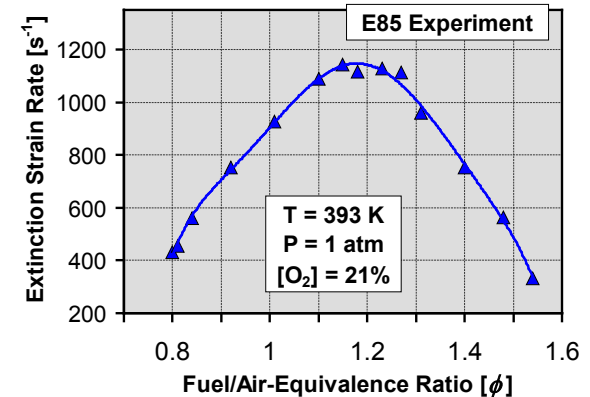
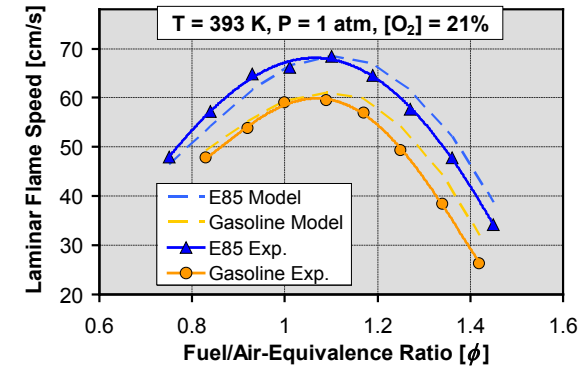
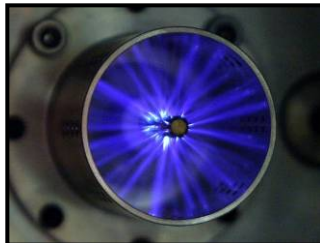
- CA50 variations make interpretation more difficult.
- Replot AHRR against % burn.
- WM shape is very repeatable.
- Stratified show large variability in burn profile.
- Less EGR stabilizes comb., but NO would increase.
- Keep EGR, but avoid slow and incomplete burns.

- Demonstrates need to go beyond averaged results.
- Continue study variability with multiple diagnostics.



Collaborations / Coordination

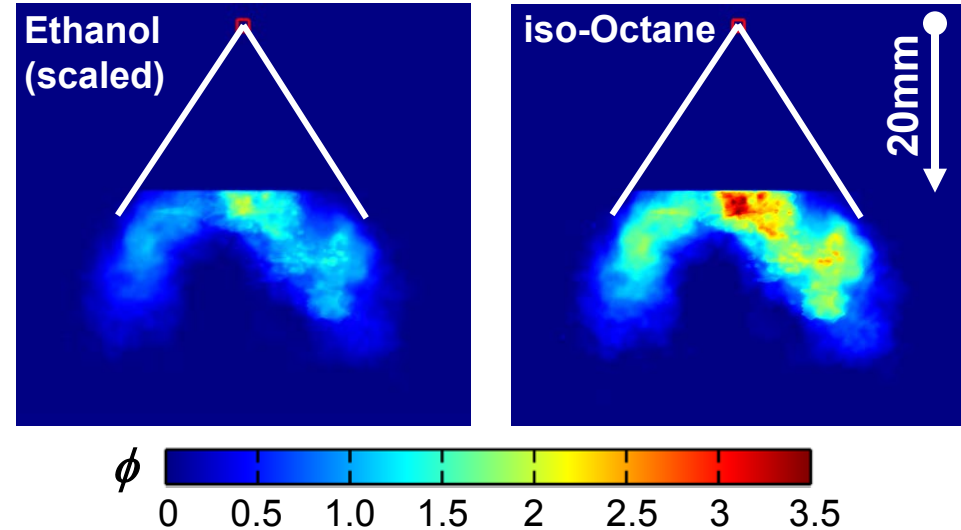
- General Motors.
 - Hardware, discussion partner of results, and for development of diagnostics.
- D.L. Reuss (formerly at GM, now at UM).
 - Development and interpretation of high-speed PIV and PLIF.
- 15 Industry partners in the Advanced Engine Combustion MOU.
 - Biannual meetings with 10 OEMs and 5 energy companies.
- LLNL (W. Pitz and M. Mehl).
 - Prediction of flame robustness for engine-conditions.
 - Development of chemical-kinetics mechanisms for gasoline-ethanol mixtures.
- USC-Los Angeles (Prof. Egolfopoulos) (not VT).
 - Flame speed and extinction measurements for gasoline/ethanol blends, and modeling.
- USC-Los Angeles (Prof. Gundersen) (not VT).
 - Corona Ignition.



Collaborations (2)

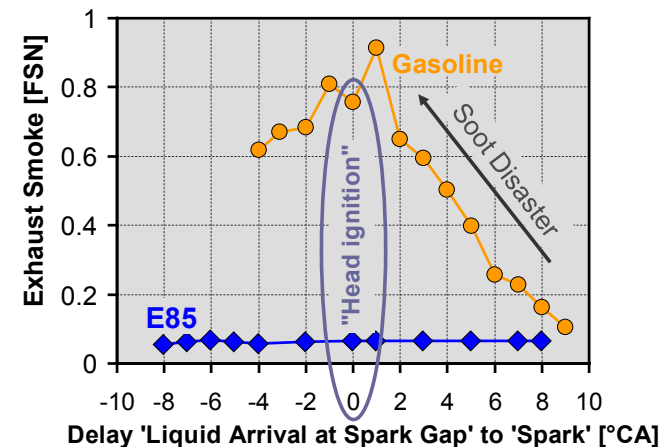
Sandia Spray Combustion (L. Pickett)

- Fuel effects on multi-hole sprays.
- Rayleigh-based measurement of fuel vapor for iso-octane.
 - Schlieren measurements indicate that air entrainment is very similar for ethanol.
- Rescale based on A/F_{st} to estimate differences in internal ϕ .
 - Iso-octane: up to $\phi = 3.5$ at 20mm from injector. — Ethanol: up to $\phi = 2$.



Project Accomplishments Cont.

- “Head Ignition” often provides stable operation with closely coupled injection and combustion.
 - Enables late SOI to drastically lower NO_x emissions.
- Typically, head ignition cannot be used for gasoline.
 - Spark needs to be retarded to allow rich regions to mix out and avoid “soot disaster”.





Future Work FY 2013 – FY 2014

- Continue PIV measurements of in-cylinder flows across speed ranges.
 - Examine relative strength of flow field and fuel jets.
 - Stratified operation with head and tail ignition.
- Study in detail interaction between flow field, spark plasma, and fuel jets.
 - Understand cyclic variability of stratified combustion for low-NO_x operation.
- Continue study effects of fuel blend (E0 to E100) on stratified operation.
 - Ignition stability, soot and NO_x exhaust emissions.
- Examine fundamental effects of charge temperature on stratified low-NO_x / soot operation with E85 and gasoline.
- Continue the development of the fuel-PLIF technique.
 - Apply PLIF to measure ϕ -fields for better understanding of fuel/air-mixing.
- Examine fuel-vaporization effects on thermal efficiency.
 - Boosted operation and high ethanol content.
- Continue using CHEMKIN to investigate flame-extinction fundamentals.
 - Provide better understanding of in-cylinder turbulence on flame quenching.
- Use FORTÉ CFD-code to study fuel effects on fuel-jet vaporization and mixing.
- Start examining the use of advanced ignition for lean/dilute combustion.

- This project is contributing to the science-base for the impact of alternative fuel blends on advanced SI engine combustion.
- Stable stratified operation was demonstrated to loads below idle.
 - Fuel economy improvement of **30% to 60%** relative throttled stoichiometric operation.
- Near-TDC fuel injection of E85 using “head-ignition” of fuel jets can enable very low exhaust NO and soot.
- Spectroscopic measurements indicate that **early E85 flames are exclusively rich**.
 - Consistent with measurements of flame-extinction rates of same E85 fuel.
- With similar heat-release, **NO emissions are much lower for E85** than for gasoline.
- PIV measurements show that E85’s short delay from injection to combustion and more injected fuel together lead to **higher turbulence level during burn-out**.
 - Should contribute to limit thermal NO formation through mixing with cooler unused air.
- **Well-mixed and stratified operation respond very differently to changes of rpm.**
- Well-mixed HRR in kW scales directly with engine speed via increased turbulence.
- On average, stratified HRR in kW remains invariant to increased engine speed.
- Stronger intake and compression flows at higher rpm lead to increased variability of stratified combustion.